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MOTION OF VISCOUS LIQUID IN ROTATING CYLINDER

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MOTION OF VISCOUS LIQUID
IN ROTATING CYLINDER

by
ChiCheng Wu

Dissertation submitted to the Faculty of the Graduate School
of the New Jersey Institute of Technology in partial fulfillment
of the requirements for the degree of
Doctor of Engineering Science
1982

APPROVAL SHEET

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ABSTRACT

A computer model for the withdrawal and buildup of the fluid from the pool bounded on the horizontal, rotating, cylindrical surface has been developed. For an incompressible Newtonian fluid, the model is based upon the modified SOLA-SURF algorithm to solve the finite-difference form of Navier-Stokes equations on primitive variables. A hybrid of centered and partial donor cell differencings is applied to the momentum convective flux. In each time cycle, the cell velocity is adjusted by the pressure corrector and iterates to satisfy the equation of continuity. The free surface position is approximated by a surface height function derived from kinematic equation. The flow field moves under control of constant fluid volume. When the bottom boundary is a free surface, it is necessary to make good initial guess of the velocities for the newly entered cells.

An example computation predicts the onset velocity of rimming flow within 5% of that of an empirical correlation for unsaturated polyester. It indicates the correlation can be applied to the fluid of medium range viscosity. The fluid velocities in the vicinity of rigid boundary are close to rotating velocity. Backward velocities occur near the free surface. The transient flow phenomena is qualitatively in agreement with the experimental observations. The criterion of constant fluid volume is held within 10%. Singularity resulted in the calculation may be an indication that the assumption of no-slip boundary condition at contact line is invalid. The size of time step is found very critical to the stability of computation.

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I. INTRODUCTION

This study was undertaken to develop a computer model to predict the incompressible, viscous, Newtonian fluid in a horizontal cylinder rotating about the major axis, a simulation of reactive rotational molding of pipe.

I.A Rotational Reactive Molding

Two types of rotational molding processes are encountered commercially, thermoplastic powder and reactive liquid. The largest quantity of product is made with the thermoplastic system. In the powder thermoplastic system, the hollow split mold is, at the beginning, charged with the ground thermoplastics, e.g., polyethylene. Then, the mold is locked close and the powder is heated until it becomes a homogeneous melt, while the mold is rotating. The size of ground powder is usually smaller than 35 mesh, but varied according to material flow properties and product shape.

Since the process involves a transient solid-liquid boundary not definable by simple function of coordinates and heat transfer is equally as important as momentum transfer, it is beyond the scope of this study. The reactive system is similar with the exception of a viscous liquid being used as the charge. It is this system that this work is directed. The horizontal cylinder is the geometry encountered when rotational molding of reactive liquid system for manufacturing pipe. Thus, the term rotational molding will be used in this work to refer to the reactive liquid system in the following discussion.

I.A.1 Technology

The basic operation of reactive rotational molding includes three steps. The schematic diagram of the steps is shown in Fig. 1.

1. Loading

Materials of up to five components (1, 2) are placed in a two-piece mold.

- a. Resin - Thermosets, e.g., polyester, epoxy, and urethane
- b. Curing agent - copolymers containing double bonds, e.g., styrene and diallyl phthalate for polyester, and amines for epoxy
- c. Catalyst (initiator) - Peroxides, e.g., methyl ketone peroxide and benzoyl peroxide
- d. Promoter (accelerator) - Compounds to reduce the curing time at room temperature, e.g., cobalt naphthenate, cobalt octoate and p-toluenesulfonic acid
- e. Filler - Any material, e.g., glass fiber, cotton, wood flour, clay, coloring matter, etc., to enhance physical properties or to reduce the cost

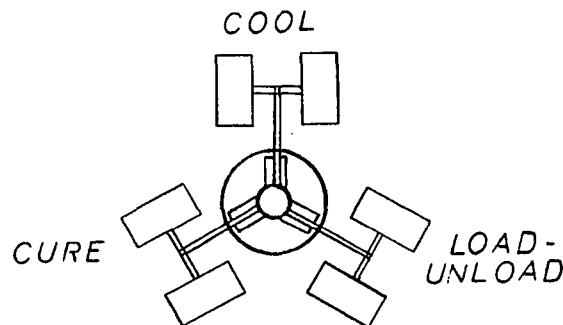


Fig. 1 Typical three-arm rotational molding unit shown in cross-section.

2. Curing

In general, the mold is rotated biaxially about its equatorial and polar axes. For cylindrical geometries the rotation is limited to a single axis, the center of the cylinder. The liquid withdraws from the pool and coats the inner surface of the mold with curing undergoing simultaneously. Curing is a free-radical cross-linking polymerization, causing resin to change from a liquid to solid. This process results in an increase in the viscosity of the resin.

During the process, two stages of phenomenal change can be recongnized (2).

- a. Setting time (Gel time) - From first initiation of free radicals to formation of soft gel
- b. Maturing time - From a soft gel to the final fully cured state

The rotating speed should be set such that the liquid resin uniformly coats the entire mold surface within the gel or setting time.

Curing is an exothermic reaction. The amount of heat liberated depends upon the type of resin, degree of unsaturation and type of copolymer. The heat release rate varies according to temperature, catalyst and promoter. After a physical dimension and a chemical system are selected, temperature becomes the only process variable. As noted in other polymerization processes, curing causes the resin to densify and the part will shrink. However, the heat of polymerization raises the temperature and stretches the polymer chain. If the temperature goes higher and

higher, the polymer chain will extend more and more. On the other hand, the curing rate will increase further and cause additional shrinkage. Such a condition may cause cracks. A high temperature also increases mobility of the fluid. A spotty weakened molding may be resulted. Because the mold is rotating, bulky, and enclosed, it is a difficult task to control the curing temperature by its surrounding environment. Frequently, it is the choice of chemical system that solves the control problem. The following factors are the most important considerations (1, 2):

- a. Degree of saturation - The more unsaturation present in the polymer, the higher the heat of curing, and the greater the shrinkage. A resin with high degree of unsaturation should cure at low temperature. The catalyst should accelerate the reaction at only moderate rate. When a high temperature and high reaction rate become necessary for a highly unsaturated resin, greater ratio of filler to resin may be added to absorb the heat and reduce the shrinkage so that cracking can be prevented.
- b. Size of batch - Resin is a poor conductor of heat. The temperature rise will increase as the batch size increases. Other than selecting a low temperature and a moderate catalyst, it is helpful to adjust the rotating speed so that the liquid coats the whole cylinder in a short time. This will increase the heat transfer area and help to uniformly spread the exotherm.

- c. Conductivity of mold - A metal mold has much greater thermal conductivity than a mold made from plastics. Aluminum and copper are most widely used materials for mold construction. When a plastic mold is used for development work, the mold is usually metalized or the plastic impregnated with metallic fillers to increase conductivity, and remove heat from the system to the surroundings.

3. Unloading

The final part can either be removed from the mold in a hot condition or after some cooling. The cooling step lengthens the cycle time. However, it provides better dimensional stability and improves properties.

I.A.2 Advantages

Rotational molding is primarily intended for the manufacture of larger objects usually made of metal and unable to be made by other plastic molding operations, e.g., injection molding, blow molding, etc. The advantages of the rotational molding process include (3):

1. Low equipment cost - It does not involve high operating pressure, high shear rate or precise metering of materials. The mold and tools are relatively low-cost and long-lasting.
2. Low operating cost - It does not involve high energy consumption for heating and compression. The scrap loss is minimal.
3. High production flexibility - It is conducive to low volume production of large articles. But it can be used to rotomold a large number of small items at a time, e.g., vinyl squeeze ear syringes. Simultaneous production of various products in one mold

is possible. It is easy to change materials and colors.

4. Special product quality - The resulting product has low residual molding stress, because no pressure is applied to the resin during process.

This study will focus on the fluid mechanics of withdrawal of the reactive liquid from the pool before curing starts. The analysis will also be limited to uni-axial rotation. The assumption is that extrapolation from one dimensional to three dimensional model is applicable in the liquid rotational molding as in powder thermoplastics molding (4).

I.B. Flow Phenomena

The flow phenomena of a horizontal rotating cylinder (5) are shown in Fig. 2. The process begins with a stationary cylinder with a pool of fluid on the bottom of the cylindrical cavity as shown in Fig. 2A. As the cylinder is rotated, the fluid adheres to the metal cylinder. Various flow regimes will be developed depending upon the angular velocity and fluid properties.

1. Withdrawl - At low rpm, a portion of liquid is drawn onto the wall, Fig. 2B.
2. Cascading - As rpm increases, the pool keeps moving upward. Some of the liquid releases from the wall and falls back to the bottom, Fig. 2C.
3. Rimming - Further increasing angular rotational velocity, the liquid is carried completely around the cylinder, Fig. 2D.
4. Collapsing - If the angular rotational velocity is decreased significantly from rimming velocity to an unstable flow, the fluid is again released to fall to the bottom, Fig. 2E.

5. Hydrocyst (Disking) - At angular velocity above that of rimming, stable thin disks perpendicular to rotating axis are formed. The disks may shift and appear at regular intervals, Fig. 2G. There is no evidence up to now that diskings occur in biaxial rotation (6), nevertheless this is the secondary flow phenomenon to be studied and avoided in the process.
6. Solid Body Rotation (SBR) - The angular rotational velocity is further increased and makes the liquid uniformly coat the cylinder, Fig. 2F.
7. Pool Rotation - If the angular rotational velocity is increased rapidly from a stationary condition, the pool will rotate, Fig. 2F.

There are other secondary fluid flow phenomena that also occur, e.g., air pumping and air entrainment. Air pumping is a phenomenon where an air layer is developed between the pool and the cylinder surface. Air entrainment is a phenomenon where air bubbles are encapsulated in the fluid layer by physical mixing. These bubbles are well distributed in the liquid.

It should be noted that the primary flow phenomena -- withdrawal, cascading, rimming and SBR -- can be found in any fluid. The secondary phenomena may be observed only in some fluids with a certain range of viscosity. For instance, the collapsing phenomenon happens mostly with low viscosity liquids such as water. Air pumping is, however, observed in highly viscous liquids.

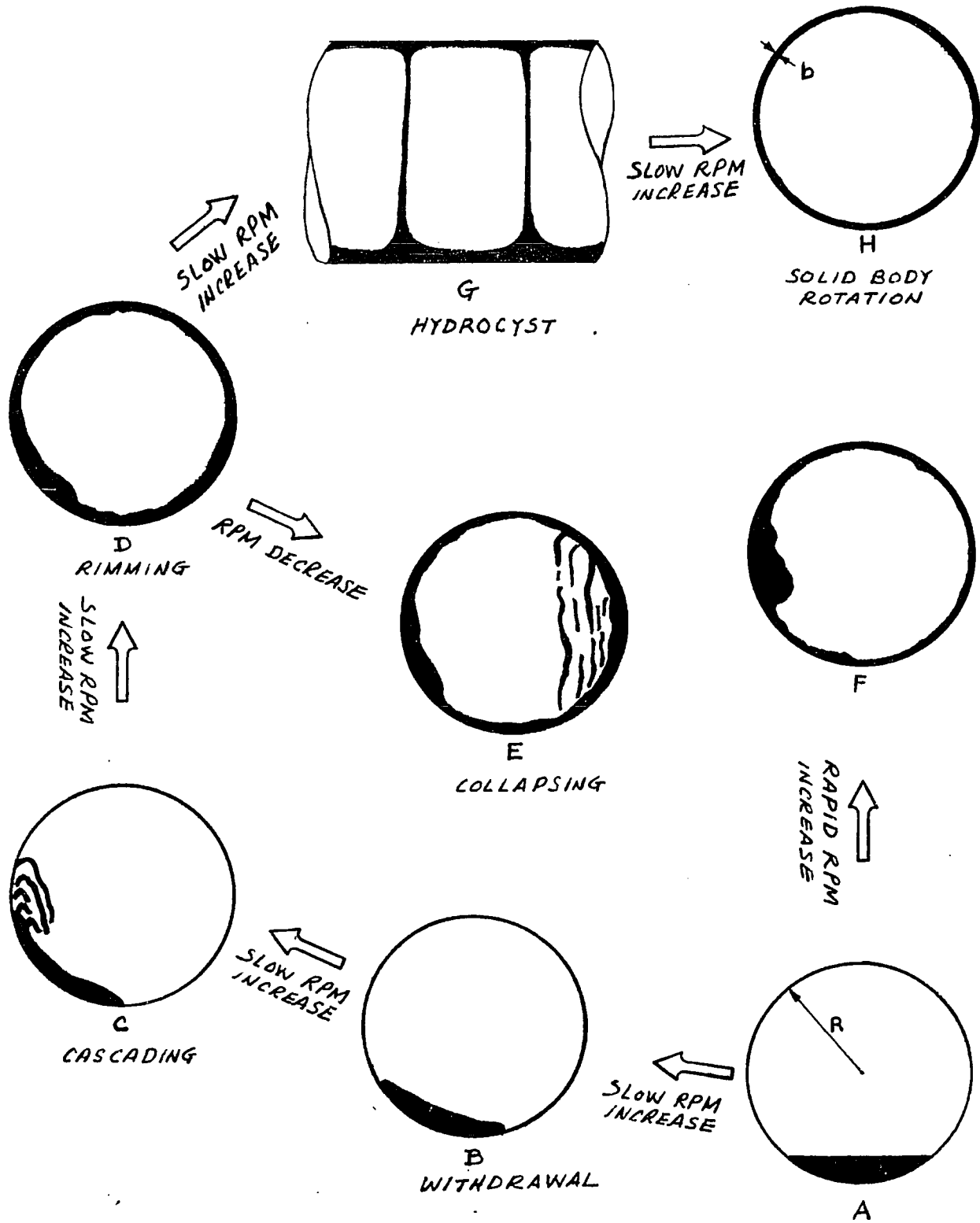


Fig. 2 Various flow phenomena for pool starting to rotate from quiescent state A

II THEORY

The general forms of continuity and momentum equations are the basis of viscous flow in a partially liquid-filled rotating cylinder. To analyze these equations coupled with initial and rigid boundary conditions has some difficulties (25):

1. The location of the free surface is supposed to be a boundary condition, but it is changing and not known a priori.
2. The shape of the free surface has nonlinear effects on the flow.
3. A stress singularity may be resulted at the intersection of free surface and rigid boundary.

Therefore, some investigators have directed their efforts to experimental examination, which is believed more likely to gain a full understanding of the transient process. However, theoretical work is necessary for interpreting the experimental results. The following is a summary of both the experimental and theoretical studies.

II.A. Analysis Of Previous Experimental Data

Progelhof et al. (5-11) analyzed previous experimental results (13- 18) and suggested a simple correlation for each flow regime.

For polyester with viscosity in the range of 200 - 7400 centipoise

$$\text{Cascading:} \quad Fr_m = 0.8 (Re_m)^{0.867} \quad (1)$$

$$\text{Rimming:} \quad Fr_m = 1.11 (Re_m)^{0.867} \quad (2)$$

$$\text{Solid Body Rotation:} \quad Fr_m = (2.37 - 0.026\psi) (Re_m)^{0.5} \quad (3)$$

$$\text{Hydrocyst:} \quad Fr_m = 0.285 (R/b) (Re_m) \quad (4)$$

For water-like liquids with viscosity less than 2 centipoise

$$\text{Cascading:} \quad Fr = 243 (Re)^{0.167} (1 - b/R)^{41.9} \quad (5)$$

$$\text{Rimming:} \quad Fr = 160 (Re)^{0.19} (1 - b/R)^{21.5} \quad (6)$$

$$\text{Collapsing:} \quad Fr = 249 (Re)^{0.062} (1 - b/R)^{31.4} \quad (7)$$

where Fr = Froude Number, $\omega^2 R^2 / gb$
 Fr_m = Modified Froude Number, $\omega^2 R / g$
 Re = Reynolds Number, $\omega R b / \nu$
 Re_m = Modified Reynolds Number, $\omega b^2 / \nu$
 b = Average film thickness, cm
 g = Gravitational acceleration, cm/sec^2
 R = Diameter of cylinder, cm
 ω = Speed of rotation, radians/sec
 ν = Kinematic viscosity, cm^2/sec

Recently, the interest in secondary flow behaviors has generated some data leading to conflicting correlations to describe the hydrocyst regime. When free surface effect was excluded, Progelhof et al. (7) proposed the following equations:

$$J = K_1 \quad Re_m < K_3 \quad (8a)$$

$$J = K_2 Re_m^N \quad Re_m > K_3 \quad (8b)$$

where J = Jay Number, $gb / \omega \nu$

K_1, K_2, K_3, N = constants

For polyester, $K_1 = 3.51, K_2 = 2.35, K_3 = 1.5, N=1$

The dimensionless group, Jay number, is defined as ratio of gravitational force ($b^3 \rho g / g_c$) to viscous force ($b^2 \eta w / g_c$). Balmer and Wang (18) concluded a complex correlation which was rearranged into the following form (8) in contrast with Eqs. 8a and 8b.

$$J = 2.995 (10^{-6}) Ca^3 (1-Vf)^{-1.216} (\eta / \rho^2)_{air} / (\eta / \rho^2) \quad (9)$$

where Ca = Capillary Number, $R\omega\eta/\sigma$

Vf = Volume fraction of liquid in cylinder

η = Absolute Viscosity, $dyne \text{ sec}/cm^2$

ρ = Density, gm/cm^3

σ = Surface tension, dynes/cm

II.B. Solution By Analytical Methods

Phillips (19) analyzed the momentum and continuity equations for an inviscid fluid by small perturbation theory. A criterion for stability of flow in a horizontal cylinder is found under steady gravity-induced disturbance:

$$\omega^2 R/g > 3/(1 - b/R) \quad (10)$$

The perturbation approximation is inapplicable under the following three conditions: (1) the film thickness is very small, (2) the flow is viscosity controlled, and (3) a wave motion of large amplitude is present. This criterion is equivalent to collapsing velocity and in agreement with other experimental data (13 - 15) based on water. Phillip's stability condition is shown in Fig. 3.

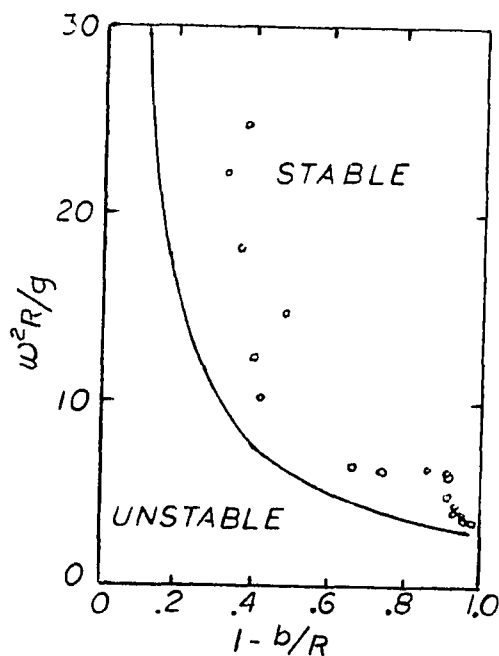


Fig. 3. Phillips' stability criterion.

(experimental data: water)

Cerro and Scriven (20) numerically solved the continuity and momentum equations for solid body rotation. With the condition of $b/R \ll 1$ (typical in rotational molding), a boundary layer approximation can be applied to these equations. Because of the unknown free surface position, the computation in a real coordinate system was very sensitive to minor changes in the boundary conditions and required considerable iterations to obtain the steady state solution.

Ruschak and Scriven (21) analyzed the steady flow as a regular perturbation from rigid body rotation. The continuity and momentum equations are transformed in terms of stream function and four dimensionless groups including Reynolds number, Froude number, Weber number and liquid loading. The general solutions is in a form a Bessel functions. Limiting cases are studied for extreme Reynolds numbers (Re_m) and small b/R . The location of thickest film shifts in the upper quadrant on the rising side, as shown in Fig. 4. The location of thinnest film is diametrically opposite.

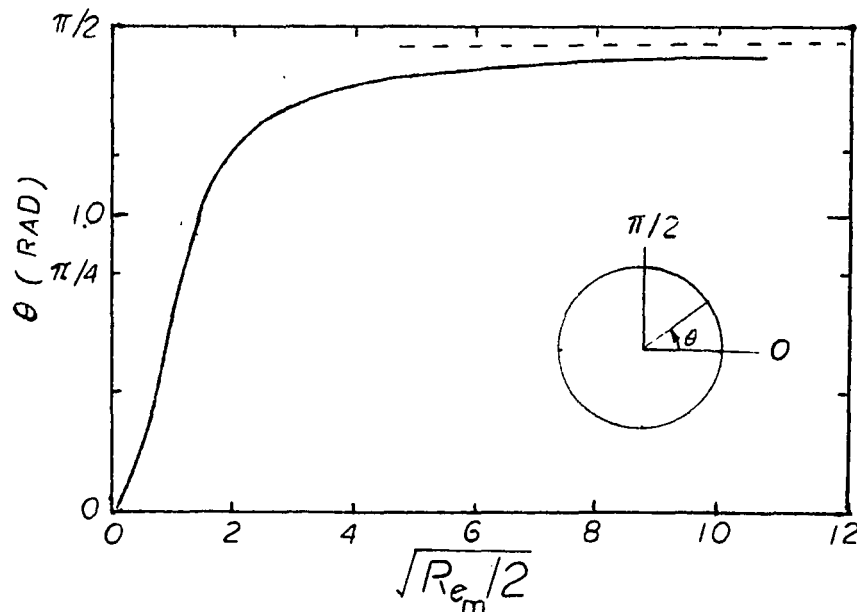


Fig. 4 Angular position, θ (in radians) of thickest film as function of Reynolds number, Re_m , in the limit $b/R \rightarrow 0$ when Fr_m is large.

Greenspan (22) also used perturbation methods to study a variant of this problem in which the free surface is replaced by a weightless mobile rigid surface. In terms of the Jacobian and stream functions, the equation is primarily solved for a rapidly rotating flow with large b/R ratio. The onset of instability is compared with Phillips' criterion in Fig. 5.

Gans (23) studied the effects of viscosity, nonlinear interaction and finite cylinder length on flow stability for a large Froude number (Fr_m) and small Ekman number (E) which is defined as $\nu/\omega R^2$. His results were comparable with Greenspan's observation (22) and Phillips' collapsing criterion (19).

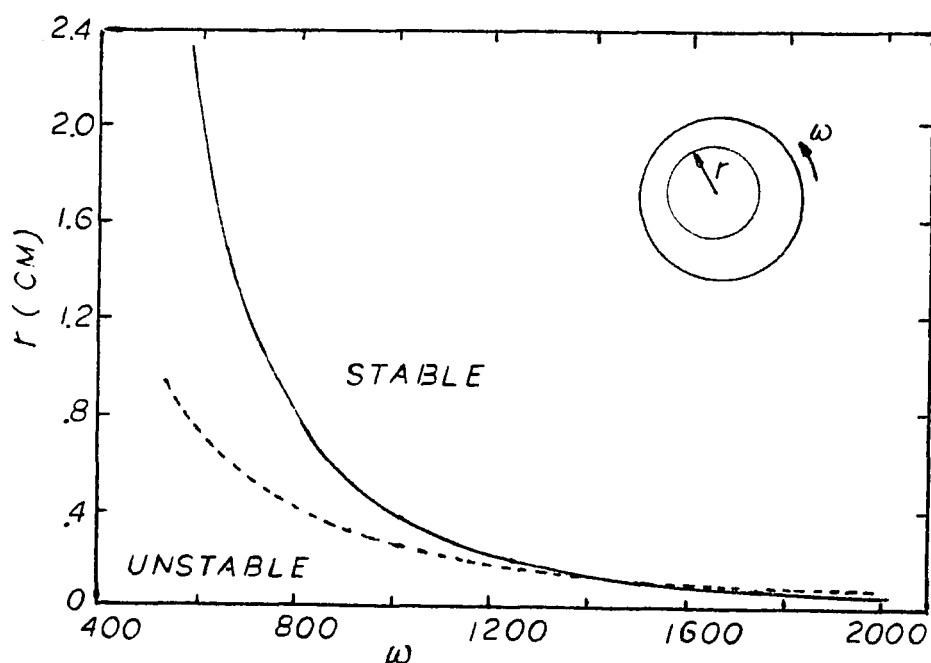


Fig. 5 Greenspan's stability criterion (solid curve) comparing Phillips' criterion (dashed curve).

Deiber and Cerro (24) transformed the continuity and momentum equations to a streamline coordinate system through a modified von Mises transformation. Using finite difference scheme, the numerical method is able to obtain a steady-state solution. It can predict both film thickness and velocity profile of rimming flow. It proposed also the following correlation of solid body rotation (7):

$$Fr_m = Re_m \quad (11)$$

which is different from Eq. 2. When the laminar circular streamline flow is broken, e.g., the fluid has a negative azimuthal velocity, the transformation is invalid. This region is defined discontinuous solution region which coincides the collapsing velocity. The general results are shown in Fig. 6.

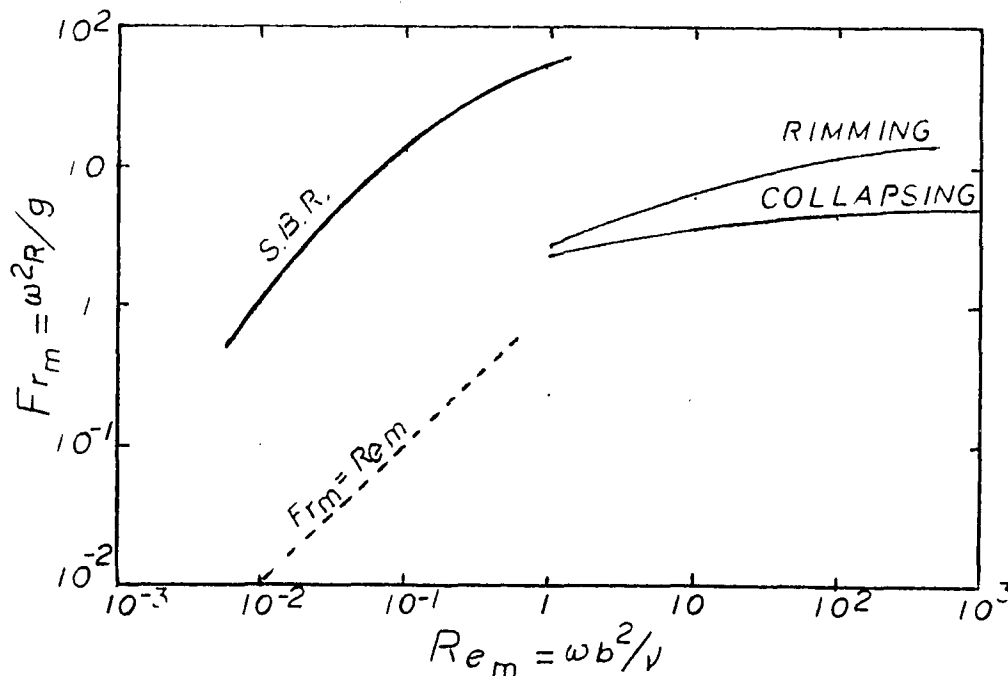


Fig. 6 Flow regimes of Deiber and Cerro, as function of Reynolds and Froude numbers.

Orr and Scriven (25) applied a finite element method to a primitive form, i.e., variables of velocity and pressure, instead of the vorticity-stream function system, of the continuity and momentum equations. A numerical solution of steady rimming flow which includes surface tension is developed. The elements need not have uniform size and shape. The Hermite cubic triangular element of Zlamal is chosen. The trial functions are cubic polynomials including ten terms of r and θ . To determine the location of free surface, the residual in the normal-stress boundary condition is used as the criterion. One example of the computation is shown in Fig. 7.

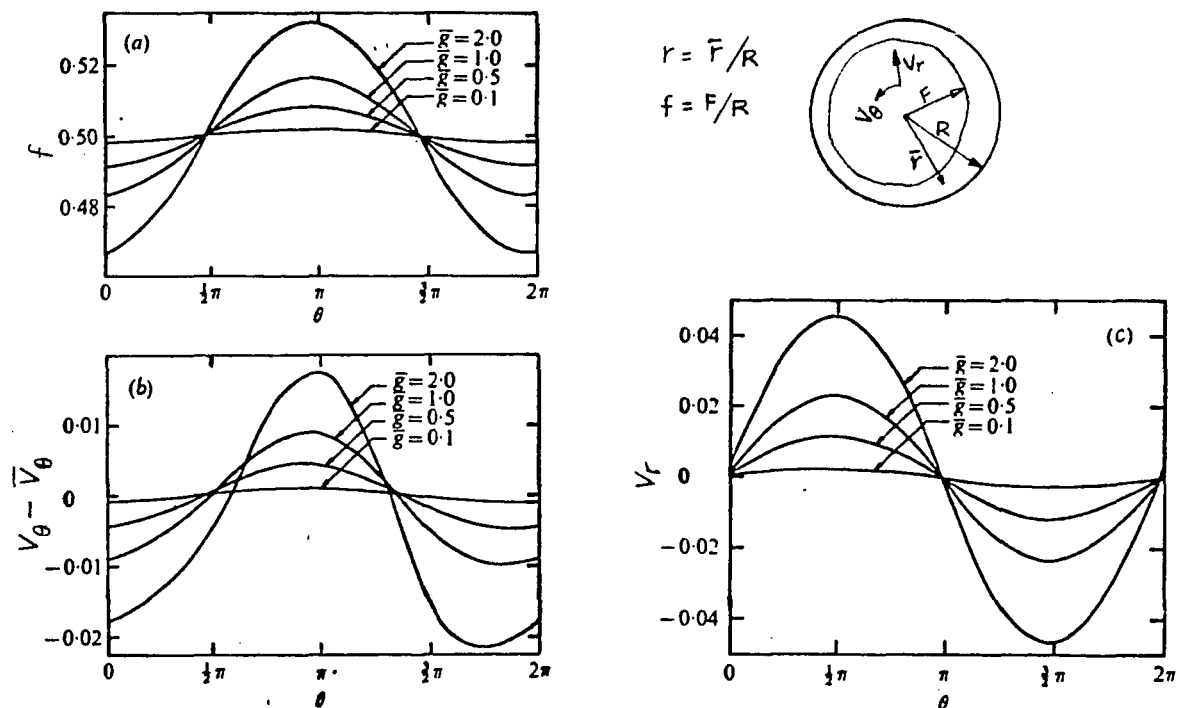


Fig. 7 Results of finite-element method, $Re_m (R/b)^2 = 1$. Illustrating the effect of increasing gravity on (a) the free-surface location, (b) the departure of the free-surface azimuthal velocity from SBR, $V_\theta - \bar{V}_\theta$, and (c) the radial velocity at free surface, V_r , at various angular positions, θ , with $\bar{g} = g/w^2 R = 1/Fr_m$.

The main drawback of the finite element method is that there is no clear indication of optimal combined choice of elements and trial functions.

All of the above mentioned analytical methods start with the continuity and momentum equations. Perturbation methods have proved useful for some cases of steady solutions. But numerical methods are capable to handle more varieties of the problem. The efforts also have been directed to steady state solutions. In the following analysis the transient solution will be developed.

III FORMULATION

A transient solution of a viscous liquid flow in a horizontal rotating cylinder is quite different from a steady rimming flow. Neither of the two features for steady flow mentioned by Orr and Scriven (25) is applicable. Those two features are (1) neither an inflow nor an outflow boundary existing and (2) no gas/liquid/solid contact line. The first feature avoids the complicated boundary conditions. The second one avoids the possibility of a singularity. The flow in this work is idealized by assuming the following conditions:

1. The fluid is incompressible and Newtonian.
2. The effect of surface tension is excluded.
3. The cylinder length is infinite. (No end effects are considered).

III.A Differential Equations

For a Newtonian, incompressible fluid at constant temperature, the momentum equations are the Navier-Stokes equations, which are usually expressed in vector form:

$$\frac{D\vec{u}}{Dt} = - \frac{1}{\rho} \nabla \bar{p} + \frac{\eta}{\rho} \nabla^2 \vec{u} + \vec{g} \quad (12a)$$

where

$$\rho = \text{Density, g/cm}^3$$

$$\bar{p} = \text{Pressure, dyne/cm}^2$$

$$\eta = \text{Newtonian viscosity, g/cm sec}$$

The equation of continuity is

$$(\nabla \cdot \vec{u}) = 0 \quad (13a)$$

By combining with Eq. 13a, Eq. 12a can be transformed into

$$\frac{\partial \vec{u}}{\partial t} + [\nabla \cdot \vec{u}\vec{u}] = - \frac{1}{\rho} \nabla \bar{p} + \frac{\eta}{\rho} \nabla^2 \vec{u} + \vec{g} \quad (12b)$$

Eq. 12b is the equation of motion that can be derived directly from momentum balance. Therefore, its finite difference form retains explicit momentum conservation. However, the difference form of Eq. 12a does not (34). The principle of momentum conservation allows no internal contributions to the time rate of momentum change in a space interval. The contrast in these two equations can be realized by summing over many cells in the difference forms of the one-dimensional equations of Eqs. 12a, and b, for example,

$$(u_i^0 - u_i) / \delta t + [u_i (u_{i+\frac{1}{2}} - u_{i-\frac{1}{2}}) + (P_{i+\frac{1}{2}} - P_{i-\frac{1}{2}}) / \rho] / \delta x = 0 \quad (12a')$$

$$(u_i^0 - u_i) / \delta t + [(u_{i+\frac{1}{2}}^2 - u_{i-\frac{1}{2}}^2) + (P_{i+\frac{1}{2}} - P_{i-\frac{1}{2}}) / \rho] / \delta x = 0 \quad (12b')$$

In the summation form, all the flux terms in Eq. 12b' can be cancelled in pairs, while some terms in Eq. 12a' can not. Since the finite difference approximation can not calculate space derivatives precisely, it becomes necessary to secure a better accuracy of approximation by using Eq. 12b instead of Eq. 12a as the starting point for deriving the difference equations. In Cartesian coordinates Eq. 12b can be written as

$$\frac{\partial u}{\partial t} + \frac{\partial(uu)}{\partial x} + \frac{\partial(uv)}{\partial y} + \frac{\partial(uw)}{\partial z} = -\frac{\partial p}{\partial x} + \nu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right] + g_x \quad (12c)$$

$$\frac{\partial v}{\partial t} + \frac{\partial(vu)}{\partial x} + \frac{\partial(vv)}{\partial y} + \frac{\partial(vw)}{\partial z} = -\frac{\partial p}{\partial y} + \nu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right] + g_y \quad (12d)$$

$$\frac{\partial w}{\partial t} + \frac{\partial(wu)}{\partial x} + \frac{\partial(wv)}{\partial y} + \frac{\partial(ww)}{\partial z} = -\frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right] + g_z \quad (12e)$$

The continuity equation is also expressed in Cartesian coordinates, as

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (13)$$

where u = X-direction velocity, cm/sec

v = Y-direction velocity, cm/sec

w = Z-direction velocity, cm/sec

p = Ratio of pressure to density, \bar{p}/ρ , cm^2/sec^2

ν = Kinematic viscosity, η/ρ , cm^2/sec

g_x, g_y, g_z = Gravitational acceleration in X, Y, Z
direction, respectively, cm/sec^2

Consider an infinitely long cylinder of radius R rotating around its Z axis (Fig. 8). Under this assumption the fluid will have no velocity component or functionality in the Z direction. Thus, 3-dimensional equations are simplified to a 2-dimensional system by eliminating all the z terms.

$$\frac{\partial u}{\partial t} + \frac{\partial(uu)}{\partial x} + \frac{\partial(uv)}{\partial y} = -\frac{\partial p}{\partial x} + \nu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right] + g_x \quad (14a)$$

$$\frac{\partial v}{\partial t} + \frac{\partial(vu)}{\partial x} + \frac{\partial(vv)}{\partial y} = -\frac{\partial p}{\partial y} + \nu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right] + g_y \quad (14b)$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (15)$$

These equations with initial and boundary conditions have not as yet been solved analytically. The numerical approximation methods, e.g., finite difference, can provide a practical solution.

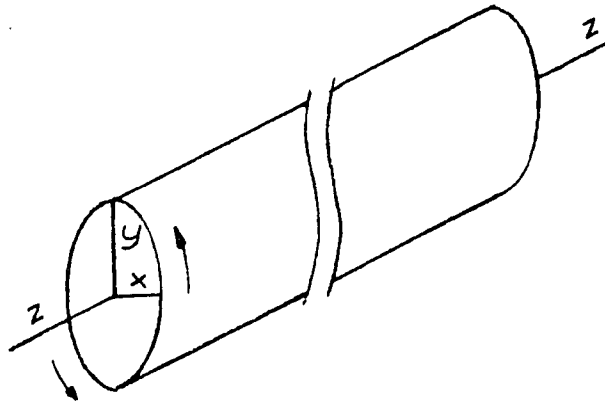


Fig. 8 Cartesian coordinates for a rotational cylinder.

III.B Finite Difference Method

The finite difference method is based upon elementary representations of the derivatives of a smooth function in terms of appropriate difference quotients (26). The Taylor series expansion is the basis of the approximation. For example, a function of two independent variables is expanded as:

$$\begin{aligned}
 u(x + \Delta x, y + \Delta y) &= u(x, y) + \frac{\partial u(x, y)}{\partial x} \Delta x + \frac{\partial u(x, y)}{\partial y} \Delta y \\
 &+ \frac{\partial^2 u(x, y)}{2 \partial x^2} \Delta x^2 + \frac{\partial^2 u(x, y)}{\partial x \partial y} \Delta x \Delta y + \frac{\partial^2 u(x, y)}{2 \partial y^2} \Delta y^2 \\
 &+ O(|\Delta x|^3 + |\Delta y|^3)
 \end{aligned} \tag{16}$$

where $O(|\Delta x|^3 + |\Delta y|^3)$ = Reminder of third order of magnitude

The first partial derivatives $\partial u / \partial x$ and $\partial u / \partial y$ are obtained by setting $\Delta y = 0$ and $\Delta x = 0$, respectively.

$$\frac{\partial u(x, y)}{\partial x} = \frac{u(x + \Delta x, y) - u(x, y)}{\Delta x} + O(\Delta x) \tag{17a}$$

$$\frac{\partial u(x, y)}{\partial y} = \frac{u(x, y + \Delta y) - u(x, y)}{\Delta y} + O(\Delta y) \tag{17b}$$

The difference approximation shown above is between the forward point and the original point. These representations of the derivative are referred to as forward differencing. The approximation can also be derived from backward differencing in which the Taylor expansion is given by

$$\begin{aligned}
 u(x - \Delta x, y - \Delta y) &= u(x, y) - \frac{\partial u(x, y)}{\partial x} \Delta x - \frac{\partial u(x, y)}{\partial y} \Delta y \\
 &+ \frac{\partial^2 u(x, y)}{2 \partial x^2} \Delta x^2 + \frac{\partial^2 u(x, y)}{\partial x \partial y} \Delta x \Delta y \\
 &+ \frac{\partial^2 u(x, y)}{2 \partial y^2} \Delta y^2 + O(|\Delta x|^3 + |\Delta y|^3)
 \end{aligned} \tag{18}$$

Then, the first derivatives are obtained by setting $\Delta y = 0$ or $\Delta x = 0$.

$$\frac{\partial u(x,y)}{\partial x} = \frac{u(x,y) - u(x - \Delta x, y)}{\Delta x} + 0 (\Delta x) \quad (19a)$$

$$\frac{\partial u(x,y)}{\partial y} = \frac{u(x,y) - u(x, y - \Delta y)}{\Delta y} + 0 (\Delta y) \quad (19b)$$

If Eq. 16 and Eq. 18 are added and $\Delta y = 0$ or $\Delta x = 0$ is set, the second derivatives are found

$$\frac{\partial^2 u(x,y)}{\partial x^2} = \frac{u(x + \Delta x, y) - 2u(x, y) + u(x - \Delta x, y)}{\Delta x^2} + 0 (\Delta x^2) \quad (20a)$$

$$\frac{\partial^2 u(x,y)}{\partial y^2} = \frac{u(x, y + \Delta y) - 2u(x, y) + u(x, y - \Delta y)}{\Delta y^2} + 0 (\Delta y^2) \quad (20b)$$

However, if combining Eqs. 16 and 18 without setting Δx or Δy equal to zero, the expression of mixed derivative $\partial^2 u / \partial x \partial y$ will yield

$$\begin{aligned} \frac{\partial^2 u(x,y)}{\partial x \partial y} \Delta x \Delta y = & \frac{1}{2} [u(x + \Delta x, y + \Delta y) + u(x - \Delta x, y - \Delta y) - 2u(x, y) \\ & - \frac{\partial^2 u(x,y)}{\partial x^2} \Delta x^2 - \frac{\partial^2 u(x,y)}{\partial y^2} \Delta y^2] + 0 (\Delta x^4 + \Delta y^4) \end{aligned} \quad (21)$$

Because the result of adding two Taylor expansions

$$\begin{aligned} u(x + \Delta x, y - \Delta y) + u(x - \Delta x, y + \Delta y) = & 2u(x, y) + \frac{\partial^2 u(x,y)}{\partial x^2} \Delta x^2 \\ & + \frac{\partial^2 u(x,y)}{\partial y^2} \Delta y^2 - 2 \frac{\partial^2 u(x,y)}{\partial x \partial y} \Delta x \Delta y + 0 (\Delta x^4 + \Delta y^4) \end{aligned} \quad (22)$$

Eq. 21 can be simplified by substituting Eq. 22 into Eq. 21.

$$\begin{aligned} \frac{\partial^2 u(x,y)}{\partial x \partial y} = & \frac{1}{4\Delta x \Delta y} [u(x + \Delta x, y + \Delta y) + u(x - \Delta x, y - \Delta y) \\ & - u(x + \Delta x, y - \Delta y) - u(x - \Delta x, y + \Delta y)] + 0 (\Delta x^2 + \Delta y^2) \end{aligned} \quad (23a)$$

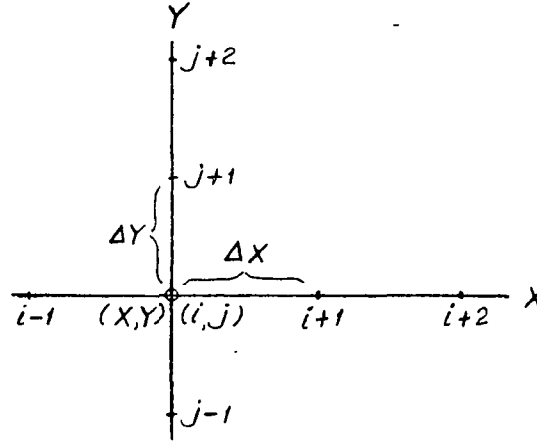
Eqs. 20a, 20b and 23a are referred to as central differencing. The first derivatives can also be expressed by central differencing technique by combining Eqs. 16 and 18.

$$\frac{\partial u(x,y)}{\partial x} = \frac{u(x + \Delta x, y) - u(x - \Delta x, y)}{2\Delta x} + 0 (\Delta x^2) \quad (24a)$$

$$\frac{\partial u(x,y)}{\partial y} = \frac{u(x, y + \Delta y) - u(x, y - \Delta y)}{2\Delta y} + 0 (\Delta y^2) \quad (24b)$$

When executing numerical computations, it is necessary to define a notation such that the individual mesh points are indexed. The lattice of mesh points used in this work is shown in Fig. 9.

Fig. 9.
Lattice of mesh points



The finite difference approximation of derivatives, Eqs. 17a, b, 19a, b, 20a, b, 23a and 24a, b yield the following expressions

$$(\partial u / \partial x)_{i,j} = [u(i+1, j) - u(i, j)] / \delta x \quad (17c)$$

$$(\partial u / \partial y)_{i,j} = [u(i, j+1) - u(i, j)] / \delta y \quad (17d)$$

$$(\partial u / \partial x)_{i,j} = [u(i, j) - u(i-1, j)] / \delta x \quad (19c)$$

$$(\partial u / \partial y)_{i,j} = [u(i, j) - u(i, j-1)] / \delta y \quad (19d)$$

$$(\partial^2 u / \partial x^2)_{i,j} = [u(i+1, j) - 2u(i, j) + u(i-1, j)] / \delta x^2 \quad (20c)$$

$$(\partial^2 u / \partial y^2)_{i,j} = [u(i, j+1) - 2u(i, j) + u(i, j-1)] / \delta y^2 \quad (20d)$$

$$(\partial^2 u / \partial x \partial y)_{i,j} = [u(i+1, j+1) + u(i-1, j-1) - u(i+1, j-1) - u(i-1, j+1)] / 4\delta x \delta y \quad (23b)$$

$$(\partial u / \partial x)_{i,j} = [u(i+1, j) - u(i-1, j)] / 2\delta x \quad (24c)$$

$$(\partial u / \partial y)_{i,j} = [u(i, j+1) - u(i, j-1)] / 2\delta y \quad (24d)$$

The indices will be written in the form of subscripts in the following discussion.

III.C Partial Donor Cell Formula

Following the conventional layout of variables and indices as shown in Fig. 10, there are at least three types of expressions that can be used to represent the momentum convective flux in the finite difference method (27).

Centered:
$$u_i = (u_{i-1/2} + u_{i+1/2})/2$$

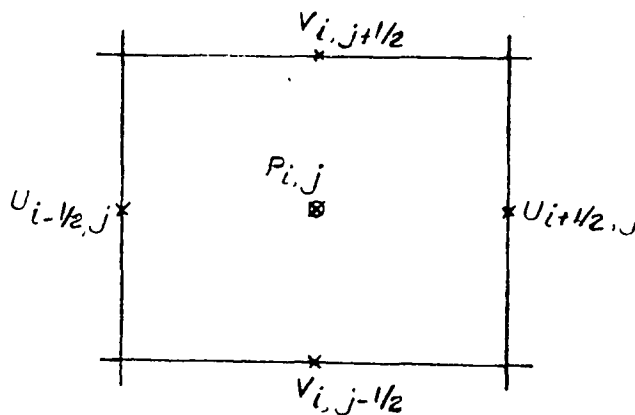
ZIP:
$$u_i = (u_{i-1/2} u_{i+1/2})^{1/2}$$

Donor:
$$u_i = [(1 + \bar{\alpha}) u_{i-1/2} + (1 - \bar{\alpha}) u_{i+1/2}] / 2$$

where $\bar{\alpha} = \frac{+}{-} 1$ Complete Donor

$1 > \bar{\alpha} > -1$ Partial Donor

Fig. 10. Conventional layout of variables and indices in the mesh.



The centered difference type has less truncation error, compared with the donor difference type, but has tendency to be numerically unstable. The ZIP technique is more accurate than the donor difference type and has better stability than the centered difference method. The donor approximation has the best stability, but the magnitude can be excessive and cause error, e.g., for flow at high Reynolds numbers. The transient fluid motion occurring in rotational molding has two characteristics.

First, it's a low Reynolds number flow and, secondly, it starts with a violent initiation in the early stage. This becomes a special circumstance to suit the donor cell flux. The partial donor cell formula providing a flexible combination of stability and accuracy is the choice for approximating the derivatives.

III.D Difference Equations (28)

The cross-section of cylinder is divided into rectangular cells of width δx and height δy (Fig. 11). The fictitious boundary cells (shaded area) are added to each side of the region to facilitate the computation. Since the fractional indices as shown in Fig. 10 are not allowed in a Fortran program, the indexing in Fig. 11 is written in the actual code. The density is assumed constant and, therefore, is lumped into the gravitational and viscosity terms. The pressure term is placed at the cell center. The velocity $u(i,j)$ is in the middle of right side of the cell, and $v(i,j)$ is at the middle of the top side, Fig. 12.

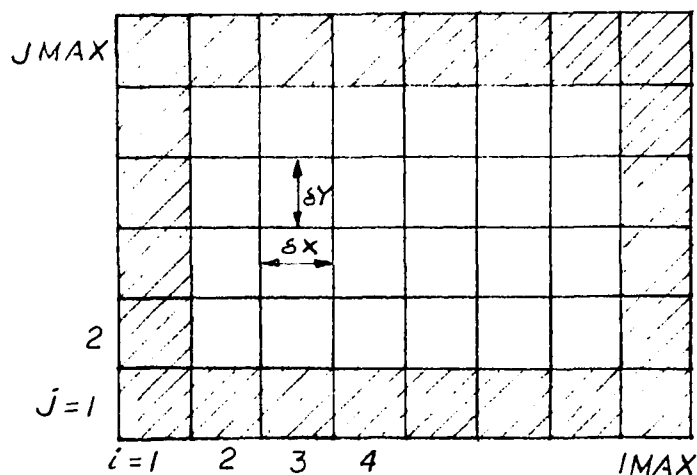


Fig. 11. Mesh indexing.

By the partial donor cell differencing method, Eqs. 14a, b and 15 are approximated by the finite difference forms:

$$u_{i,j}^{\circ} = u_{i,j} - \delta t [(p_{i+1,j} - p_{i,j})/\delta x - g_x + UUX + UVY - VISU] \quad (25a)$$

$$v_{i,j}^{\circ} = v_{i,j} - \delta t [(p_{i,j+1} - p_{i,j})/\delta y - g_y + UVX + VVY - VISV] \quad (25b)$$

$$(u_{i,j}^{\circ} - v_{i-1,j}^{\circ})/\delta x + (v_{i,j}^{\circ} - v_{i,j-1}^{\circ})/\delta y = 0 \quad (26)$$

where the momentum convective and viscous fluxes are approximated by

$$UUX = \partial uu/\partial x = [(u_{i,j} + u_{i+1,j})^2 + \alpha |u_{i,j} + u_{i+1,j}| (u_{i,j} - u_{i+1,j}) - (u_{i-1,j} + u_{i,j})^2 - \alpha |u_{i-1,j} + u_{i,j}| (u_{i-1,j} - u_{i,j})]/4\delta x \quad (27a)$$

$$UVY = \partial uv/\partial y = [(v_{i,j} + v_{i+1,j}) (u_{i,j} + u_{i,j+1}) + \alpha |v_{i,j} + v_{i+1,j}| (u_{i,j} - u_{i,j+1}) - (v_{i,j-1} + v_{i+1,j-1}) (u_{i,j-1} + u_{i,j}) - \alpha |v_{i,j-1} + v_{i+1,j-1}| (u_{i,j-1} - u_{i,j})]/4\delta y \quad (27b)$$

$$VISU = \nu (\partial^2 u/\partial x^2 + \partial^2 u/\partial y^2) = \nu [(u_{i+1,j} - 2u_{i,j} + u_{i-1,j})/\delta x^2 + (u_{i,j+1} - 2u_{i,j} + u_{i,j-1})/\delta y^2] \quad (27c)$$

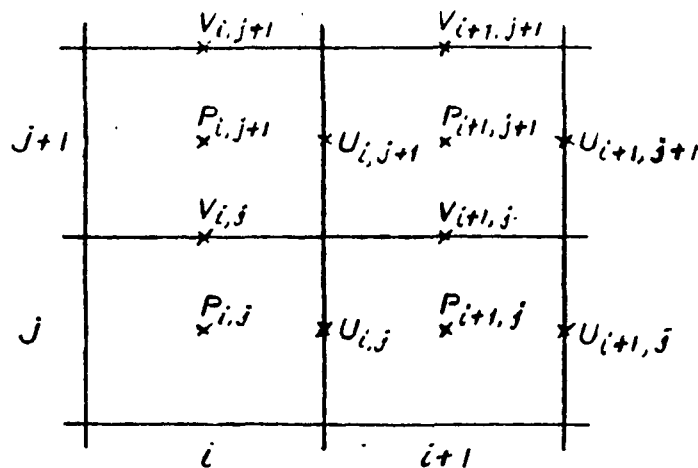


Fig. 12. Locations of cell variables in a mesh.

$$\begin{aligned}
UVX = \partial uv / \partial x = & [(u_{i,j} + u_{i,j+1}) (v_{i,j} + v_{i+1,j}) \\
& + \alpha |u_{i,j} + u_{i,j+1}| (v_{i,j} - v_{i+1,j}) - (u_{i-1,j} + u_{i-1,j+1}) \times \\
& (v_{i-1,j} + v_{i,j}) - \alpha |u_{i-1,j} + u_{i-1,j+1}| (v_{i-1,j} - v_{i,j})] / 4\delta x \quad (27d)
\end{aligned}$$

$$\begin{aligned}
VVY = \partial vv / \partial y = & [(v_{i,j} + v_{i,j+1})^2 + \alpha |v_{i,j} + v_{i,j+1}| \times \\
& (v_{i,j} - v_{i,j+1}) - (v_{i,j-1} + v_{i,j})^2 - \alpha |v_{i,j-1} + v_{i,j}| \times \\
& (v_{i,j-1} - v_{i,j})] / 4\delta y \quad (27e)
\end{aligned}$$

$$\begin{aligned}
VISV = \nu (\partial^2 v / \partial x^2 + \partial^2 v / \partial y^2) = & \nu [(v_{i+1,j} - 2v_{i,j} + v_{i-1,j}) / \delta x^2 \\
& + (v_{i,j+1} - 2v_{i,j} + v_{i,j-1}) / \delta y^2] \quad (27f)
\end{aligned}$$

The superscripts in the equations denote a new time level, i.e., at time $(n+1)\delta t$. The terms with no superscript are at the old time level, i.e., at time $n\delta t$. The coefficient α has a value from 0 to 1. The details of deriving Eqs 25a, b and 26 are included in Appendix I.

IV COMPUTER MODEL

The numerical model is based on SOLA-SURF algorithm (28), which is a simplified Marker-and-Cell (MAC) method (29). The MAC method was the first technique to use pressure and velocity as the primary variables to treat problems involving free surface. The method defines the field variables in an Eulerian mesh of calculational cells as shown in Fig. 12. The method uses weightless marker particles to specify the free surface. The pressure boundary condition is applied at the cell centers. In the later versions of the MAC method, the crude approximation of pressure is improved by satisfying the free surface stress conditions more accurately and in applying the pressure at the actual locations instead of cell centers (30). The marker particles can also be eliminated. The SOLA-SURF code includes both improvements. However, it is not directly applicable to the rotational molding geometry because of the following reasons:

1. The code is written in Eulerian concept of a fluid moving through a stationary network of cells, while a rotational cylinder may be better represented by a Lagrangian system, in which the fluid is covered by a mesh of cells whose vertices move with the fluid. Since the Eulerian technique is most useful for problems involving large fluid distortion, this study will modify the SOLA-SURF code, but keep this advantage for a moving mesh of cells.
2. The code is for uni-directional flow. In a rotational cylinder, the fluid moves along the wall. The flow field will be treated as the combination of two uni-directional flows.
3. The code is for a confined free surface with no advancing front, but on the contrary, the present problem is characterized by the

expansion of fluid covered area. Initial conditions have to be defined for each newly entered call.

4. Although cylindrical coordinates can be used in this code and makes the mathematics easier for the cells in the cylinder, it is not suitable physically for this transient problem. The cell size in cylindrical coordinate system varies, smaller for those cells near the center of the cylinder and larger for those cells near the wall. This is advantageous to the calculation of the free surface because of smaller cells. But the larger cell area is where the advancing front of fluid is entering. The large cell can not meet the requirement of high accuracy. Therefore, a Cartesian coordinates is used and, consequently, it is necessary to define the curve boundary.

The major considerations of modeling the Eqs. 25a, b and 26, including iteration procedure, initial conditions, boundary conditions and numerical stability, are discussed in the following sections.

IV.A Rigid Surface Location and Slope

The cell size is uniform throughout the whole cross-section area of the cylinder. The cells at a rigid boundary are partially inside the cylinder and partially outside. A criterion is necessary to determine if a specific boundary cell is "inside" the cylinder. When a boundary cell is considered "outside" the cylinder, it becomes the fictitious cell as shown in Fig. 11. In this calculation, if the inside fraction of the total cell area is greater than or equal to $\frac{1}{4}$, it is flagged "inside" (31). To calculate the inside area, a straight line (dotted line) connecting intersections of curve and Y-division lines is used in place on the curve boundary as shown in Fig. 13. The area of trapezoid GHED

approximates the bottom cell area at $X = i + 1$, and FGDC the bottom cell area at $X = i$. Both have an area greater than a quarter of the uniform cell area $\delta x \delta y$. Hence, both cells (i, j) and $(i + 1, j)$ are considered "inside" the boundary. The BFC area is smaller than $\delta x \delta y / 4$ and, therefore, the cell $(i - 1, j)$ is not inside the boundary. The bottom cell at $X = i - 1$ can be found to be $(i - 1, j + 1)$.

For a curved boundary, the fictitious cells are not outside a rectangle as shown in Fig. 11. Some of the fictitious cells may be at $j = 1$. Some may be at $j = 2, 3, 4$, etc. Likewise, they are not limited to $i = 1$ and IMAX. But every fictitious cell is, by definition, immediately next to the boundary cell.

At each boundary cell, the velocity of the boundary surface is known. This velocity vector can be viewed as a combination of the two Cartesian velocity components u and v , which can be calculated from the

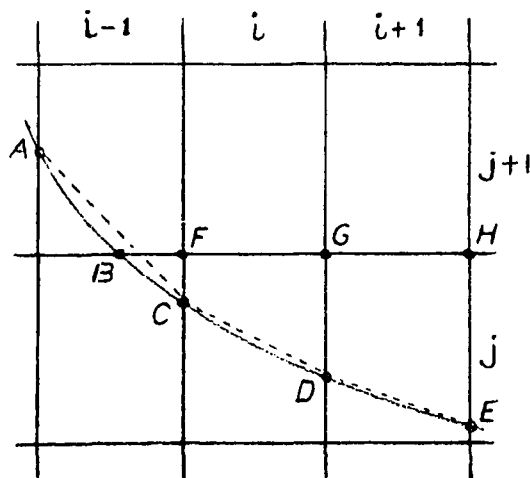


Fig. 13. Approximation of curved boundary.

equations.

$$BU_i = AS \cos (\tan^{-1} SLOB_i)$$

$$BV_i = AS \sin (\tan^{-1} SLOB_i)$$

where BU = u velocity at boundary, cm/sec

BV = v velocity at boundary, cm/sec

AS = Rotating velocity, cm/sec

SLOB = Tangential slope at boundary

The SOLA-SURF algorithm is restricted to the cells with slopes of the surface equal to or less than the aspect ratio of cell ($\delta y / \delta x$). If the slope is greater than the aspect ratio, the simple boundary conditions for the free surface in sec. IV.F.1 will not be applicable. The simple forms for approximating the free surface location in sec. IV.D will involve too much error. In a cylinder, it is predictable that the slope of free surface will increase rapidly as the cells approach either the left or right side boundary. This may be due to the steep slope of the rigid surface at both side boundaries. Therefore, an i column is excluded from the "inside" of cylinder if its rigid surface has a slope greater than $\delta y / \delta x$. This exclusion should not distort the general picture of cylinder. A 5% or less exclusion of the total area would be acceptable. Both the calculations of surface location and slope are outlined in Appendix II.

IV.B Sign Convention

The cylinder is divided into two parts: lower and upper halves. Since the i-indexing is following the direction of rotation, the indexing is from left to right for the lower half of the cylinder and right to left for the upper half of the cylinder if the rotation is counter-clockwise. U-velocity is positive (+) when it is following the indexing direction and

negative (-) when counter to the direction. The j -indexing is from bottom to the top. Both halves have the same indices. V -velocity is positive when it is upward and negative when it is downward. The gravitational constants g_x and g_y also have the same convention as u - and v -velocities. Hence, g_y being downward for all of the time has a negative sign. Fig. 14a shows these conventions. When a cell is involved in calculations of both halves, three points should be noted. First, it will have two i -indices, one for the lower half and another for the upper. Second, the u -velocity will have different signs: namely, positive for the lower half and negative for the upper, or vice versa. Third, the placing of u -velocity in the upper half is different from in the lower half because of the difference in rotation directions as shown in Fig. 14b.

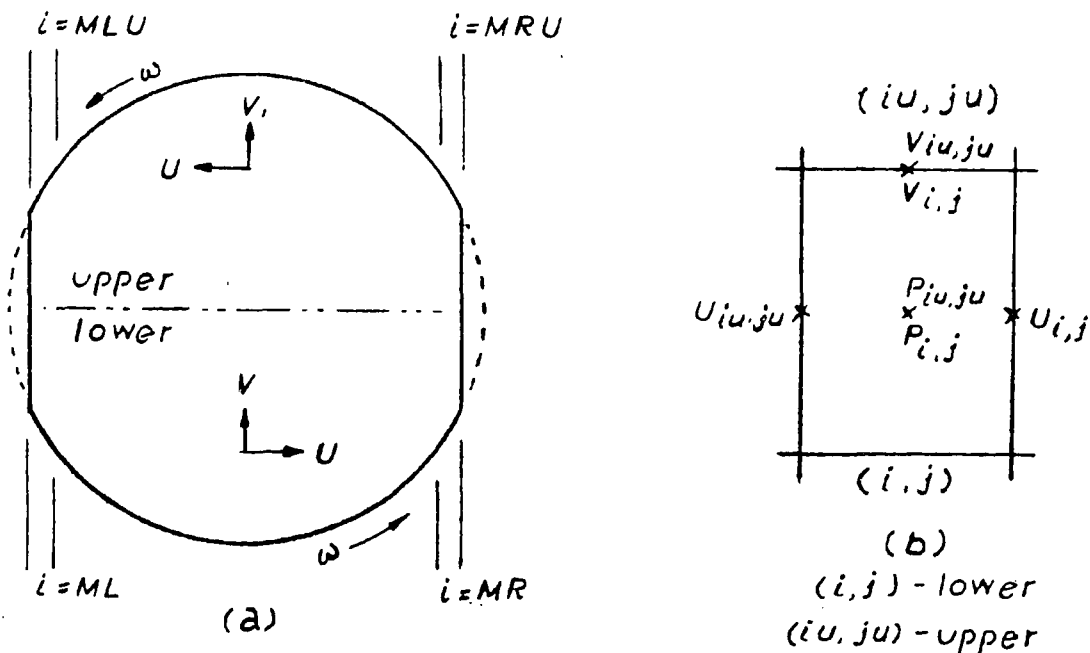


Fig. 14. Sign Convention.

IV.C Velocity Iteration

The velocity iteration procedure in SOLA-SURF is adopted in this model. In every time cycle, the velocities of each cell is predicted from Eqs. 25a, b. These predictions probably will not satisfy Eq. 26. The discrepancy is defined by

$$D = (u_{i,j} - u_{i-1,j})/\delta x + (v_{i,j} - v_{i,j-1})/\delta y \quad (28)$$

When absolute value of D is less than a specified value called tolerance, the iteration is terminated for that time cycle. Otherwise, the pressure is adjusted proportional to the discrepancy.

$$\delta p = -D/(2\delta t/\delta x^2 + 2\delta t/\delta y^2) \quad (29)$$

A new set of velocities is calculated according to the following equations:

$$u_{i,j} = u_{i,j} + \delta p \delta t / \delta x \quad (30a)$$

$$u_{i-1,j} = u_{i-1,j} - \delta p \delta t / \delta x \quad (30b)$$

$$v_{i,j} = v_{i,j} + \delta p \delta t / \delta y \quad (30c)$$

$$v_{i,j-1} = v_{i,j-1} - \delta p \delta t / \delta y \quad (30d)$$

Returning to Eq. 28 for convergency check, these new predictions start a new iteration. This procedure repeats until $D \leq$ tolerance is satisfied in every cell of the network. A new time cycle will, then, start. The derivations of Eqs. 29 and 30a - d are described in Appendix III.

It should be noted that Eq. 29 is applicable only to cells at interior mesh points. For cells at the free surface or a rigid boundary, the following equations are used:

Top Free Surface

$$\delta p = (1 - \eta_T) p_{i,jT-1} - p_{i,jT} \quad (31a)$$

where $\eta_T = \delta y / [H_i - (jT - 2.5)\delta y]$

$jT = Y$ - index of top cell

H_i = height of top surface from mesh baseline

Bottom Free Surface

$$\delta p = (1 - \eta_B) p_{i,jB+1} - p_{i,jB} \quad (31b)$$

where $\eta_B = \delta y / [(jB - 0.5)\delta y - HB_i]$

$jB = Y$ - index of bottom cell

$HB_i =$ height of bottom surface from mesh baseline

Top Rigid Surface

$$\delta p = [v_{i,jT-1} - (u_{i,jT} + u_{i-1,jT}) (H_{i+1} - HB_{i-1}) / 4\delta x - \lambda_T (u_{i,jT} - u_{i-1,jT}) \delta y / \delta x] / [(1 + 2\lambda_T \delta y^2 / \delta x^2) \delta t / \delta y] \quad (31c)$$

where $\lambda_T = (jT - 2) - H_i / \delta y$

Bottom Rigid Surface

$$\delta p = [v_{i,jB-1} - (u_{i,jB} + u_{i-1,jB}) (H_{Bi+1} - HB_{i-1}) / 4\delta x - \lambda_B (u_{i,jB} - u_{i-1,jB}) \delta y / \delta x] / [(1 + 2\lambda_B \delta y^2 / \delta x^2) \delta t / \delta y] \quad (31d)$$

where $\lambda_B = (jB - 1) - HB_i / \delta y$

The derivations in details of Eqs. 31a - d are explained in Appendix IV.

IV.D Free Surface Position

After the velocity field converges in every cell, a time cycle is concluded with a set of new free surface positions. The movement of free surface can be calculated according to the kinematic equations, which involve velocities and surface slope. The velocities $u(i,j)$ and $v(i,j)$ for a particular cell are assumed applying to the center of the surface, 0. The particle at 0 moves to an arbitrary point D during the time period t , as shown in Fig. 15(a). Paralleling to the original surface, a line CD is drawn as shown in Fig. 15(b). The net displacement of the free surface is from 0 to C. By a geometrical relationship,

$$\begin{aligned} OC &= ED = AD - AE \\ &= OB - OA (\Delta H / \delta x) \end{aligned}$$

That is equivalent to

$$H_i = v \delta t - u \delta t (\Delta H / \delta x) \quad (32)$$

where u, v = mid-surface velocities

Eq. 32 is expressed in finite difference form to obtain the equation for top free surface.

$$\begin{aligned} H_i^o = H_i + \delta t \left\{ -[(u_{i,jT} + u_{i-1,jT}) (H_{i+1} - H_{i-1}) \right. \\ \left. - \gamma |u_{i,jT} + u_{i-1,jT}| (H_{i+1} - 2H_i + H_{i-1})] / 4\delta x \right. \\ \left. + \lambda_T v_{i,jT} + (1 - \lambda_T) v_{i,jT-1} \right\} \end{aligned} \quad (33a)$$

where γ = weight of partial donor cells

λ_T = ratio of liquid height to δy

Similarly, the equation for bottom free surface is

$$\begin{aligned} HB_i^o = HB_i + \delta t \left\{ -[(u_{i,jB} + u_{i-1,jB}) (HB_{i+1} - HB_{i-1}) \right. \\ \left. - \gamma |u_{i,jB} + u_{i-1,jB}| (HB_{i+1} - 2HB_i + HB_{i-1})] / 4\delta x \right. \\ \left. + (1 - \lambda_B) v_{i,jB} + \lambda_B v_{i,jB-1} \right\} \end{aligned} \quad (33b)$$

where λ_B = ratio of void height to δy

The derivation of Eqs. 33a, b are described in Appendix V.

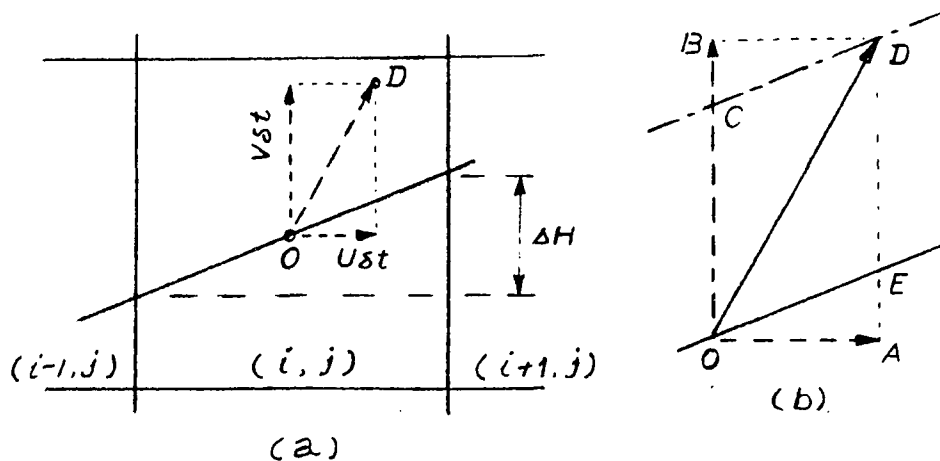


Fig. 15 Kinematic equation for free surface location

IV.E Initial Conditions

The initial conditions includes (1) the top free surface before the cylinder rotation is started, (2) the velocities and pressures at the moment the cylinder is starting to rotate, and (3) the velocities and pressures for the newly entered cells.

IV.E.1 Initial Top Free Surface

The location of the initial top free surface can be specified by either the most left or the most right contact point of the surface with the wall. For example, in Fig. 16(a), the distance of the most left point from $i = 0$ is

$$TX = AR - [AR^2 - (AR - FLHT)^2]^{1/2} \quad (34)$$

where AR = radius of cylinder, cm

$FLHT$ = initial fluid height in the cylinder, cm

Since this algorithm is inapplicable to the i column of which the thickness of liquid layer is limited in one single cell, e.g., columns i_1 and i_2 in Fig. 16(b). The most left column for calculation, IL , may not be exactly at TX . IL is the first i column from the left with thickness of two or more cells.

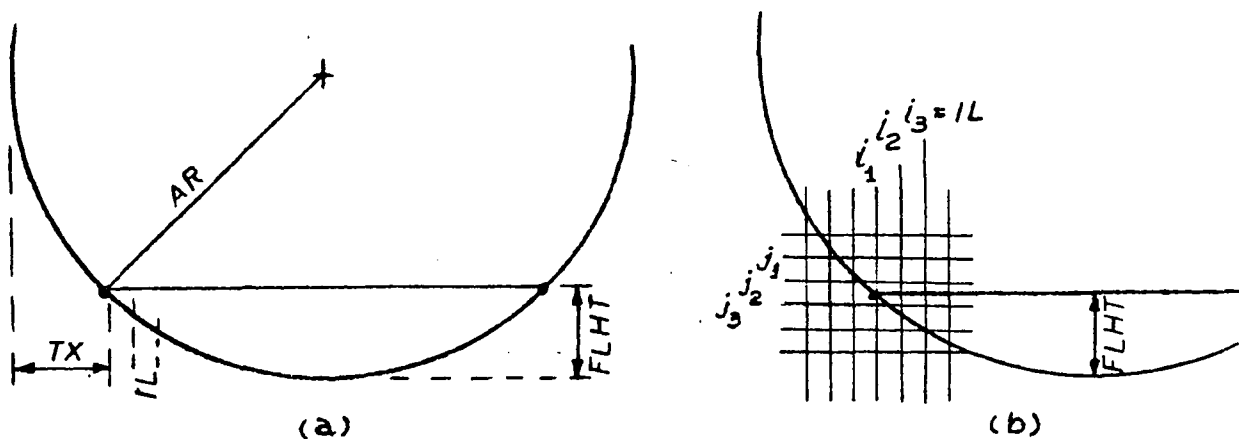


Fig. 16 Initial top free surface.

IV.E.2 Initial Velocities and Pressures

When the cylinder starts to rotate, it is assumed that only the cells at the boundary layer move. Accordingly,

$$u_{i,jB} = BU_i \quad (35a)$$

$$v_{i,jB} = BV_i \quad (35b)$$

The velocities at all other cells are assumed to be zero.

However, every cell filled with liquid completely or partially should have a pressure value. The initial pressure is the hydrostatic pressure.

$$P_{i,j} = -g_y [H_i - (j - 1.5)\delta y] \quad (35c)$$

IV.E.3 Initial Conditions for Newly Entered Cells

After the new free surface is calculated by Eq. 33b for the upper half, velocities and pressure have to be assumed for these newly entered cells so that the new velocities can be predicted from Eqs. 25a, b in the next time cycle.

$$u_{i,j} = (u_{i+1,j} + u_{i-1,j} + u_{i,j+1} + u_{i,j-1})/4 \quad (37a)$$

$$v_{i,j} = (v_{i+1,j} + v_{i-1,j} + v_{i,j+1} + v_{i,j-1})/4 \quad (37b)$$

$$P_{i,j} = g_y [(j - 1.5)\delta y - HB_i] \quad (38)$$

When the calculation is in the lower half, these initial conditions are optional. The velocity iteration is able to converge with the assumption of zero pressures and zero velocities at the newly entered cells.

IV.E.4 Initial Conditions for Newly Entered Column

The assumed velocities in newly entered columns of cells are 2nd and 4th quadrants:

$$u_{i,j} = u_{i-1,j} \quad (39a)$$

$$v_{i,j} = v_{i-1,j} \quad (39b)$$

1st and 3rd quadrants:

$$u_{i,j} = 2u_{i-1,j1} - u_{i-2,j2} \quad (40a)$$

$$v_{i,j} = 2v_{i-1,j1} - v_{i-2,j2} \quad (40b)$$

where $j1$ and $j2$ have the same distance to the top rigid surface in $i-1$ and $i-2$ columns, respectively, as j in i column. Fig. 17 illustrates the relations among j , $j1$ and $j2$. The pressures in the new column are calculated by Eqs. 36 and 38.

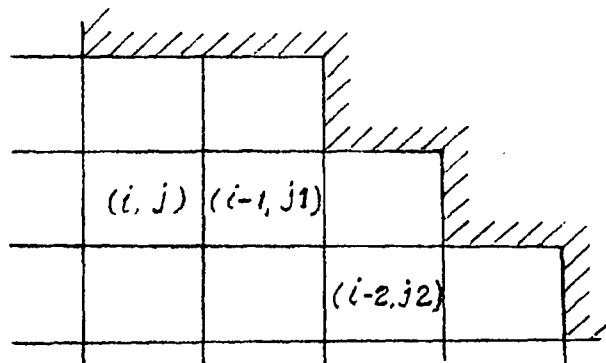


Fig. 17 Relative positions of j , $j1$ and $j2$.

IV.F Boundary Conditions

The boundary conditions include velocities at the boundary cells, the fictitious cells and the empty cells surrounding the free surface. A no-slip boundary condition can be applied to force the tangential velocities of the fluid at the walls to be identical to the rotational velocity at the walls. For a free surface, the continuative boundary condition will make the adjacent cell outside the fluid to have the same velocities as the cell inside the fluid.

IV.F.1 Free Surface

During the propagation of fluid movement, the image cells $(i-1, j)$ of the front wave are assigned the velocities following continuative boundary condition.

$$v_{i+1,j} = v_{i,j} \quad (41a)$$

$$u_{i+1,j} = u_{i,j} \quad (41a)$$

For both top and bottom free surfaces, the fluid rarely occupies a full surface cell. Thus, v velocities can be calculated from the continuity equation.

Top Free Surface

$$v_{i,j} = v_{i,j-1} - (u_{i,j} - u_{i-1,j})\delta y/\delta x \quad (42a)$$

$$u_{i,j+1} = u_{i,j} \quad (42b)$$

Bottom Free Surface

$$v_{i,j-1} = v_{i,j} + (u_{i,j} - u_{i-1,j})\delta y/\delta x \quad (43a)$$

$$u_{i,j-1} = u_{i,j} \quad (43b)$$

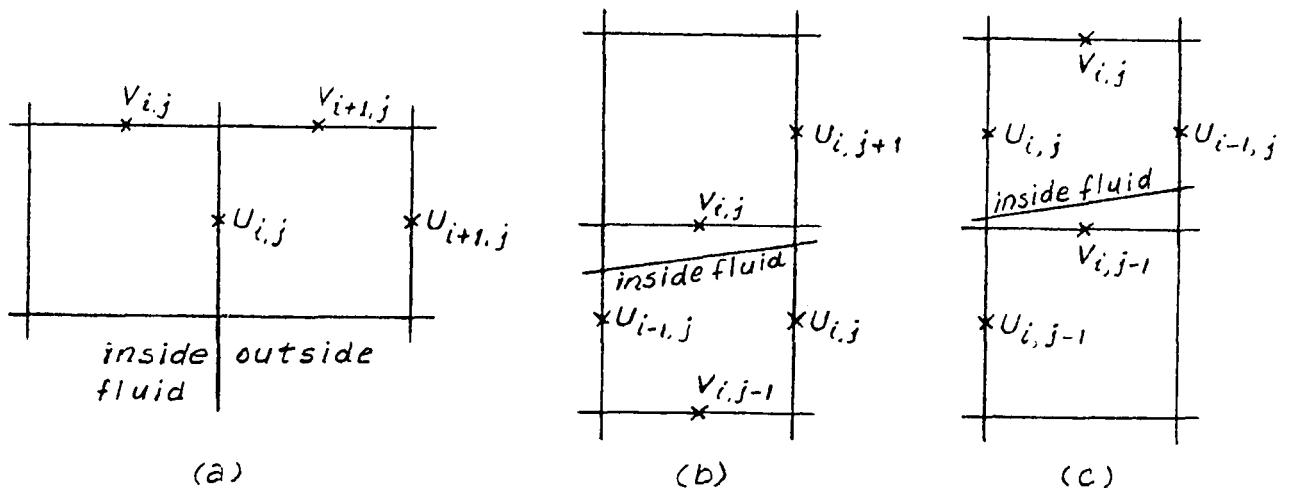


Fig. 18. Free surface boundary conditions (a) Eqs. 41a, b,

(b) Eqs. 42a, b, (c) Eqs. 43a, b.

IV.F.2 Rigid Boundary

After excluding some cells from the calculation scheme as discussed in sec. IV.A, the cylinder is no longer complete circular, but a circular segment with two vertical flat portions on its right and left hand sides. It now becomes necessary to assume a rotating film occupying no space inside the wall and to make the cylinder stationary. This rotating film replaces the rotational cylinder. A Lagrangian system, thus, can be treated as a Eulerian system. Then, the horizontal velocities at the most right column for lower half and the most left column for upper half are zero. This forces the liquid to stop at the vertical walls. The vertical velocities may be assumed equal to the rotating speed of the cylinder.

Lower half:

$$u_{MR,j} = 0 \quad (44a)$$

$$v_{MR,j} = AS \quad (44b)$$

Upper half:

$$u_{MLU,j} = 0 \quad (45a)$$

$$v_{MLU,j} = -AS \quad (45b)$$

where MR = The most right column in the lower half

MLU = The most left column in the upper half

However, a better approach to obtain the v-velocity is by assuming a linear relation in the near wall area, as shown in Fig. 19(a).

$$v_{MR,j} = (2AS + v_{MR-1,j})/3 \quad (44c)$$

$$v_{MLU,j} = (-2AS + v_{MLU-1,j})/3 \quad (45c)$$

For cells outside the vertical walls, the following velocities are assumed:

$$u_{MR+1,j} = 0 \quad (46a)$$

$$v_{MR+1,j} = 2v_{MR,j} - v_{MR-1,j} \quad (46b)$$

$$u_{MLU+1,j} = 0 \quad (47a)$$

$$v_{MLU+1,j} = 2v_{MLU,j} - v_{MLU-1,j} \quad (47b)$$

For a bottom rigid boundary as shown in Fig. 19(b), $v_{i,j-1}$ is outside the wall. It can be calculated by equation of continuity, Eq. 43a.

As the rotating velocity is the combination of u and v velocities,

$$AS = v_{i,j-1} \sin \theta + u_{i,j-1} \cos \theta$$

$u_{i,j-1}$ is computed by

$$u_{i,j-1} = (AS - v_{i,j-1} \sin \theta) \cos \theta \quad (48)$$

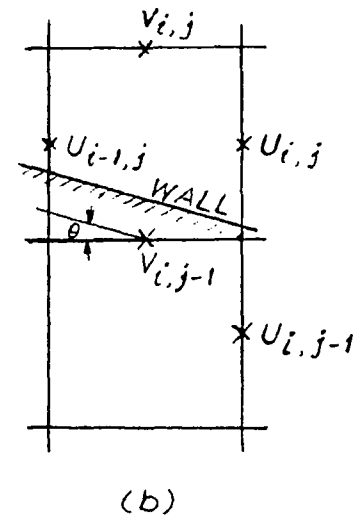
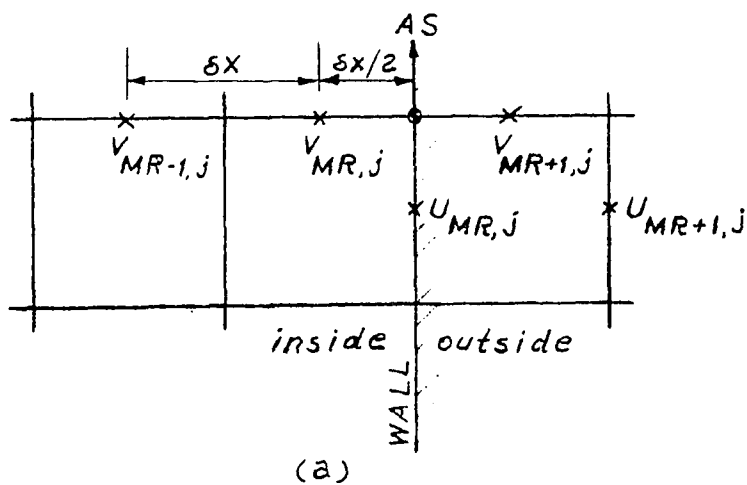


Fig. 19. Rigid boundary conditions.

IV.G Parameters and Numerical Stability

To insure numerical stability some parameters must be correctly chosen, e.g., the weights of partial donor cell method. Though some rules of thumb have been suggested, an optimum value can be achieved only by trial and error.

IV.G.1 α, γ - Weight of Partial Donor Cell

In Eqs. 25a, b, α has value in the range of 0 to 1 as stated previously. It is limited by the inequality

$$1 \geq \alpha = k \max (|u\delta t/\delta x|, |v\delta t/\delta y|) \quad k > 1 \quad (49)$$

A good choice of k is suggested 1.2 to 1.5 (28).

γ in Eqs. 33a and b is often equal to α . Its only constraint is

$$1 \geq \gamma \geq 0 \quad (50)$$

The example problem in section IV.I demonstrates the allowable difference between values of α and γ , with $\alpha = 0.9$ and $\gamma = 0.2$.

IV.G.2 OMG - Over-Relaxation Factor

To accelerate the convergence of velocity iteration, OMG is incorporated into Eq. 29.

$$\begin{aligned} p &= -(OMG) (D)/2\delta t(1/\delta x^2 + 1/\delta y^2) \\ &= -(BETA) (D) \end{aligned} \quad (51)$$

where $BETA = OMG/2\delta t(1/\delta x^2 + 1/\delta y^2)$

OMG is in the range of 1 to 2. If it is 2, the iteration will become unstable. A value of 1.8 is often the optimum.

IV.G.3 ϵ - Tolerance of Discrepancy

The tolerance used as the criterion of convergence test for Eq. 28 should be small enough to ensure satisfying the continuity equation. A value of 10^{-3} is typical for non-dimensional velocity in the order of 0.1 - 1. If the value is too small, the improvement of accuracy is

minimal. But the computer time becomes excessive. For this study, a value up to 0.1% of the average velocity is adequate.

IV.G.4 δt - Time Increment

The time increment is the most important factor in stability.

There are at least two criteria to be satisfied (28).

$$\delta t < \min (|\delta x/u|, |\delta y/v|) \quad (52a)$$

$$\delta t < \delta x^2 \delta y^2 / 2 \nu (\delta x^2 + \delta y^2) \quad (52b)$$

Another criterion suggests to include a parameter $(\phi + 1)$ in denominator of inequality 52b.

$$\delta t < \delta x^2 \delta y^2 / 2(\phi + 1) \nu (\delta x^2 + \delta y^2) \quad (52c)$$

where the value of ϕ is in the range of 0 - 2 (30).

IV.H Computation Procedures

The input data of the calculation defines the block boundary (IMAX, JMAX), cell size (DELX, DELY), initial liquid height (FLHT), rotation velocity (AS), kinematic viscosity (NU) and gravitational acceleration (GX, GY). It includes also the above mentioned parameters ω , γ , $\Omega M G$, ϵ , and time increment. Then, the actual rigid boundary, the mesh of cells and initial free surface are drawn. Both initial profiles of velocity and pressure are set up.

A section computation is followed to calculate the fluid moving from left of right in the lower half. When the fluid reaches the right boundary, it climbs up along the wall up to the top right boundary. This is the computation in "lower section I" shown in Fig. 20. The velocities and pressure of the cells at the front of the wave are re-defined for the "upper section". Thus, the computation in upper half starts. This section is for liquid moving from right to left with a bottom free surface.

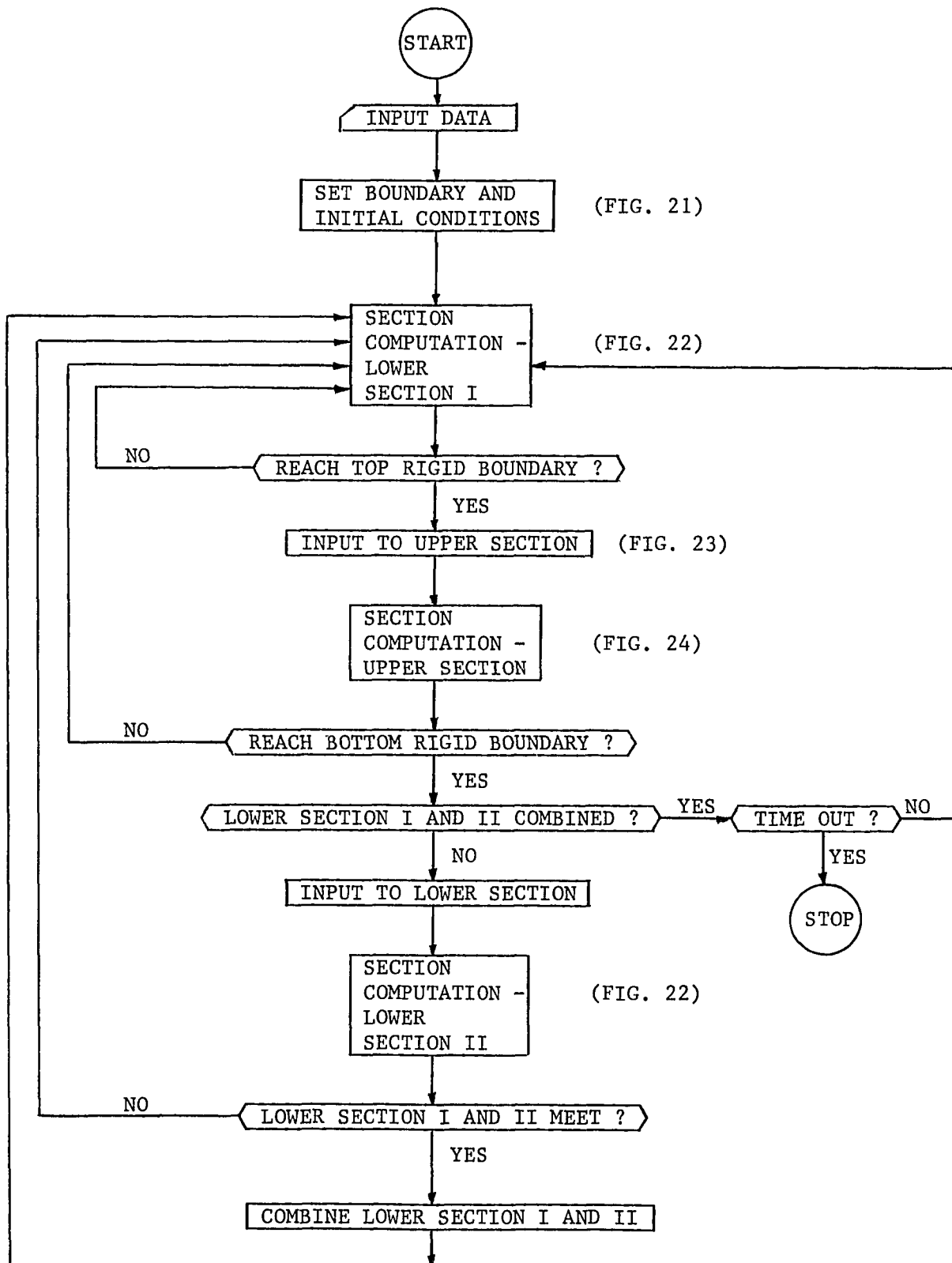


Fig. 20 Overall flow chart.

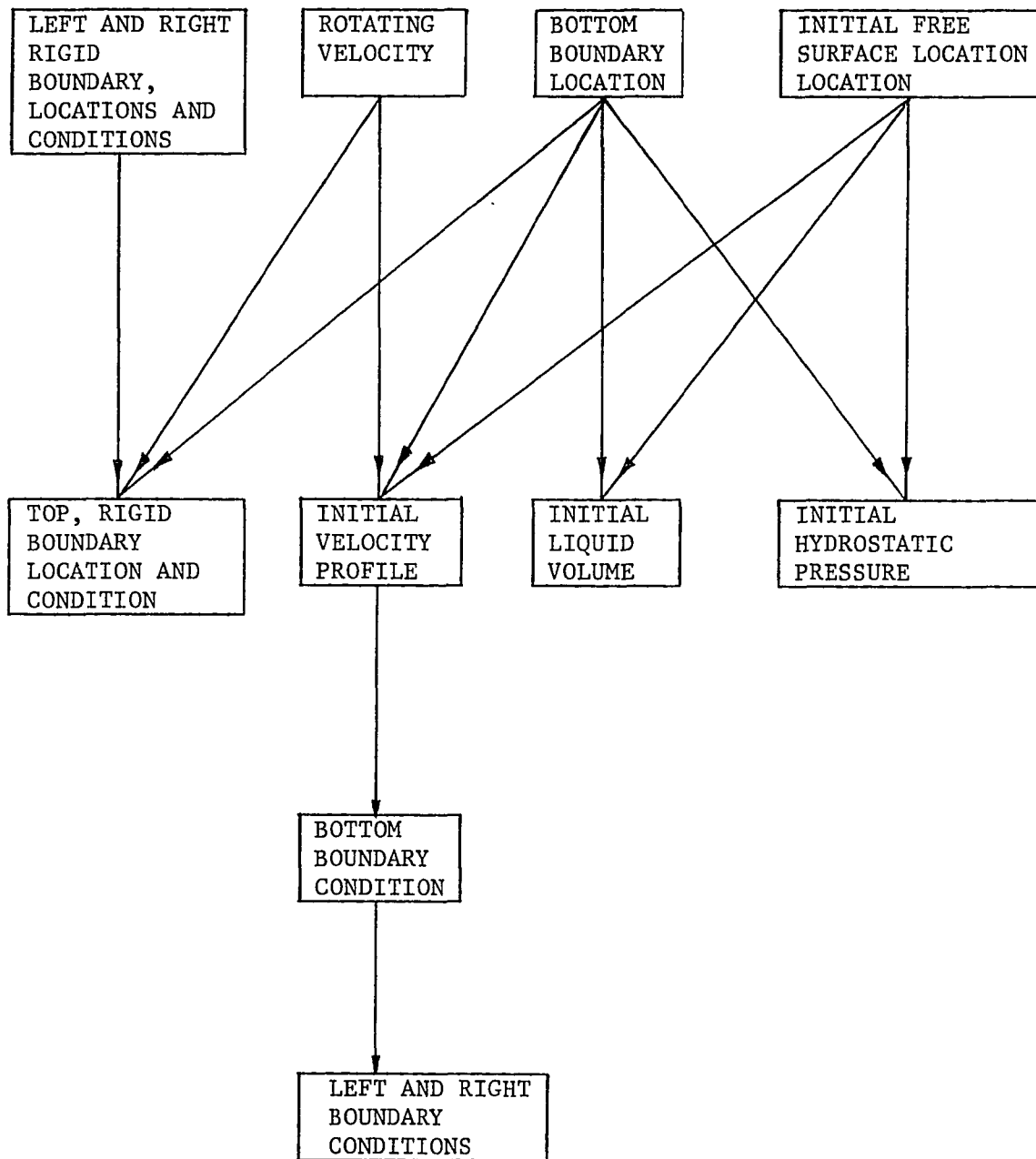


Fig. 21. Boundary and initial conditions

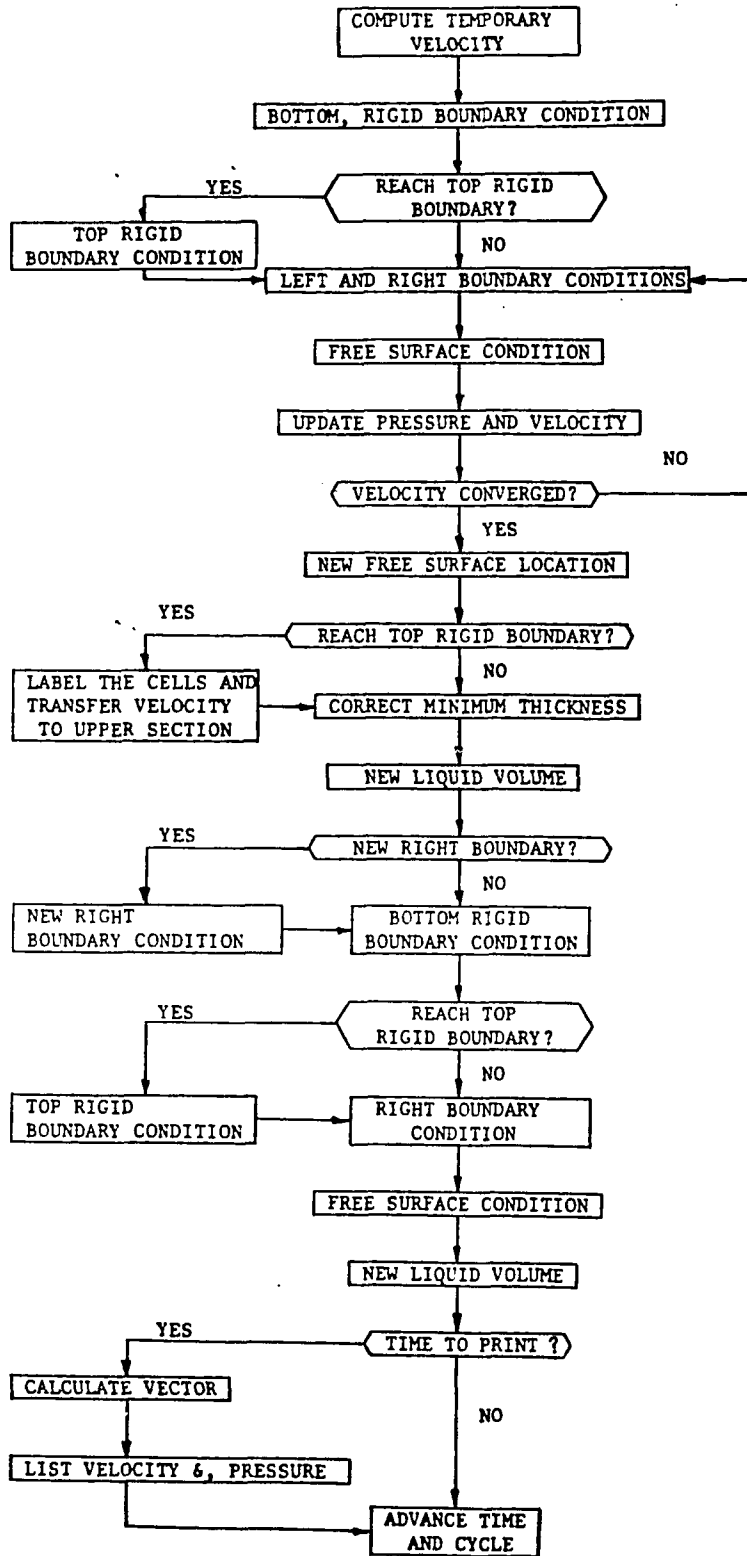


Fig. 22. Section computation - lower section I & II.

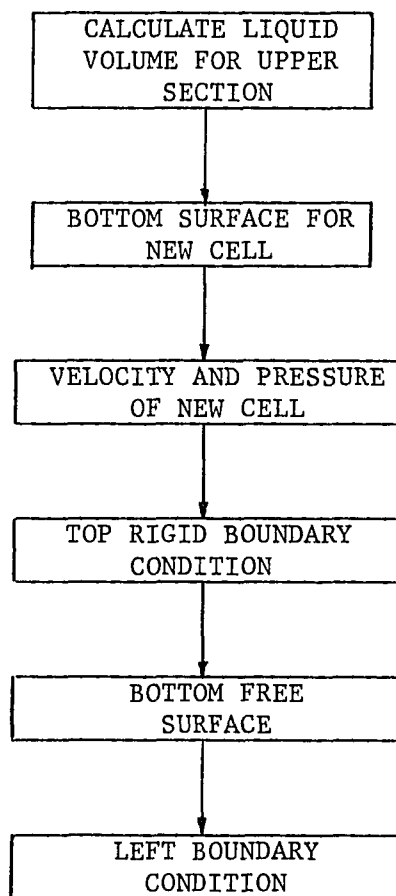


Fig. 23. Input to upper section.

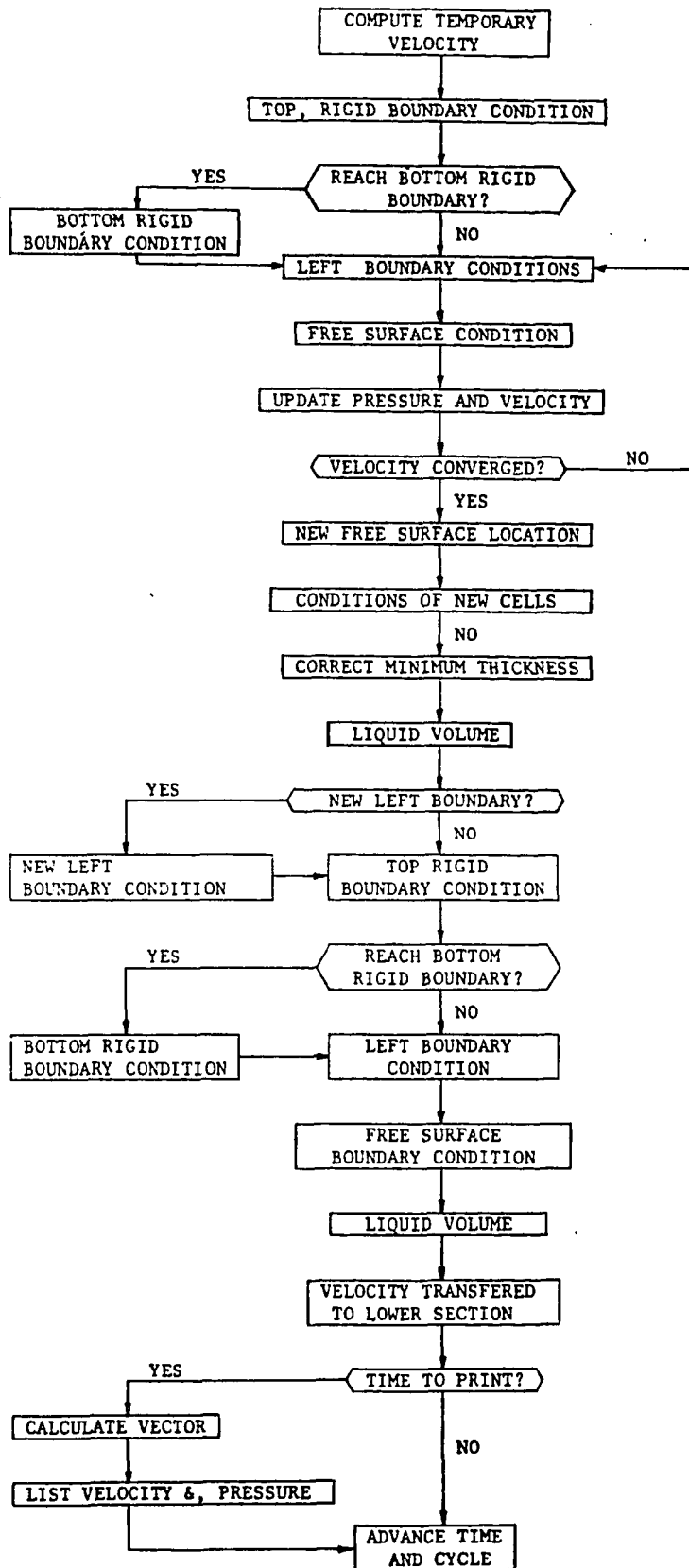


Fig. 24. Section computation - upper section.

These two section computations constitutes the calculation loop until the liquid reaches the left boundary and falls along the wall to the bottom rigid boundary. The computation of "lower section II" is added to the loop when the fluid touches the left bottom boundary. The front wave of lower section II will move faster to catch up the rear part of lower section I. After the liquid becomes a ring by connecting lower section I and II, the latter is consolidated into the former. The calculation loop consists of two sections -- lower I and upper -- again. But now the lower section I covers the entire area from ML to MR. The computational process will stop at specified time.

The section computations for lower and upper sections are basically the same. At the beginning the velocities are predicted by Eqs. 25a, b. Then, the boundary conditions for the rigid boundaries and free surfaces are set. If the discrepancy of continuity equation is less than tolerance, i.e., D (Eq. 28) $< \epsilon$, iteration of that time cycle is concluded. Otherwise, a new set of conditions are obtained from Eqs. 29 and 30 a - d. A test of convergency is repeated. When the velocities converge, a new free surface is located. Since the computation method requires minimum thickness of two cells in any column, it is necessary to eliminate the columns with single cell in the tailpart of the fluid. On the other hand, a second cell is made to of each single-cell column in the remaining part. If the total liquid volume is less than the original, the fluid enters into a new column to keep a constant volume in the cylinder. The initial and boundary conditions of the new column are to be calculated before a new time cycle begins.

IV.I Example

An example is give with listing in Appendix VII. The cylinder

size is chosen to be comparable to a laboratory equipment. The cell size is a compromise of accuracy and computer time. The kinematic viscosity was assumed to be in the same range as the silicone oil used in laboratory experiments. The input data are listed in Table I.

The indexing of the mesh is shown in Fig. 25. The horizontal index of lower half is 1-66, the upper 66-131. The vertical index is 1-34. After exclusion of cells of high aspect ratio, the mesh of calculation is reduced to 4-63 and 69-128, which includes one fictitious column of cells at each side. The excluded area is 3.4% of the circular cylinder. The actual circumference of the modified cylinder is 4.2% shorter than that of the whole circular cylinder. The initial liquid height results an average film thickness 11% of the radius, comparing with the maximum value of 20% in the rotational molding. Appendix VI details the calculations of these numbers

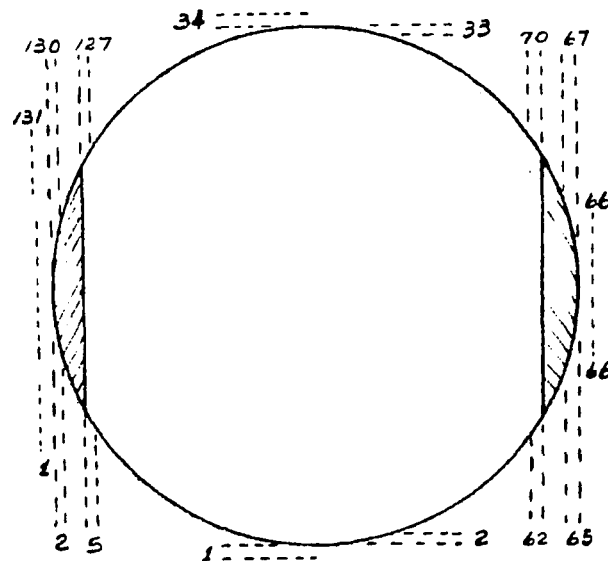


Fig. 25. Index in the example problem. Shaded area is excluded from calculation.

TABLE I
INPUT DATA OF EXAMPLE

System	ITEMS	VALUES
Cylinder radius	(AR)	6.4 cm
Mesh spacing of cell X-axis	(δx)	0.2 cm
Mesh spacing of cell Y-axis	(δy)	0.4 cm
Rotation velocity	(AS)	220 cm/sec
Initial fluid height	(FLHT)	3.5 cm
Kinematic viscosity	(NU)	2 cm ² /sec
Gravitational acceleration X-axis	(g_x)	0
Gravitational acceleration Y-axis	(g_y)	-980 cm/sec ²
Parameters		
Weight of partial donor cell- convective flux	(α)	0.9
Weight of partial donor cell- free surface	(γ)	0.2
Over-relaxation factor	(OMG)	1.4
Convergence criterion	(ϵ)	0.1
Time increment	(δt)	0.0005 sec

NOTE: names of items in the listings

δx = DELX , δy = DELY

g_x = GX , g_y = GY

α = DPLHA (lower), ULPHA (upper)

OMG = OMG (lower) , UMG (upper)

γ = GAMMA

ϵ = EPSI (lower), UPSI (upper)

δt = DELT

The convergence criterion can be checked by assuming a value of discrepancy to be .01 for both u and v velocities in Eq. 28.

$$D = .01/.2 + .01/.4 = 0.75$$

Considering a rotational velocity of 220 cm/sec, the tolerance value of 0.1 represents error in the order of $10^{-3}\%$.

The time increment is to meet with both inequalities 52a, b.

$$\delta t < \min (|.2/220|, |.4/220|) = 0.0009$$

$$\delta t < (.2)^2 (.4)^2 / 2(2) (.2^2 + .4^2) = 0.008$$

A value of 0.0005 is chosen by trial-and-error.

The controlling amount of donor cell fluxing has to satisfy the inequality 49.

$$1 \geq \alpha = k \max (|220 \times 0.0005/.2|, |220 \times 0.0005/.4|)$$

By trial-and-error method the k value is selected as 1.7 and α as 0.9.

Similarly, the optimum values of γ and OMC are found to be 0.2 and 1.4, respectively. Each parameter may have different values for lower and upper halves.

The result of calculation indicates the time of traveling around a cylinder is 327 time cycles. With the assumption of no-slip rotation throughout the calculation, the number of time cycles is 294 (Appendix VI). Since the actual number of time cycle is greater than the hypothetical number for no-slip rotation, the computer model is considered valid.

V DISCUSSION AND CONCLUSIONS

Several rotating velocities have been tested in the computer program. The comparison in Table II indicates correspondence between numerical analysis and experimental results which are summarized in Eqs. 1 and 2. By numerical method, the change from cascading to rimming flow occurs at a rotational velocity of 190-200 cm/sec. Eq. 2 gives the onset velocity of rimming flow at 203 cm/sec. Eq. 1 shows that the flow in the range of 140-190 cm/sec is cascading type. Fig. 26 shows the calculated points are

TABLE II

	COMPARISON OF NUMERICAL METHOD WITH EXPERIMENTAL RESULTS						
	AS (cm/sec)						
	140	180	190	200	210	220	
t (sec)	.0008	.0006	.00055	.00055	.00053	.0005	
cycle (1)	337	314	310	255	271	259	
cycle (2)	390	335		258	277	260	
cycle (3)	400	400	315	276	318	267	
ILU (3)	122	122	121	127	127	127	
flow	cascading \longrightarrow			(4)	rimming \longrightarrow		
w(rad/sec)	21.88	28.13	29.69	31.25	32.81	34.38	
Re _m	5.36	6.89	7.27	7.66	8.04	8.42	
0.8 Re _m ^{0.867}	3.43	4.27	4.47				
1.11 Re _m ^{0.867}				6.2	6.48	7.04	
Fr _m	3.13	5.17	5.76	6.38	7.03	7.72	

- Notes: (1) This is the cycle number when liquid reaches I = 121 (upper)
 (2) This is the cycle number when liquid reaches I = 122 (upper)
 (3) This is the cycle number when liquid reaches I = ILU (upper)
 (4) The trend of results in cycles 310 to 315 shows it will not become the rimming flow.

not in the experimental range, but fit the correlation of experimental results very well. The assumption of Newtonian fluid in the low shear

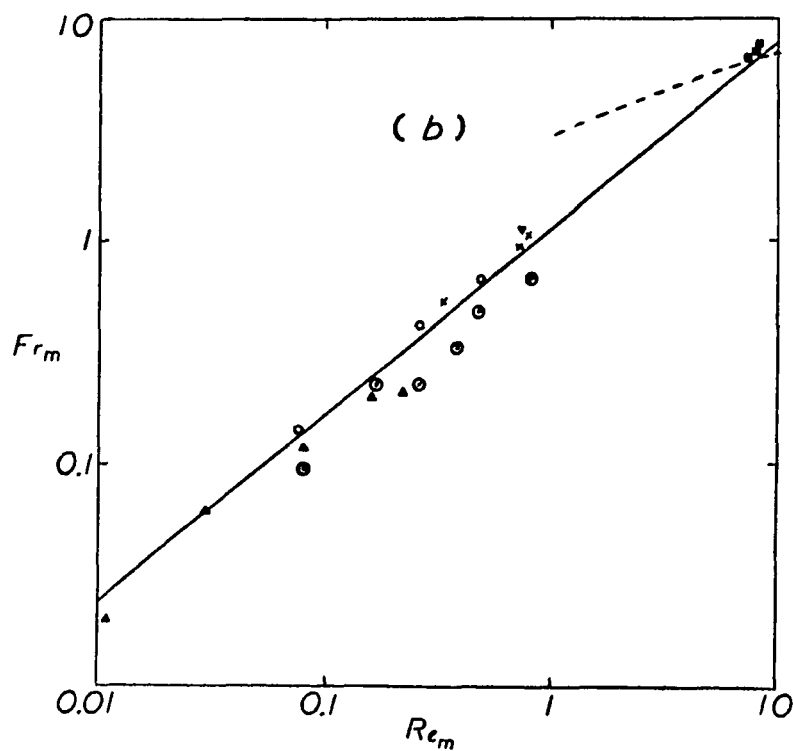
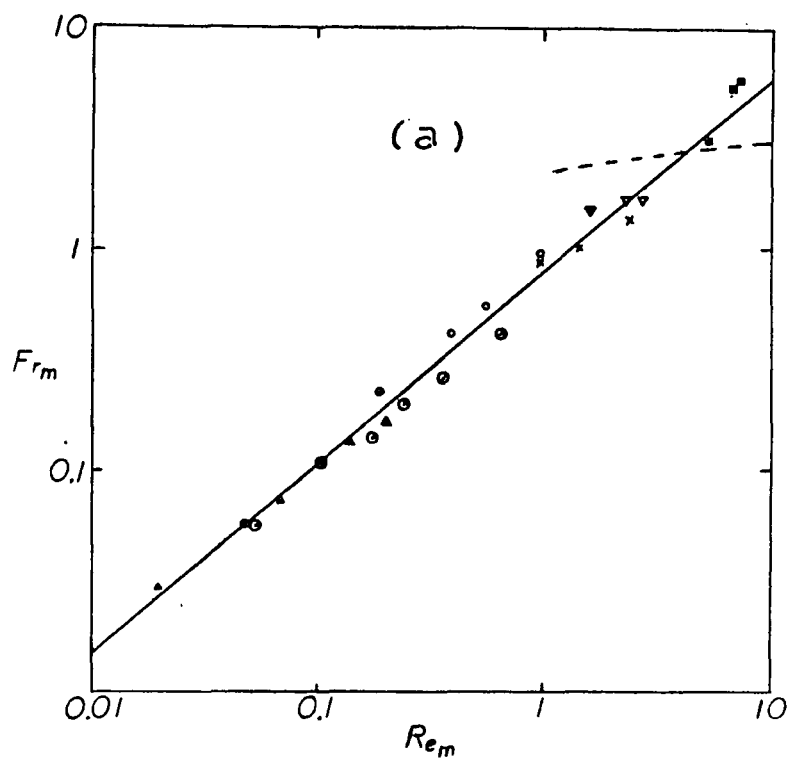
Fig. 26 Comparison of numerical method with experimental results (8).
(a) cascading regime,
(b) rimming regime.

Symbols Viscosity (ν)

▲	62.24
⊙	42.46
○	29.46
×	15.56
▽	12.45
■	2.0

NOTES:

1. Numerical method in this work is for $2 \text{ cm}^2/\text{sec}$.
2. The experimental correlations are shown by straight solid lines.
3. The prediction by Deiber and Cerro (24) is shown by dotted line.



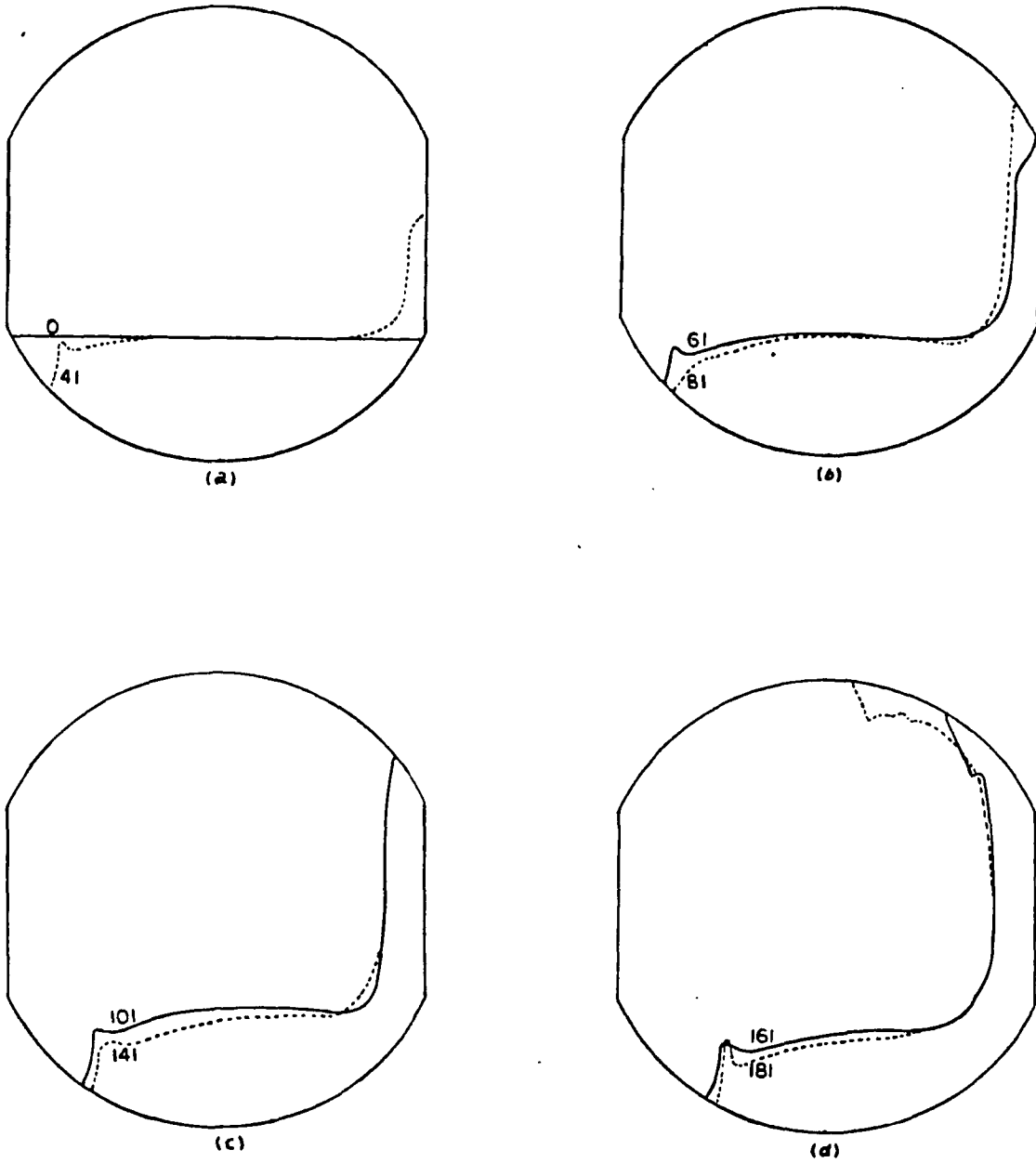


Fig. 27 Transient flow phenomena of the example. Free Surface changed as cylinder rotating from time cycle # 0 to 560.

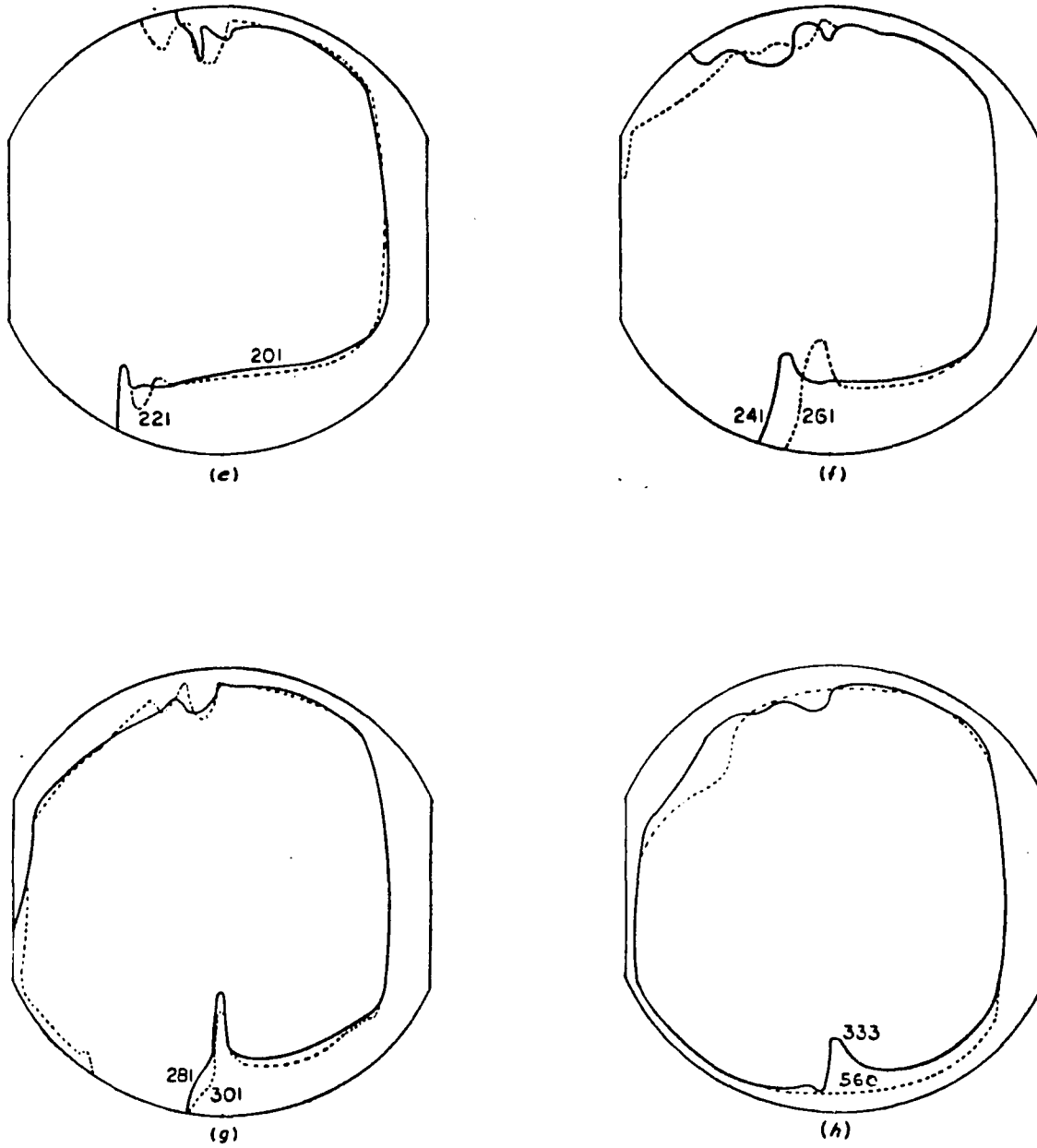


Fig. 27 Transient flow phenomena of the example. Free Surface changed as cylinder rotating from time cycle # 0 to 560.
(cont'd)

rates and low Reynolds numbers (<100) is also valid.

The transient flow phenomena of the example problem are shown in Fig. 27 (a) - (h). In Fig. 27 (a), a quiescent pool is shown at time cycle = 0, or prior to any motion. When the counter-clockwise rotation of the cylinder starts, a portion of the fluid is carried along the wall. In Fig. 27 (b), the fluid reaches the top rigid boundary at time cycle #61. One time cycle is equivalent to one t , or 0.0005 sec. As the time cycle advances from #61 to #261, the upper boundary is covered gradually by the fluid, while the liquid layer in the lower half becomes thinner and thinner. Coming down the vertical wall, the fluid moves fast to complete a ring at time cycle #333 as shown in Fig. 27 (h).

The accurate experimental data for comparison with the results of transient flow is still in need. The fluid velocities in the vicinity of rigid boundary are close to the rotating velocity. Backward velocities occur near the free surface. Both are expected. A comparison can also be made with steady rimming flow in previous works by Scriven et al. (21, 25). From Fig. 4, the thickest film should be at $\theta = 1.18$ for $Re_m = 7.66$. From Fig. 7a, which is for the value of $Re_m (R/b)^2$ equal to 1, the thinnest spot is $\theta = \pi$ and the thickest $\theta = 0$. In the physical dimension of the example problem, this is equivalent to $Re_m = 0.012$. Based on the series of change in Fig. 27 (a) - (h), the predicted thickest spot is in the range $\theta = 0$ to $\pi/2$ and the thinnest $\theta = \pi$ to $3\pi/2$. This concurs with the conclusion on page 12.

The results of calculation reveal variations of liquid volume with amount up to $\pm 10\%$ as shown in Fig. 28. The average of variation is + 0.56%, and the standard deviation is 3.77%. This variation reflects the

inaccuracy in approximating the position and pressure at the free surface. The tracking of the interface location is a surface height function method method, as described on page 33. This is a simple definition requiring a minimum storage of information. Other methods, such as line segments and marker particles, provide better approximation, but require more storage. The volume of fluid method (32), under testing for this rotational cylinder problem at present, has demonstrated very powerful in approximation and required only small amount of storage. The calculation of cell center pressure at the free surface on page 32 is merely an interpolation along y - direction. This is very simple, but may result incorrect pressure and, consequently, obtain spurious velocity. The appearance of humps at the free surface is a clear indication of this error. An improved method (30) is to use the nearest neighboring full cell for interpolation when there is more than one neighboring full cell. This will make the free surface smooth and may decrease the variation of volume.

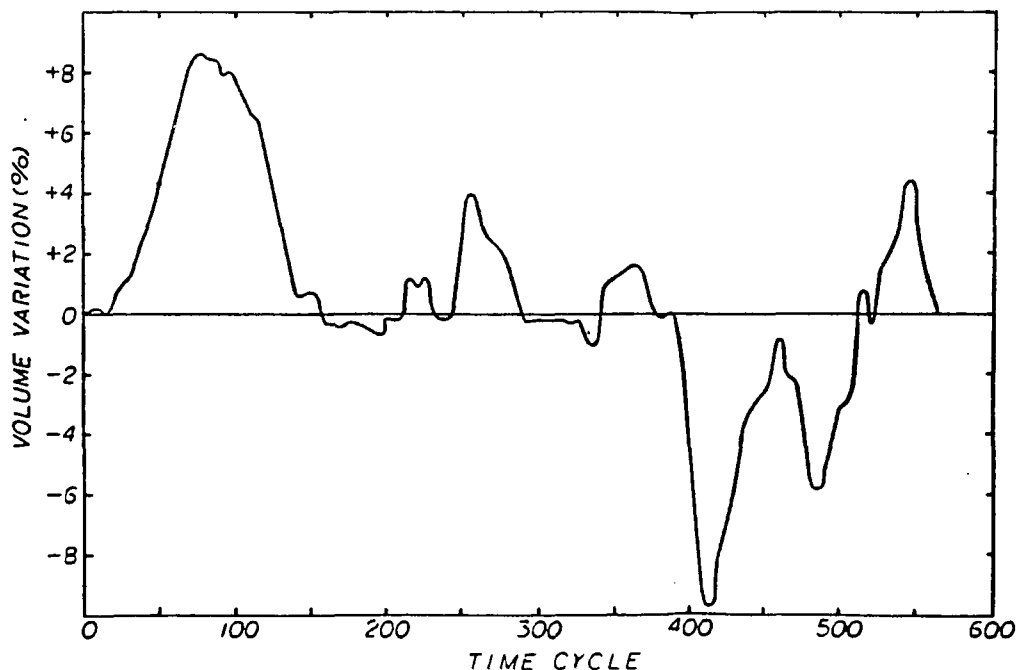


Fig. 28 Volume variation at time cycles.

In Fig. 27 (e) - (g), the tail part of the lower half results a high narrow wave. The bump stays in the tail and increases its height until a complete ring is formed. Then, it diminished rapidly. This is a phenomenon of singularity. It exists in all of the rotating velocities tested in this work. It builds up gradually as the fluid flow moves upwards and returns to a smooth surface in a later stage as shown in Fig. 29 (a). The number of time cycles to yield a smooth surface depends on the rotating velocity. The higher the velocity, the more the time cycle. This is illustrated in Fig. 29 (b).

Singularity is a result of discontinuity. In a field computation using finite difference technique, most of the discontinuities arise from the edges and noses of curvature. To overcome this difficulty, the mesh size may be reduced to a fine enough network (35). The singularity at

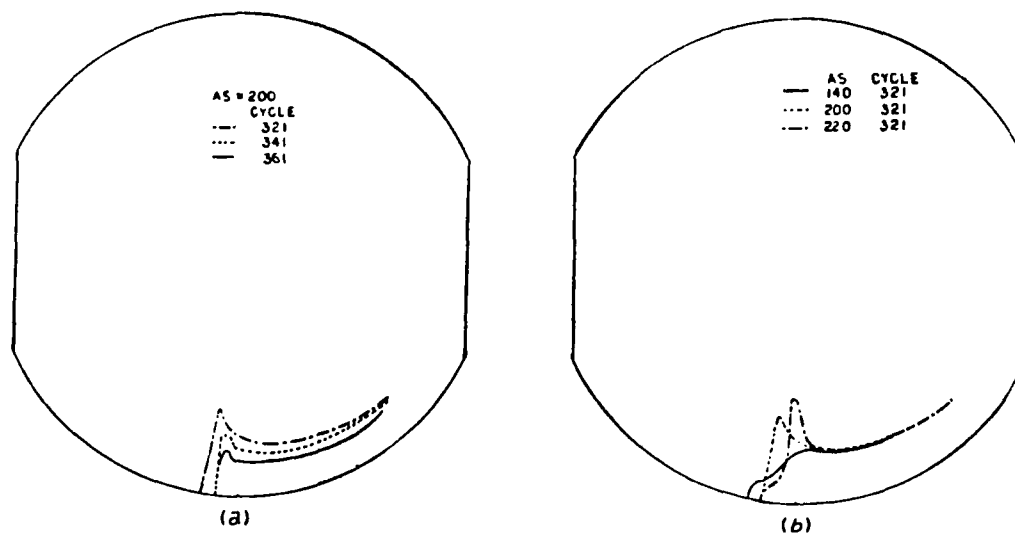


Fig. 29 Phenomena of singularity.

the contact line between solid and free surface has been investigated by Huh and Scriven (36) and others (37, 38). Huh and Scriven proposed a hydrodynamic model for steady state flow over a rigid surface. In terms of stream function, the solution for a flat, no-slip solid yielded a realistic velocity field. However, the shear stress and pressure field increased to infinity as the contact line was approached. The singularity indicates the breakdown of the hydro-dynamic model and suggests discontinuity physically existing around the contact line. The most important reason for this discontinuity is the no-slip boundary condition (53), that demands the liquid immediately next to the rigid wall have no relative velocity to the wall. According to no-slip condition the advancing contact angle should be a function of wall velocity as shown in Fig. 30. But experiments indicate the contact angle increases irregularly as the velocity increases. Both rolling and sliding have been mentioned in the observations. To correct the boundary condition, the no-slip condition may be replaced by a dynamic condition which includes a slip factor or momentum transfer coefficient varying in the range of no-slip to partial slip. Unfortunately, no such a condition has been proved successfully without certain limitations (39,

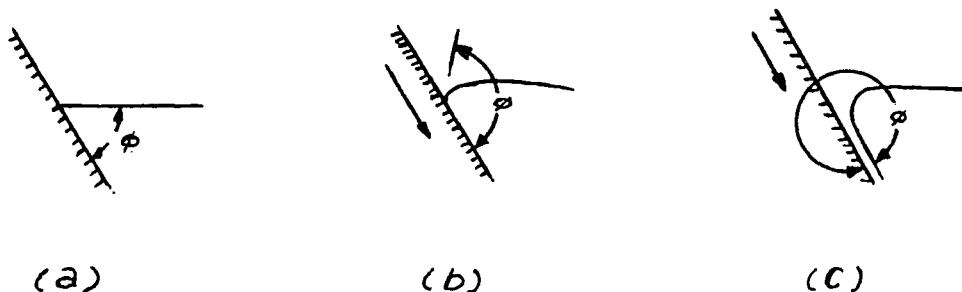


Fig. 30 Contact angle. (a) equilibrium, (b) low speed,
(c) high speed.

40, 53). Other possible reasons for the singularity are cavitation, compressibility effects, breakdown of Newtonian fluid relation, etc. Due to the fact that surface tension has a significant effect on the characteristics of contact angle (54), it is impossible to obtain a complete picture of the development of singularity in any calculation excluding surface tension as in this work. Since the bump either diminishes in a later time cycle or exerts no significant effect on the result of rimming velocity, no attempt has been made to correct this difficulty.

During the development of model, various trials have been made to deduce some guidelines. The most important one, of course, is stability problem, which is inherent in all the numerical approximations. The set of parameters in the above example may not result in a convergence for other fluids of higher or lower viscosity. In general, a set of parameters is good for a viscosity at several velocities. When the velocity changes, the main concern is time increment. Some test runs in the example indicate that a value of Δt with 10% off its optimum may result in a divergence. Fortunately, the optimum ranges of parameters are not so narrow. Therefore, it is the parameters to be chosen first. If a set of parameters allows liquid to enter into the upper half, those values are adequate for the whole run.

The next important is the initial conditions in the upper half. Since the algorithm is using both the upstream and underneath cell velocities for computing discrepancy in Eq. 28, an intelligent guess of initial conditions is necessary to ensure a convergence in upper half. Eqs. 37a, b are found able to yield such a result. But other forms can be tried, too, for instance,

$$u_{i,j} = (u_{i-1,j} + u_{i,j+1})/2 \quad (53a)$$

$$v_{i,j} = (v_{i-1,j} + v_{i,j+1})/2 \quad (53b)$$

An acceptable initial condition usually does not relate only to a single velocity of surroundings, such as Eqs. 39a, b. The initial conditions for a whole new column are less critical.

The third is correcting the minimum thickness. As mentioned in previous sections, this algorithm applies only to columns with at least two cells. If the free surface is inside the bottom cell of lower half or the top cell of the upper half, it becomes necessary to make up a second cell, i.e., to extend the liquid into the second cell. The question is how far it should extend to. In the lower half, it is to the top edge of the bottom cell.

$$H_1 = (jB - 1) \delta y \quad (54)$$

In the upper half, it is to the bottom edge of the top cell.

$$HB_1 = (jT - 2) \delta y \quad (55)$$

Comparison of this method with others is not available at present. (See Appendix VII for brief discussion on numerical methods for fluid dynamics.) The advantages of MAC method are basically its primitive variables and the capability to handle free surface conditions. The method has been proved extremely powerful in water wave problems. The example was run on a Amdahl 470 computer and required cpu 3 min 25.48 sec for 560 time cycles and 2 min 48.26 sec for 460 time cycles.

The accuracy of this method can be checked based on truncation and round-off errors. The truncation error in the centered differencing is second order in the spatial cell size and in the upstream differencing is first order. Since this method is using a hybrid of centered and partial donor cell differencings, it can produce greater accuracy than the upstream differencing. The round-off error is minimized by double precision

in computer calculation. Although cell refinement can alleviate these error problems, it is usually impractical for engineering purposes. When the difference equations are conditional stable, the fineness of time step must satisfy the stability criteria, such as inequalities 52a and b, that involve the cell size. Therefore, a decrease in cell size may cause δt decrease not by ratio of the later size to the previous size, but by square of the ratio. The computer time will increase rapidly to accommodate to the smaller cell size and much shorter time step. Other convective modeling techniques which have error of high order while avoiding the stability problems on a comparable large cell size is recommended for better accuracy. For instance, the Leonard algorithm (46) with error of third order could be tried. With this type of high efficiency technique, a cylindrical coordinate system with larger cell size can be considered so that the error in approximating the location of rigid boundary could be eliminated.

The question of multiple solution in a free-surface flow has been investigated primarily for flow between disks and swirling flow. Some steady state problems have been solved analytically or numerically to obtain non-unique solutions. At least three situations may lead to the multiple solutions:

- (1) Wrong theory or assumption (41);
- (2) Physically existing conditions, such as in the transition of couette flow and steady flow between rotating disks (42, 43); and
- (3) The location and shape of free surface that is unknown a priori.

The last reason indicates the multiplicity of solution is inherent in the Navier-Stokes equation (44). Frequently, an approximation of the location and shape of free surface is built in the numerical method to provide assumed data of free surface. With those data, the iteration procedure

will try to satisfy the free surface conditions. Due to the highly non-linear character of the Navier-Stokes equation, it is possible to have more than one set of location and shape satisfying the free surface conditions. Though it may be very difficult to prove uniqueness mathematically, only one of the solutions is expected physically possible, as discussed by Taylor (45). In MAC and MAC-derived techniques, it has been pointed out by B. D. Nichols, co-author of SOLA algorithm, in a private communication that no multiple solutions have been found.

In conclusion, the model is able to predict both the transient flow and steady flow phenomena. The numerical stability problem can be solved by trial-and-error method with help of stability theorem. However, it should be noted that this is only a simplified model. Modifications may be necessary to fit specific problems.

VI RECOMMENDATION

To have a better simulation of liquid rotational molding, further studies are required to eliminate some of the assumptions and restrictions.

Recently a SOLA-VOF program (32) has been developed to include surface tension forces. It also has a flexible mesh generator to provide cells with various different sizes. This will alleviate the aspect ratio problem. The whole circumference can be approximated by the diagonals of the cells.

Constantly changing viscosity is a phenomenon associated with curing. As viscosity increases, the parameters and time increment should be changed. An automatic time-step control in SOLA-VOF to adjust the time increment is not enough. A parameters adjustment has to be added. The relationship of time-temperature-viscosity (12) had better be a separate program. Even though the MAC method is capable to handle heat transfer differential equations, it is believed to be over-entangled., to couple heat transfer, reaction heat and fluid mechanics The main reason is, again, numerical stability.

APPENDIX I

FINITE DIFFERENCE REPRESENTATION OF NAVIER-STOKES EQUATIONS

The approximation of Eqs. 14 a, b and 15 by finite difference and partial donor cell methods is illustrated in this appendix.

The gravity constants g_x and g_y need no approximation. The velocity terms $\partial u/\partial x$ and $\partial v/\partial y$ are taking the form of backward differencing, i.e., Eqs. 19 c, d. The transient and pressure terms $\partial u/\partial t$, $\partial v/\partial t$, $\partial p/\partial x$ and $\partial p/\partial y$ are expressed by forward differencing similar to Eqs. 17 c, d, for example,

$$\partial u_{i,j}/\partial t = (u_{i,j} - u_{i,j})/\Delta t$$

$$\partial p_{i,j}/\partial x = (p_{i+1,j} - p_{i,j})/\Delta x$$

The viscous fluxes $\nu(\partial^2 u/\partial x^2 + \partial^2 u/\partial y^2)$ and $\nu(\partial^2 v/\partial x^2 + \partial^2 v/\partial y^2)$

including second derivatives are represented by the central differencing, i.e., Eqs. 20 c, d. The partial donor cell method is applied only to convective fluxes.

A. UUX

The first derivative is given by central differencing.

$$\begin{aligned} \partial uu/\partial x = UUX &= u(u_{i+\frac{1}{2},j} - u_{i-\frac{1}{2},j})/\Delta x \\ &= (uu_{i+\frac{1}{2},j} - uu_{i-\frac{1}{2},j})/\Delta x \end{aligned} \quad (I-1)$$

For the first term $uu_{i+\frac{1}{2},j}$, the partial donor cell defines

$$u = [(1+\bar{\alpha})u_{i,j} + (1-\bar{\alpha})u_{i+1,j}]/2 \quad (I-2)$$

where $\bar{\alpha}$ = weight of donor cell with value $1 \geq \bar{\alpha} \geq -1$

IF $\bar{\alpha} = \alpha(\text{sign } u_{i+\frac{1}{2},j})$ and $1 \geq \alpha \geq 0$, then, it becomes

$$\begin{aligned} uu_{i+\frac{1}{2},j} &= [u_{i+\frac{1}{2},j} u_{i,j} + \alpha(\text{sign } u_{i+\frac{1}{2},j}) u_{i+\frac{1}{2},j} u_{i,j} \\ &\quad + u_{i+\frac{1}{2},j} u_{i+1,j} - \alpha(\text{sign } u_{i+\frac{1}{2},j}) u_{i+\frac{1}{2},j} u_{i+1,j}]/2 \end{aligned} \quad (I-3)$$

Since

$$\text{and } (\text{sign } u_{i+\frac{1}{2},j}) u_{i+\frac{1}{2},j} = |u_{i+\frac{1}{2},j}|$$

$$u_{i+\frac{1}{2},j} = (u_{i,j} + u_{i+1,j})/2$$

Eq. I-3 is finalized as

$$uu_{i+\frac{1}{2},j} = [(u_{i,j} + u_{i+1,j})^2 + \alpha |u_{i,j} + u_{i+1,j}| (u_{i,j} - u_{i+1,j})] / 4 \quad (\text{I-4})$$

For the second term $uu_{i-\frac{1}{2},j}$, the partial donor cell representation of Eq. I-5 is developed through steps similar to Eqs. I-2, I-3 and I-4.

By setting

$$u = [(1 + \bar{\alpha})u_{i-1,j} + (1 - \bar{\alpha})u_{i,j}] / 2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } u_{i-\frac{1}{2},j}) \quad (1 \geq \alpha \geq 0)$$

then

$$uu_{i-\frac{1}{2},j} = [u_{i-\frac{1}{2},j} u_{i-1,j} + \alpha (\text{sign } u_{i-\frac{1}{2},j}) u_{i-\frac{1}{2},j} u_{i-1,j} + u_{i-\frac{1}{2},j} u_{i,j} - \alpha (\text{sign } u_{i-\frac{1}{2},j}) u_{i-\frac{1}{2},j} u_{i,j}] / 2$$

Since

$$(\text{sign } u_{i-\frac{1}{2},j}) u_{i-\frac{1}{2},j} = |u_{i-\frac{1}{2},j}|$$

and

$$u_{i-\frac{1}{2},j} = (u_{i-1,j} + u_{i,j}) / 2$$

therefore

$$uu_{i-\frac{1}{2},j} = [(u_{i-1,j} + u_{i,j})^2 + \alpha |u_{i-1,j} + u_{i,j}| (u_{i-1,j} - u_{i,j})] / 4 \quad (\text{I-5})$$

Combining Eqs. I-1, I-4, and I-5 results

$$UUX = [(u_{i,j} + u_{i+1,j})^2 + \alpha |u_{i,j} + u_{i+1,j}| (u_{i,j} - u_{i+1,j}) - (u_{i-1,j} + u_{i,j})^2 - \alpha |u_{i-1,j} + u_{i,j}| (u_{i-1,j} - u_{i,j})] / 4 \delta x \quad (27a)$$

B. UVY

Following central differencing scheme, the first derivative is

$$\begin{aligned} \partial uv / \partial y &= UVY = u(v_{i+\frac{1}{2},j} - v_{i+\frac{1}{2},j-1}) / \delta y \\ &= (uv_{i+\frac{1}{2},j} - uv_{i+\frac{1}{2},j-1}) / \delta y \end{aligned} \quad (I-6)$$

It should be noted that the X index of v terms is not i, but rather $i + \frac{1}{2}$. The reason can be visualized in Fig. 31. The terms of the first derivative have to be on the same line, either horizontally or vertically. In Fig. 31(a), the terms for $\partial uu / \partial x$ are on a horizontal line. To represent $\partial uv / \partial y$ every term is on a vertically line as shown in Fig. 31(b).

Applying partial donor cell method to the first term $uv_{i+\frac{1}{2},j}$ in Eq. I-6,

$$u = [(1+\bar{\alpha}) u_{i,j} + (1-\bar{\alpha}) u_{i,j+1}] / 2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and setting

$$\bar{\alpha} = \alpha (\text{sign } v_{i+\frac{1}{2},j}) \quad (1 \geq \alpha \geq 0)$$

it becomes

$$\begin{aligned} uv_{i+\frac{1}{2},j} &= [u_{i,j} v_{i+\frac{1}{2},j} + \alpha (\text{sign } v_{i+\frac{1}{2},j}) v_{i+\frac{1}{2},j} u_{i,j} \\ &+ u_{i,j+1} v_{i+\frac{1}{2},j} - \alpha (\text{sign } v_{i+\frac{1}{2},j}) v_{i+\frac{1}{2},j} u_{i,j+1}] / 2 \end{aligned} \quad (I-7)$$

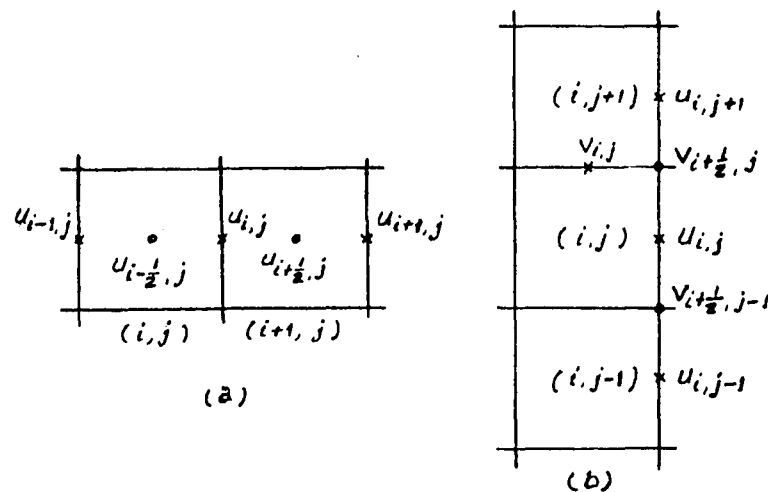


Fig. 31. Finite difference terms for (a) $\partial uu / \partial x$ and (b) $\partial uv / \partial y$.

By definition of

$$(\text{sign } v_{i+\frac{1}{2},j}) v_{i+\frac{1}{2},j} = |v_{i+\frac{1}{2},j}|$$

and a centered convective flux

$$v_{i+\frac{1}{2},j} = (v_{i,j} + v_{i+1,j})/2,$$

Eq. I-7 results

$$\begin{aligned} uv_{i+\frac{1}{2},j} = & \left[(u_{i,j} + u_{i,j+1}) (v_{i,j} + v_{i+1,j}) \right. \\ & \left. + \alpha |v_{i,j} + v_{i+1,j}| (u_{i,j} - u_{i,j+1}) \right] / 4 \end{aligned} \quad (\text{I-8})$$

Similarly, the second term $uv_{i+\frac{1}{2},j-1}$ of Eq. I-6 obtains the

expression Eq. I-9. By setting

$$u = \left[(1+\bar{\alpha})u_{i,j-1} + (1-\bar{\alpha})u_{i,j} \right] / 2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } v_{i+\frac{1}{2},j-1}) \quad (1 \geq \alpha \geq 0)$$

and by definition

$$(\text{sign } v_{i+\frac{1}{2},j-1}) v_{i+\frac{1}{2},j-1} = |v_{i+\frac{1}{2},j-1}|$$

then by setting

$$v_{i+\frac{1}{2},j-1} = (v_{i,j-1} + v_{i+1,j-1})/2$$

the result is

$$\begin{aligned} uv_{i+\frac{1}{2},j-1} = & \left[(u_{i,j-1} + u_{i,j}) (v_{i,j-1} + v_{i+1,j-1}) \right. \\ & \left. + \alpha |v_{i,j-1} + v_{i+1,j-1}| (u_{i,j-1} - u_{i,j}) \right] / 4 \end{aligned} \quad (\text{I-9})$$

Combining Eqs. I-6, I-8 and I-9 yields

$$\begin{aligned} UVY = & \left[(u_{i,j} + u_{i,j+1}) (v_{i,j} + v_{i+1,j}) + \alpha |v_{i,j} + v_{i+1,j}| x \right. \\ & (u_{i,j} - u_{i,j+1}) - (u_{i,j-1} + u_{i,j}) (v_{i,j-1} + v_{i+1,j-1}) \\ & \left. - \alpha |v_{i,j-1} + v_{i+1,j-1}| (u_{i,j-1} - u_{i,j}) \right] / 4 \delta y \end{aligned} \quad (27b)$$

C. UVX

The derivation for $\partial uv / \partial x$ is basically the same step by step of operation as for $\partial uv / \partial y$. From Fig. 32(a), the terms on a horizontal line are to be used in the central differencing.

$$\begin{aligned}\partial uv / \partial x &= UVX = v(u_{i,j+\frac{1}{2}} - u_{i-1,j+\frac{1}{2}}) / \delta x \\ &= (vu_{i,j+\frac{1}{2}} - vu_{i-1,j+\frac{1}{2}}) / \delta x\end{aligned}\quad (I-10)$$

For the first term $vu_{i,j+\frac{1}{2}}$, setting

$$v = [(1+\bar{\alpha})v_{i,j} + (1-\bar{\alpha})v_{i+1,j}] / 2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } u_{i,j+\frac{1}{2}}) \quad (1 \geq \alpha \geq 0)$$

then

$$\begin{aligned}vu_{i,j+\frac{1}{2}} &= [u_{i,j+\frac{1}{2}} (v_{i,j} + v_{i+1,j}) \\ &\quad + \alpha (\text{sign } u_{i,j+\frac{1}{2}}) u_{i,j+\frac{1}{2}} (v_{i,j} - v_{i+1,j})] / 2\end{aligned}\quad (I-11)$$

By definition

$$(\text{sign } u_{i,j+\frac{1}{2}}) u_{i,j+\frac{1}{2}} = |u_{i,j+\frac{1}{2}}|$$

then, setting

$$u_{i,j+\frac{1}{2}} = (u_{i,j} + u_{i,j+1}) / 2$$

Eq. I-11 becomes

$$\begin{aligned}vu_{i,j+\frac{1}{2}} &= [(u_{i,j} + u_{i,j+1}) (v_{i,j} + v_{i+1,j}) \\ &\quad + \alpha |u_{i,j} + u_{i,j+1}| (v_{i,j} - v_{i+1,j})] / 4\end{aligned}\quad (I-12)$$

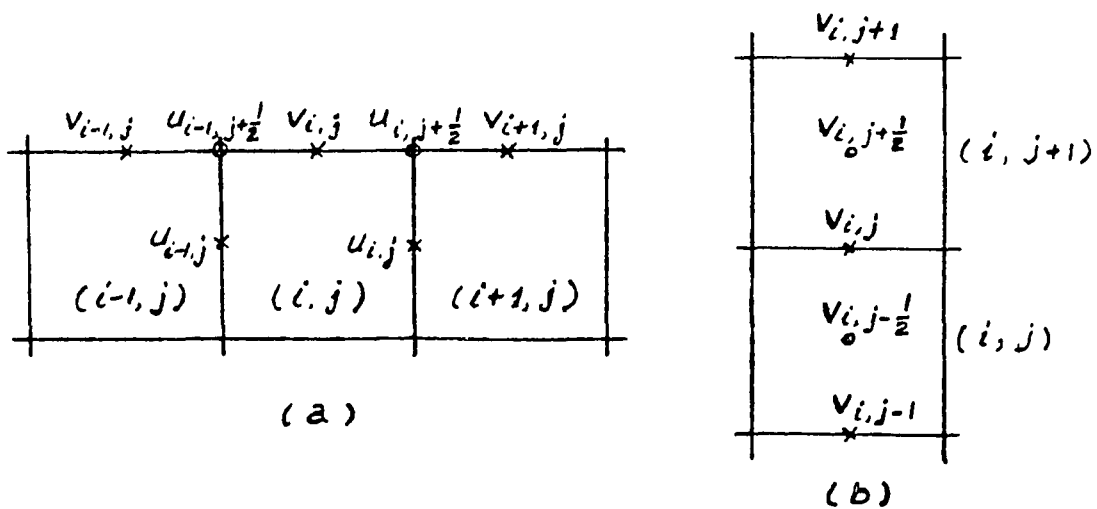


Fig. 32. Finite difference terms for (a) $\partial uv / \partial x$ and (b) $\partial vv / \partial y$.

For the second term $vu_{i-1,j+\frac{1}{2}}$ in Eq. I-10, the similar procedures are followed to yield Eq. I-13.

By setting

$$v = [(1+\bar{\alpha})v_{i-1,j} + (1-\bar{\alpha})v_{i,j}]/2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } u_{i+1,j+\frac{1}{2}}) \quad (1 \geq \alpha \geq 0)$$

then

$$(\text{sign } u_{i-1,j+\frac{1}{2}}) u_{i-1,j+\frac{1}{2}} = |u_{i-1,j+\frac{1}{2}}|$$

setting

$$u_{i-1,j+\frac{1}{2}} = (u_{i-1,j} + u_{i,j+1})/2$$

then

$$\begin{aligned} vu_{i-1,j+\frac{1}{2}} = & [(u_{i-1,j} + u_{i-1,j+1}) (v_{i-1,j} + v_{i,j}) \\ & + \alpha |u_{i-1,j} + u_{i-1,j+1}| (v_{i-1,j} - v_{i,j})]/4 \end{aligned} \quad (\text{I-13})$$

Combining Eqs. I-10, I-12 and I-13 obtains

$$\begin{aligned} UVX = & [(u_{i,j} + u_{i+1,j}) (v_{i,j} + v_{i+1,j}) \\ & + \alpha |u_{i,j} + u_{i,j+1}| (v_{i,j} - v_{i-1,j}) \\ & - (u_{i-1,j} + u_{i-1,j+1}) (v_{i-1,j} + v_{i,j}) \\ & - \alpha |u_{i-1,j} + u_{i-1,j+1}| (v_{i-1,j} - v_{i,j})]/4\delta x \end{aligned} \quad (27d)$$

D. VVY

This derivation is closely similar to that in section A for $\partial uu/\partial x$. All the terms involved are shown in Fig. 32(b).

$$\begin{aligned} \partial vv/\partial y = VVY = & v(v_{i,j+\frac{1}{2}} - v_{i,j-\frac{1}{2}})/\delta y \\ = & (vv_{i,j+\frac{1}{2}} - vv_{i,j-\frac{1}{2}})/\delta y \end{aligned} \quad (\text{I-14})$$

For the first term $vv_{i,j+\frac{1}{2}}$, since

$$v = [(1+\bar{\alpha})v_{i,j} + (1-\bar{\alpha})v_{i,j+1}]/2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } v_{i,j+\frac{1}{2}}) \quad (1 \geq \alpha \geq 0)$$

and by definition

$$(\text{sign } v_{i,j+\frac{1}{2}})v_{i,j+\frac{1}{2}} = |v_{i,j+\frac{1}{2}}|$$

and by centered differencing

$$v_{i,j+\frac{1}{2}} = (v_{i,j} + v_{i,j+1})/2$$

the result is

$$vv_{i,j+\frac{1}{2}} = \left[(v_{i,j} + v_{i,j+1})^2 + \alpha |v_{i,j} + v_{i,j+1}| \times (v_{i,j} - v_{i,j+1}) \right] / 4 \quad (\text{I-15})$$

Similarly, the second term $vv_{i,j-\frac{1}{2}}$ is expanded and consolidated. Since

$$v = \left[(1+\bar{\alpha})v_{i,j-1} + (1-\bar{\alpha})v_{i,j} \right] / 2 \quad (1 \geq \bar{\alpha} \geq -1)$$

and

$$\bar{\alpha} = \alpha (\text{sign } v_{i,j-\frac{1}{2}}) \quad (1 \geq \alpha \geq 0)$$

and by definition

$$(\text{sign } v_{i,j-\frac{1}{2}})v_{i,j-\frac{1}{2}} = |v_{i,j-\frac{1}{2}}|$$

and by centered differencing

$$v_{i,j-\frac{1}{2}} = (v_{i,j-1} + v_{i,j})/2$$

the result is

$$vv_{i,j-\frac{1}{2}} = \left[(v_{i,j-1} + v_{i,j})^2 + \alpha |v_{i,j-1} + v_{i,j}| \times (v_{i,j-1} - v_{i,j}) \right] / 4 \quad (\text{I-16})$$

Combining Eqs. I-14, I-15 and I-16 results

$$\begin{aligned} VVY = & \left[(v_{i,j} + v_{i,j+1})^2 + \alpha |v_{i,j} + v_{i,j+1}| \times \right. \\ & (v_{i,j} - v_{i,j+1}) - (v_{i,j-1} + v_{i,j})^2 \\ & \left. - \alpha |v_{i,j-1} + v_{i,j}| (v_{i,j-1} - v_{i,j}) \right] / 4sy \quad (27e) \end{aligned}$$

APPENDIX II

SURFACE LOCATION AND SLOPE

In this appendix, several cases of surface determination are discussed. Also described is the slope of the surface. Since one quadrant is mirror image of any other, the calculation is performed for third quadrant only.

A. Surface Location

In Fig. 33(a), a curved boundary line is crossing a cell (i, j), which is located by BX and BY.

$$BX = (i - 1)\delta x \tag{II - 1}$$

From geometrical relationship, it yields

$$(AR - BY)^2 = AR^2 - (AR - BX)^2$$

Rearranging the equation for BY, it results

$$BY = AR - [2(AR)(BX) - BX^2]^{\frac{1}{2}} \tag{II - 2}$$

At least four different relationships between JBL and JBR, which are defined in the followings, should be discussed as shown in Fig. 33(b).

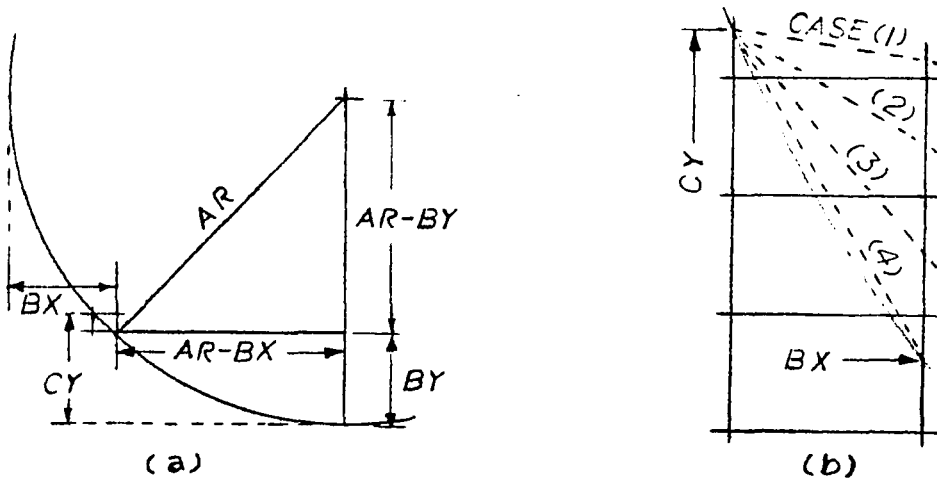


Fig. 33 Computation of surface location.

$$BL = CY/\delta y$$

$$JBL = \text{INT} (BL)$$

$$BR = BY/\delta y$$

$$JBR = \text{INT} (BR)$$

1. $JBL = JBR$ (Fig. 34(a))

If $(BR - JBR) + (BL - JBL) \geq 1.5$, $JBR + 1$ is the boundary cell.

Otherwise, $JBL + 2$ is the boundary.

2. $JBL - 1 = JBR$ (Fig. 34(b))

$$(BL - JBL) / [1 - (BR - JBR)] = (1 - PX) / PX = BK$$

$$\text{Let } RBX = 1 / (1 + BK) = PX$$

If $0.5 [1 - (BR - JBR)] / (1 + BK) \geq \frac{1}{4}$, $JBR + 1$ is the boundary.

Otherwise, $JBL + 1$ is the boundary

3. $JBL - 2 = JBR$

In either line A or line B of Fig. 34(c) the boundary is the cell of $JBL + 2$. The triangular area in the cell of $JBL + 1$ is always less than $\frac{1}{4}$ of a cell.

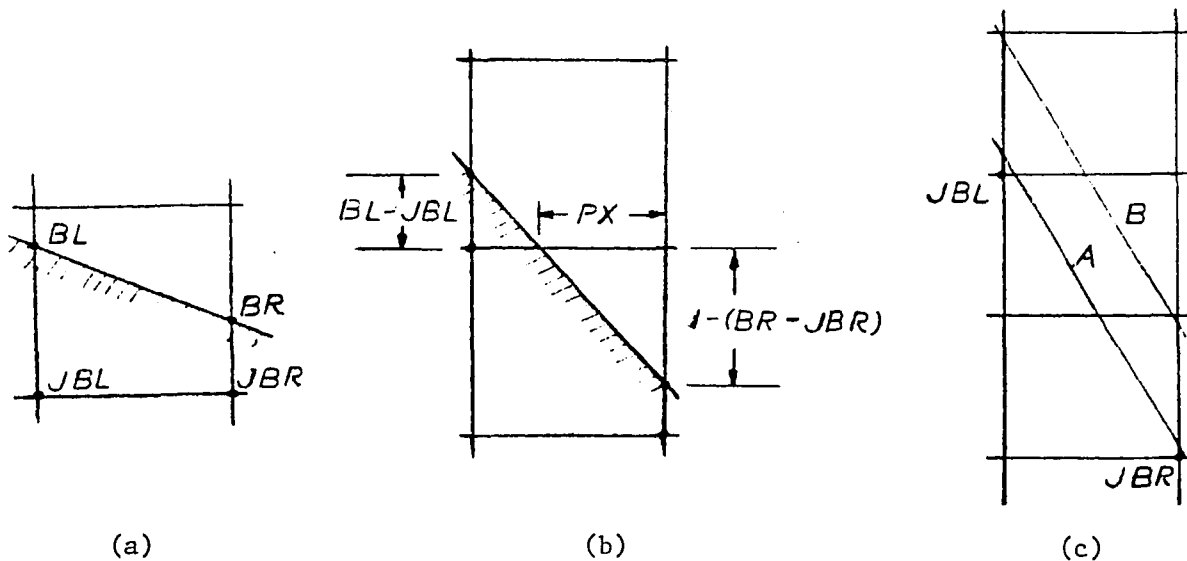


Fig. 34 Determination of the boundary.

(a) $JBL = JBR$, (b) $JBL - 1 = JBR$, (c) $JBL - 2 = JBR$.

4. JBL - 3 \geq JBR

For a steep slope boundary the boundary cell is at the midpoint of BY and CY.

B. Slope of Boundary

By connecting the center of drum, O, to mid-point of bottom surface of cell i and defining $X_i = BX - X/2$, the angle θ can be computed.

$$\theta = \tan^{-1} [(AR - X_i)/(AR - HB)]$$

The slope of boundary at cell i is

$$SLOB_i = \tan \theta$$

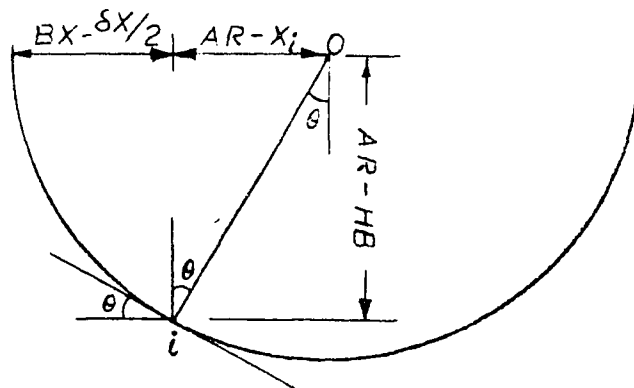


Fig. 35 Slope of boundary.

APPENDIX III

ADJUSTMENTS OF CELL PRESSURE AND VELOCITY

The derivations of Eqs. 29 and 30a - d in Sec. IV.C Velocity Iteration are described in this appendix.

The momentum equation 25a is re-written in two groups of variables; one group contains pressure item, another group does not.

$$\begin{aligned} u_{i,j}^o &= u_{i,j} + \delta t(g_x - UUX - UVY - VISU) - (p_{i+1,j} - p_{i,j}) \delta t / \delta x \\ &= UCON_{i,j} - (p_{i+1,j} - p_{i,j}) \delta t / \delta x \end{aligned} \quad (III-1a)$$

where $UCON_{i,j} = u_{i,j} + \delta t(g_x - UUX - UVY - VISU)$

Since $UCON_{i,j}$ consists of the terms in old time level, it is a constant in the new time level. Similarly, Eq. 25b is re-written.

$$v_{i,j}^o = VCON_{i,j} - (p_{i,j+1} - p_{i,j}) \delta t / \delta y \quad (III-1b)$$

Two more equations in the same form are needed.

$$u_{i-1,j}^o = UCON_{i-1,j} - (p_{i,j} - p_{i-1,j}) \delta t / \delta x \quad (III-1c)$$

$$v_{i,j-1}^o = VCON_{i,j-1} - (p_{i,j} - p_{i,j-1}) \delta t / \delta x \quad (III-1d)$$

From the above four equations III-1a thru 1d, the following derivatives are resulted:

$$\partial u_{i,j}^o / \partial p_{i,j} = \delta t / \delta x \quad (III-2a)$$

$$\partial u_{i-1,j}^o / \partial p_{i,j} = -\delta t / \delta x \quad (III-2b)$$

$$\partial v_{i,j}^o / \partial p_{i,j} = \delta t / \delta y \quad (III-2c)$$

$$\partial v_{i-1,j}^o / \partial p_{i,j} = -\delta t / \delta y \quad (III-2d)$$

Eqs. III-2a thru 2d can be expressed in difference form in respect to iteration cycle.

$$\begin{aligned} u_{i,j}^{n+1} &= u_{i,j}^n + (p_{i,j}^{n+1} - p_{i,j}^n) \delta t / \delta x \\ &= u_{i,j}^n + \delta p \delta t / \delta x \end{aligned} \quad (III-3a)$$

$$\begin{aligned} u_{i-1,j}^{n+1} &= u_{i-1,j}^n - (p_{i,j}^{n+1} - p_{i,j}^n) \delta t / \delta x \\ &= u_{i-1,j}^n - \delta p \delta t / \delta x \end{aligned} \quad (\text{III-3b})$$

$$\begin{aligned} v_{i,j}^{n+1} &= v_{i,j}^n + (p_{i,j}^{n+1} - p_{i,j}^n) \delta t / \delta y \\ &= v_{i,j}^n + \delta p \delta t / \delta y \end{aligned} \quad (\text{III-3c})$$

$$\begin{aligned} v_{i,j-1}^{n+1} &= v_{i,j-1}^n - (p_{i,j}^{n+1} - p_{i,j}^n) \delta t / \delta y \\ &= v_{i,j-1}^n - \delta p \delta t / \delta y \end{aligned} \quad (\text{III-3d})$$

where $\delta p = p_{i,j}^{n+1} - p_{i,j}^n$

$n, n+1 =$ Iteration cycles in new time level.

The velocities u and v are both in the new time level but different iteration cycles as indicated by the superscripts. Substituting Eqs. III - 3a thru 3d into discrepancy equation similar to Eq. 28 yields

$$\begin{aligned} D^{n+1} &= (u_{i,j}^{n+1} - u_{i-1,j}^n) / \delta x + 2 \delta p \delta t / \delta x^2 \\ &\quad + (v_{i,j}^n - v_{i,j-1}^n) / \delta y + 2 \delta p \delta t / \delta y^2 \end{aligned} \quad (\text{III-4})$$

Because a convergence has not been reached in the n th iteration, the discrepancy is

$$D^n = (u_{i,j}^n - u_{i-1,j}^n) / \delta x + (v_{i,j}^n - v_{i,j-1}^n) / \delta y$$

Eq. III-4 is simplified.

$$D^{n+1} = D^n + \delta p (2 \delta t / \delta x^2 + 2 \delta t / \delta y^2) \quad (\text{III-5})$$

To diminish the discrepancy in $(n+1)$ th iteration, i.e., $D^{n+1} = 0$, the pressure is adjusted by an amount of δp .

$$\delta p = - D^n / (2 \delta t / \delta x^2 + 2 \delta t / \delta y^2) \quad (\text{III-6})$$

$$p^{n+1} = p^n + \delta p \quad (\text{III-7})$$

Eq. III-6 is identical to Eq. 29. Eqs. III-3 a, b, c, d are the same as Eqs. 30 a - d, respectively.

APPENDIX IV

PRESSURE ADJUSTMENT AT BOUNDARY

This appendix details the pressure adjustment at exterior of the mesh, either rigid boundary or free surface, i.e., Eqs. 30a - d.

A Free Surface

The pressure at free surface cell is computed from the cells above and below the surface by a linear interpolation.

A.1 Top Free Surface

The pressure at mid-surface is p_s .

$$\begin{aligned} p_s &= p_{i,jT} - (p_{i,jT-1} - p_{i,jT}) \Delta y / \delta y \\ &= (1 + \Delta y / \delta y) p_{i,jT} - p_{i,jT-1} \Delta y / \delta y \\ &= p_{i,jT} / \eta_T + (1 - 1 / \eta_T) p_{i,jT-1} \end{aligned} \quad (IV-1)$$

where $1 / \eta_T = 1 + \Delta y / \delta y$

Re-arranging Eq. IV-1 for $p_{i,jT}$ results

$$p_{i,jT} = \eta_T p_s + (1 - \eta_T) p_{i,jT-1} \quad (IV-2)$$

The factor of surface location is included in the η_T term.

$$\begin{aligned} \eta_T &= \delta y / (\delta y + \Delta y) \\ &= \delta y / (\delta y + H_i - H_c) \\ &= \delta y / [\delta y + H_i - (jT - 1.5) \delta y] \end{aligned}$$

where $\Delta y = H_i - H_c$

$$H_c = (jT - 1.5) \delta y$$

When uniform pressure is applied at the free surface, it is convenient to set $p_s = 0$. Then, Eq. IV-2 becomes

$$p_{i,jT} = (1 - \eta_T) p_{i,jT-1} \quad (IV-3)$$

A value of $p_{i,jT}$ computed by Eq. IV-3 is the correct one. Therefore, the pressure adjustment, δp , is

$$\delta p = (1 - \eta_T) p_{i,jT-1} - p_{i,jT} \quad (31a)$$

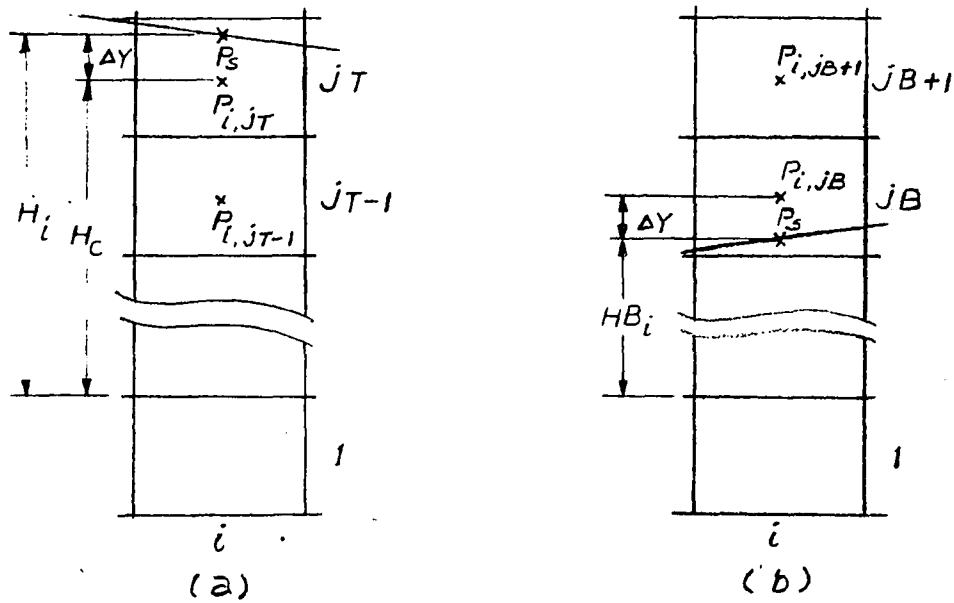


Fig. 36 Pressure adjustment at free surface

(a) Top free surface, (b) Bottom free surface

A.2 Bottom Free Surface

The steps of derivation is similar to top free surface. The terms are shown in Fig. 36(b).

$$\begin{aligned}
 P_s &= P_{i,jB} + (P_{i,jB} - P_{i,jB+1}) \Delta y / \delta y \\
 &= (1 + \Delta y / \delta y) P_{i,jB} - P_{i,jB+1} \Delta y / \delta y \\
 &= P_{i,jB} / \eta_B + (1 - 1/\eta_B) P_{i,jB+1}
 \end{aligned} \tag{IV-4}$$

where $1/\eta_B = 1 + \Delta y / \delta y$

and $\eta_B = \delta y / (\Delta y + \delta y)$
 $= \delta y / [\delta y + (j_B - 1.5)\delta y - HB_i]$

Re-arranging Eq. IV-4 for $p_{i,jB}$,

$$P_{i,jB} = \eta_B P_s + (1 - \eta_B) P_{i,jB+1} \tag{IV-5}$$

When $p_s = 0$, A correct $p_{i,jB}$ is

$$P_{i,jB} = (1 - \eta_B) P_{i,jB+1}$$

The pressure adjustment is

$$\delta p = (1 - \eta_B) P_{i,jB+1} - P_{i,jB} \tag{31b}$$

B. Rigid Surface

The surface pressure at rigid surface will make the normal velocity equal to zero.

$$\begin{aligned}
 u_n &= u \cdot n = 0 \\
 &= (iu - jv) (-i \sin \theta + j \cos \theta) \\
 &= -u \sin \theta + v \cos \theta \\
 u_n / \cos \theta &= -u \tan \theta + v \quad (IV-6)
 \end{aligned}$$

B.1 Bottom Rigid Surface

The horizontal velocity at mid-surface is the average of two cell velocities.

$$u = (u_{i,jB} + u_{i-1,jB})/2 \quad (IV-7)$$

The vertical velocity can be found by interpolation shown in Fig. 37.

$$v = v_{i,jB-1} + (v_{i,jB} - v_{i,jB-1})\Delta y/\delta y \quad (IV-8)$$

The slope of surface is

$$\tan \theta = (HB_{i+1} - HB_{i-1})/2\delta x \quad (IV-9)$$

Substituting Eqs. IV-7, IV-8 and IV-9 into Eq. IV-6 results

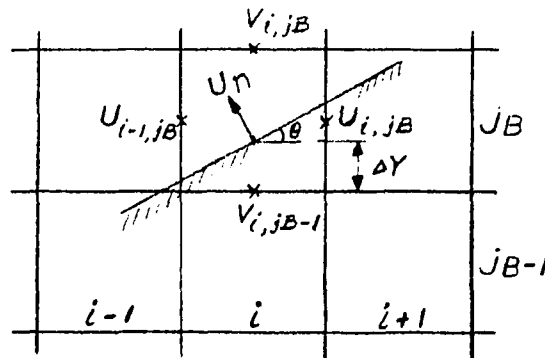


Fig. 37 Pressure adjustment at bottom rigid surface.

$$\begin{aligned} u_n / \cos\theta = & - (u_{i,jB} + u_{i-1,jB}) (HB_{i+1} - HB_{i-1}) / 4\delta x \\ & + v_{i,jB} \Delta y / \delta y + v_{i,jB-1} (1 - \Delta y / \delta y) \end{aligned} \quad (IV-10)$$

Since $v_{i,jB-1}$ is inside the rigid boundary, it would be eliminated by equation of continuity, i.e.,

$$v_{i,jB-1} = v_{i,jB} + (u_{i,jB} - u_{i-1,jB}) \delta y / \delta x \quad (IV-11)$$

Eq. IV-10 is expanded and re-arranged to the following:

$$\begin{aligned} u_n / \cos\theta = & - (u_{i,jB} + u_{i-1,jB}) (HB_{i+1} - HB_{i-1}) / 4\delta x + v_{i,jB} \\ & + (u_{i,jB} - u_{i-1,jB}) (1 - \Delta y / \delta y) (\delta y / \delta x) \end{aligned} \quad (IV-12)$$

The only term to be computed in the right hand side of Eq. IV-12 is y .

$$\begin{aligned} 1 - \Delta y / \delta y = & 1 - [HB_i - (jB-2)\delta y] / \delta y = (jB-1) - HB_i / \delta y \\ = & \lambda_B \end{aligned} \quad (IV-13)$$

The velocity is function of pressure as expressed in Eq. III-2a. The increment of pressure can be found by Newton-Ralphson method.

$$\begin{aligned} \delta p = & - u_n / (\partial u_n / \partial p) \\ = & - (u_n / \cos\theta) / [\partial (u_n / \cos\theta) / \partial p] \end{aligned} \quad (IV-14)$$

The denominator is, from Eqs. IV-12 and IV-13,

$$\begin{aligned} \partial (u_n / \cos\theta) / \partial p = & - (\partial u_{i,jB} / \partial p + \partial u_{i-1,jB} / \partial p) (HB_{i+1} - HB_{i-1}) / 4\delta x \\ & + \partial v_{i,jB} / \partial p + (\partial u_{i,jB} / \partial p - \partial u_{i-1,jB} / \partial p) \lambda_B (\delta y / \delta x) \end{aligned} \quad (IV-15)$$

where

$$p = p_{i,jB}$$

Substituting Eqs. III-2a thru d into Eq. IV-15 yields

$$\begin{aligned} \partial (u_n / \cos\theta) / \partial p = & \delta t / \delta y + 2 \lambda_B (\delta t / \delta x) (\delta y / \delta x) \\ = & [1 + 2 \lambda_B (\delta y / \delta x)^2] \delta t / \delta y \end{aligned} \quad (IV-16)$$

Substituting Eqs. IV-12, IV-13 and IV-16 into IV-14 results Eq. 31d.

B.2 Top Rigid Surface

Similar to the step of derivation in above section for bottom rigid surface, the terms are shown in Fig. 38.

The velocities at mid-surface are

$$u = (u_{i,jT} + u_{i-1,jT})/2 \quad (IV-17)$$

$$\begin{aligned} v &= v_{i,jT-1} + (v_{i,jT} - v_{i,jT-1})\Delta y/\delta y \\ &= v_{i,jT-1} (1 - \Delta y/\delta y) + v_{i,jT} \Delta y/\delta y \end{aligned} \quad (IV-18)$$

The slope of surface is

$$\tan\theta = (H_{i+1} - H_{i-1})/2\delta x \quad (IV-19)$$

The term of $v_{i,jT}$ is eliminated by continuity equation.

$$v_{i,jT} = v_{i,jT-1} - (u_{i,jT} - u_{i-1,jT}) (\delta y/\delta x) \quad (IV-20)$$

Eq. IV-18 becomes

$$v = v_{i,jT-1} - (u_{i,jT} - u_{i-1,jT}) (\delta y/\delta x) (\Delta y/\delta y) \quad (IV-21)$$

$\Delta y/\delta y$ is the only term requiring computation in right hand side of Eq. IV-21.

$$\begin{aligned} \Delta y/\delta y &= [H_i - (jT-2)\delta y]/\delta y = H_i/\delta y - (jT-2) \\ &= \lambda_T \end{aligned} \quad (IV-22)$$

Eq. IV-6 becomes

$$\begin{aligned} u_n/\cos\theta &= - (u_{i,jT} + u_{i-1,jT}) (H_{i+1} - H_{i-1})/4\delta x \\ &+ v_{i,jT-1} - \lambda_T (u_{i,jT} - u_{i-1,jT})\delta y/\delta x \end{aligned} \quad (IV-23)$$

The denominator of Eq. IV-14 is

$$\begin{aligned} \partial(u_n/\cos\theta)/\partial p &= - \delta t/\delta y - 2\lambda_T(\delta t/\delta x) (\delta y/\delta x) \\ &= - [1 + 2\lambda_T (\delta y/\delta x)^2] \delta t/\delta y \end{aligned} \quad (IV-24)$$

Substituting Eqs. IV-23 and IV-24 into Eq. IV-14 results in Eq. 31 c.

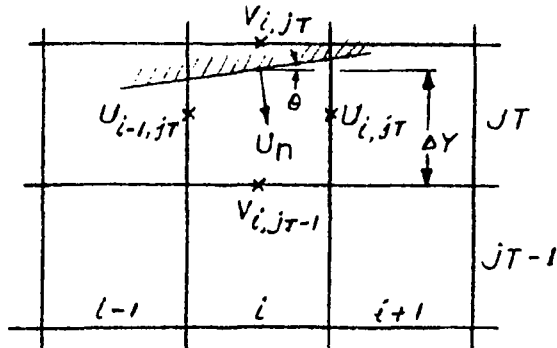


Fig. 38 Pressure adjustment
at top rigid surface

APPENDIX V

FREE SURFACE LOCATION

To Express Eq. 32 in finite difference form for top free surface and bottom free surface is explained in this appendix.

A. Top Free Surface

The vertical velocity at free surface is found by interpolation shown in Fig. 39

$$\begin{aligned} v &= v_{i,jT-1} + (v_{i,jT} - v_{i,jT-1})\Delta y/\delta y \\ &= v_{i,jT} \Delta y/ y + v_{i,jT-1} (1 - \Delta y/\delta y) \\ &= v_{i,jT} \lambda_T + v_{i,jT-1} (1 - \lambda_T) \end{aligned} \quad (V-1)$$

where $\lambda_T =$ Eq. IV-22

The horizontal velocity is the average of upstream and downstream ones.

$$u = (u_{i,jT} + u_{i-1,jT})/2 \quad (V-2)$$

Similar to Eq. I-2, the H terms are defined by partial donor cell method.

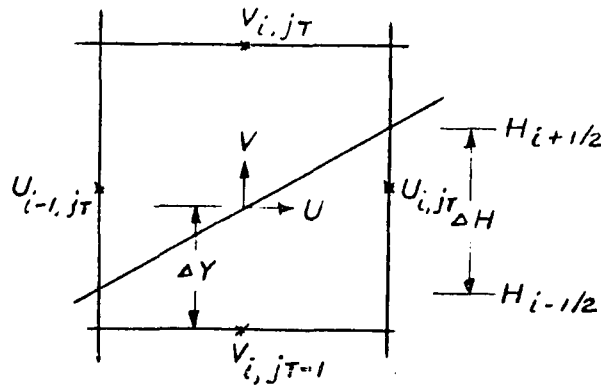
$$H = H_{i+1/2} - H_{i-1/2} \quad (V-3)$$

$$H_{i+1/2} = [(1 + \bar{v}) H_i + (1 - \bar{v}) H_{i+1}]/2 \quad (V-4)$$

$$H_{i-1/2} = [(1 + \bar{v}) H_{i-1} + (1 - \bar{v}) H_i]/2 \quad (V-5)$$

where $\bar{v} =$ weight of donor cell with value $1 \geq \bar{v} \geq -1$

Fig. 39 Location of top free surface



$$\begin{aligned}
 u \Delta H &= (u_{i,jT} + u_{i-1,jT}) (H_{i+1/2} - H_{i-1/2})/2 \\
 &= (u_{i,jT} + u_{i-1,jT}) [H_{i+1} - H_{i-1} \\
 &\quad - \bar{\tau}(H_{i+1} - 2H_i + H_{i-1})]/4 \quad (V-6)
 \end{aligned}$$

Same reasoning as in Appendix I, if

$$\bar{\tau} = \tau (\text{sign } u_{i-1/2,jT}) \quad (1 \geq \tau \geq 0)$$

Eq. V-6 is simplified.

$$\begin{aligned}
 u \Delta H &= [(u_{i,jT} + u_{i-1,jT}) (H_{i+1} - H_{i-1}) - \tau |u_{i,jT} + u_{i-1,jT}| \\
 &\quad (H_{i+1} - 2H_i + H_{i-1})]/4 \quad (V-7)
 \end{aligned}$$

Substituting Eqs. V-1 and V-7 into Eq. 32 results Eq. 33 a

B. Bottom Free Surface

The procedure of derivation of Eq. 33b is similar to the above section. The vertical velocity is

$$\begin{aligned}
 v &= v_{i,jB-1} + (v_{i,jB} - v_{i,jB-1}) \Delta y / \delta y \\
 &= v_{i,jB} \Delta y / \delta y + v_{i,jB-1} (1 - \Delta y / \delta y) \\
 &= (1 - \lambda_B) v_{i,jB} + \lambda_B v_{i,jB-1} \quad (V-8)
 \end{aligned}$$

where $\lambda_B = \text{Eq. IV-13}$

The horizontal velocity is

$$u = (u_{i,jB} + u_{i-1,jB})/2 \quad (V-9)$$

The partial donor cell scheme is applied to HB terms, as shown in Fig. 40.

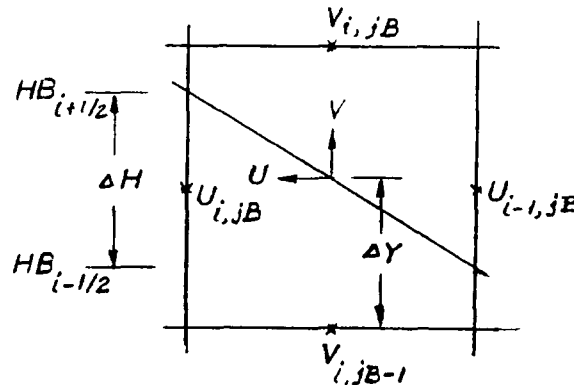


Fig. 40 Location of bottom free surface.

$$\Delta H = HB_{i+1/2} - HB_{i-1/2} \quad (V-10)$$

$$HB_{i+1/2} = [(1 + \bar{\nu}) HB_i + (1 - \bar{\nu}) HB_{i+1}]/2 \quad (V-11)$$

$$HB_{i-1/2} = [(1 + \bar{\nu}) HB_{i-1} + (1 - \bar{\nu}) HB_i]/2 \quad (V-12)$$

Substituting Eqs. V-11 and V-12 into Eq. V-10 and setting

$$\bar{\nu} = \nu (\text{sign } u_{i-1/2, jB})$$

result the following expression:

$$\begin{aligned} u \Delta H = & [(u_{i, jB} + u_{i-1, jB}) (HB_{i+1} - HB_{i-1}) - \\ & \nu |u_{i, jB} + u_{i-1, jB}| (HB_{i+1} - 2HB_i + HB_{i-1})]/4 \end{aligned} \quad (V-13)$$

Again, substituting Eqs. V-8 and V-13 into Eq. 32 yields Eq. 33b.

APPENDIX VI

COMPARISON OF DIMENSION APPROXIMATION IN EXAMPLE

Because of an approximation of physical dimension taken in the example, its effect has to be checked and be sure in negligible range.

A. Excluded Area

The excluded area can be found from table of circular segments (33) as shown in Fig. 41.

$$R = 6.4 \text{ cm}$$

$$h = (5 - 2) \text{ cm} = 0.6 \text{ cm}$$

$$h/R = 0.09375$$

From the table of circular segments, the area is found

$$\text{shaded area}/R^2 = 0.0533$$

Because there are two shaded areas,

$$\text{ratio of excluded area} = 2(0.0533)R^2/\pi R^2 = 3.4\%$$

The excluded area is 3.4% of the cylinder.

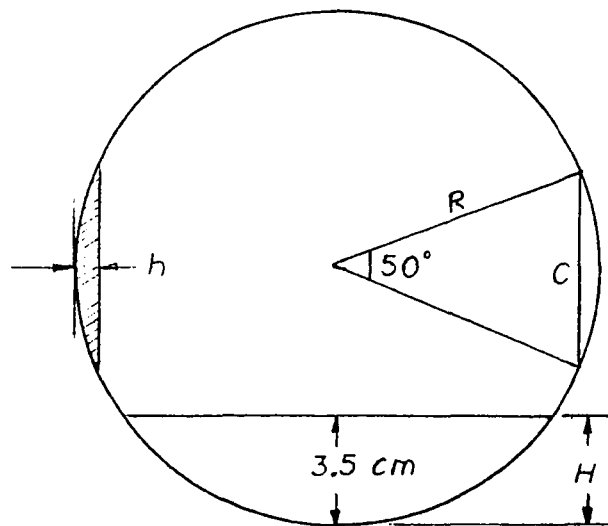


Fig. 41 Physical dimension of example problem.

B. Shortening of Circumference

The circumference of flat wall is also calculated from table of circular segments (33). As ratio of h/R is 0.09375 in above section, it is found

$$c/R = 0.845$$

Therefore, the flat wall (chord) is 5.41 cm and the central angle is 50° .

The segment of circumference is

$$2\pi R (50/360) = 5.58$$

The difference between the complete and the shortened circumferences is

$$2(5.58 - 5.41)/2\pi R = 0.0085$$

or 0.85%, which is negligible.

C. Fluid Occupied Area

The segment area occupied by fluid is computed by the same table (33) as in the above section A.

$$H/R = 3.5/6.4 = 0.547$$

$$\text{area}/R^2 = 0.695$$

$$\text{area} = 0.695(6.4)^2 = 28.47 \text{ cm}^2$$

The result of numerical approximation is 28.32 cm^2 , or 99.5% of the theoretical value.

D. Film Thickness

The film thickness, b , is computed by assuming uniform thickness around the circumference.

$$b = 28.32/2\pi R = 0.7 \text{ cm}$$

The ratio of thickness to the radius of drum is

$$b/R = 0.11$$

Which is smaller than 20% but enough to cover more than a layer of single cell, i.e., 0.4 cm thickness in this example.

E. Time To Cover The Whole Circumference

In the example, the fluid travels from $I = 59$ to the upper half and returns to the lower half. Two portions in the lower half meet at $I = 31$. The total distance can be divided into five parts, as listed in the following table.

Table III
Total Traveling in the Example

<u>Parts</u>	<u>From I =</u>	<u>To I =</u>	<u>Distance (cm)</u>
(1)	59	62	*1
(2)	62	70	5.41 *2
(3)	70	127	14.51 *3
(4)	127	5	5.41 *2
(5)	5	31	7.25 *1

Notes:

1. The total number of cells traveled thru is 29, which is approximated half of the distance in part (3)
2. The chord length computed in sec. (b)
3. The segment for central angle 130° is $2\pi R(130/360) = 14.51$ cm

The total traveling is approximately 32.58 cm. The minimum time to cover the whole circumference is $32.58/220 = 0.148$ sec for rotating velocity 220 cm/sec. That is $0.148/0.0005 = 296$ cycle. The numerical method results a complete ring at 327th cycle.

APPENDIX VII

NUMERICAL METHODS FOR FLUID DYNAMICS

As the physical law in fluid dynamics is expressed by differential equations, the best method of solution is including the most of theoretical method and the least of premanipulation, such as complex variables. But when the problem is more complex, more approximate procedure, e.g., boundary layer, perturbation method, etc., becomes necessary. To even more complicate problem involving odd boundary shape or three dimensions, only the direct numerical method is applicable. In this very last category of problems, a question often raised is how to select a numerical method. This appendix is discussing the criteria of numerical modeling and the applications of the important numerical approaches.

Harlow (47) compiled and classified the numerical methods for transient flow according to the flow speed, coordinate systems, and solution methods. For the high-speed flow, several approaches including Eulerian, Lagrangian, combined Eulerian and Lagrangian, Fourier series, Monte Carlo method, etc., have been tried. For the low-speed flows, the Eulerian approach is dominant. Other methods, such as Lagrangian and Fourier solution, need more studies to improve and extend applications.

The Eulerian approach allows the fluid undergoing great distortions without loss of accuracy, but it loses the sharp definition of contact surface. On the contrary, Lagrangian method can easily apply fine zoning at the interface and arbitrary curve shape of rigid boundary, but it is less accurate when the cells are highly distorted. In the Eulerian approach, the primitive variable method has the advantage of easy applying free surface boundary condition. This gain is offset by the difficulty of

accurate pressure iteration. The vorticity-stream function method eliminates the problems of pressure term and continuity equation. However, it is difficult to encounter with unknown boundary condition on vorticity.

A series of comparisons among nine methods for incompressible, viscous, steady flow in a driven cavity was prepared by Langley Research Center (48). The Reynolds number of the test case was 100. The results indicate that the vorticity - stream function methods have better accuracy and more rapid convergency. The primitive variable methods including SMAC, Spalding, Crocco, and Chorin methods have accuracy very sensitive to the convergency tolerance in the pressure iteration. If the tolerance is refined, computational time will exscalate. The reason is the inadequate pressure iteration scheme. Therefore, the pressure are not in good agreement with those yielded from other methods. But accurate velocities can be obtained, because of the high accuracy in predicting the pressure gradients. Since the techniques were not optimized, the comparison was not definitive.

Recently, Cebeci et al (49) used problems of driven cavity and entrance flow in a channel to compare Spalding method, vorticity - stream function method (by ADI - SOR solution procedure) and stream function method (by biharmonic formulation). The test cases covered a Reynolds number in the range of 50 to 3200. The conclusion is that the biharmonic method is superior to the other two techniques. However, the main drawbacks are the large storage requirement and longer computer time for the matrix computation.

For the free surface flows, Yeung (50) surveyed the methodologies of stream function. Three major methods were reviewed: finite differences, finite elements, and boundary - intergral equations. The finite difference

is used inevitably in transient problems. The finite element has more flexibility to deal with arbitrary boundary geometry with little loss of accuracy. This is the newest method of the above mentioned three. More storage is required and some difficulties as mentioned on page 16 have to be resolved. The integral - equation formulation is able to reduce the space dimension by one and requires less storage. However, there is no accurate and efficient means to integrate the Green function.

The most important criteria of numerical modeling are stability, accuracy, and convergence. The next is size of data-storage. Emmons (51) and Orszag and Israeli (52) discussed those criteria for numerical simulation and served as the excellent starting point in selection of method.

APPENDIX VIII
COMPUTER LISTING OF THE EXAMPLE

REQUESTED OPTIONS (EXECUTE): NODECK,NOLIST,NOXREF,NOMAP,NOFIPS,FIXED,OPT(0),LANGLVL(66)
OPTIONS IN EFFECT: NOLIST NOMAP NOXREF GOSTMT NODECK SOURCE TERM OBJECT FIXED
OPTIMIZE(0) LANGLVL(66) NOFIPS FLAG(I) NAME(MAIN) LINECOUNT(60)

*.....1.....2.....3.....4.....5.....6.....7.....8

C DATA SET SOLAS7 AT LEVEL 223 AS OF 82/05/13

ISN 1 IMPLICIT REAL*(A-H,O-Z)
ISN 2 DIMENSION U(140,35),V(140,35),VN(140,35),VN(140,35),UV(140,35),
IP(140,35),ANG(140,35),H(140),HNI(140),JT(140),JTO(140),JTU(140),
2HB(140),HBN(140),JB(140),RLH(140),BL(140),JBL(70),BR(70),JBR(70),
3DR(70),SLOB(70),CTA(70),VSIN(140),VCOS(140),BU(140),BV(140),
4HTU(140),TU(140),TV(140),TVSIN(140),TVCOS(140),CODTB(140),
5CODBB(140),HBU(140),JBU(140),JBO(140),DCTA(70),DS(140),TTM(140)
ISN 3 REAL NU
ISN 4 INTEGER CYCLE,CODTB,CODBB,SIGNL,TTM
ISN 5 PRINT 35

C * * READ AND PRINT INITIAL INPUT DATA

ISN 6 IBAR=64
ISN 7 JBAR=16
ISN 8 DELX=.2
ISN 9 DELY=.4
ISN 10 AR=JBAR*DELY
ISN 11 DELT=.0005
ISN 12 NU=2.
ISN 13 EPSI=.1
ISN 14 UPSI=.1
ISN 15 FLHT=3.5
ISN 16 UI=0.
ISN 17 VI=0.
ISN 18 AS=220.
ISN 19 DLPHA=.9
ISN 20 ULPHA=.9
ISN 21 GAMMA=.2
ISN 22 OMG=1.4
ISN 23 UMG=1.4
ISN 24 GX=0.
ISN 25 GY=-980.

ISN 26 25 FORMAT(7X,'TOP BOUNDARY CONDITION FAILED TO CONVERGE')
ISN 27 26 FORMAT(7X,'BOTTOM FREE SURFACE MODIFIED',2X,'I=',I3,3X,'JB1=',I3)
ISN 28 27 FORMAT(7X,'* * * BOTTOM SURFACE MODIFIED * * *')
ISN 29 29 FORMAT(10X,'ILL1=',I4,5X,'JT(ILL)=' ,I6,5X,'H(ILL)=' ,IPE12.5)
ISN 30 30 FORMAT(10X,'IR=' ,I4,5X,'JT(IR+1)=' ,I6,5X,'H(IR+1)=' ,IPE12.5)
ISN 31 31 FORMAT(10X,'IRU=' ,I4,5X,'ILU=' ,I4,5X,'IRUT=' ,I6,5X,'TDVOL=' ,IPE12.
15,5X,'KASE=' ,I3)
ISN 32 32 FORMAT(10X,'IRUT=' ,I4,5X,'JB(IRUT)=' ,I6,5X,'HB(IRUT)=' ,IPE12.5)
ISN 33 33 FORMAT(10X,'IRUT=' ,I4,5X,'JB(IRUT+1)=' ,I6,5X,'HB(IRUT+1)=' ,IPE12.5
1)
ISN 34 35 FORMAT(1H1)
ISN 35 38 FORMAT(2X,I3,'-',I2,2X, I3,4(4X,IPE12.5),2(4X,I3),2(6X,IPE12.5))
ISN 36 39 FORMAT(6X,'ITER=' ,I5,10X,'TIME=' ,IPE12.5,10X,'SIGNL=' ,I4,10X,
1'VOLUME=' ,IPE12.5)
ISN 37 40 FORMAT(10X,'TOP BOUNDARY WAS REACHED AT I=' ,I5)
ISN 38 41 FORMAT(10X,'BOTTOM BOUNDARY WAS REACHED AT I=' ,I5)

*.....1.....2.....3.....4.....5.....6.....7.....8

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ISN 39 42 FORMAT(10X,'LR=',I4,5X,'JT(LR)=' ,I6,5X,'H(LR)=' ,IPE12.5)
ISN 40 47 FORMAT(4X,'I',7X,'J',10X,'U',15X,'V',15X,'P',15X,'H',6X,'SUR CELL'
      1,3X,'BOT CELL',9X,'UV',14X,'ANG')
ISN 41 48 FORMAT(2X,I3,5X,I3,4(4X,IPE12.5),2(4X,I3),2(6X,IPE12.5))
ISN 42 49 FORMAT(6X,'ITER=',I5,10X,'TIME=',IPE12.5,10X,'CYCLE=',I4,10X,
      1,VOLUME=',IPE12.5)

```

C

C * * COMPUTE CONSTANT TERMS AND INITIALIZE NECESSARY VARIABLES

C

```

ISN 43 MEET=0
ISN 44 ILU=0
ISN 45 IRU=0
ISN 46 LL=0
ISN 47 LR=0
ISN 48 IMAX=IBAR+2
ISN 49 JMAX=JBAR+2
ISN 50 IM1=IMAX-1
ISN 51 JM1=JMAX-1
ISN 52 RDX=1.0/DELY
ISN 53 RDY=1.0/DELY
ISN 54 IM2=2*JM1
ISN 55 JM2=2*JM1
ISN 56 IMX=IMAX/2
ISN 57 T=0.
ISN 58 CHPRT=50
ISN 59 TWPRT=9.*DELT
ISN 60 SIGNAL=0
ISN 61 TTU=0.
ISN 62 CUPRT=50.
ISN 63 TUPRT=17.*DELT
ISN 64 BETA=OMG/(2.*DELT*(RDX**2+RDY**2))
ISN 65 UBETA=UMG/(2.*DELT*(RDX**2+RDY**2))
ISN 66 FVUU=0.
ISN 67 FLOI=0.
ISN 68 LYLE=0
ISN 69 TTD=0.
ISN 70 CLPRT=10.
ISN 71 TLPRT=8.*DELT

```

C

C * * DETERMINE BOTTOM BOUNDARY LOCATION

C

```

ISN 72 BX=0.
ISN 73 CY=AR
ISN 74 BL(2)=CY*RDY
ISN 75 JBL(2)=BL(2)
ISN 76 DL=BL(2)-DFLOAT(JBL(2))
ISN 77 DO 190 I=2,IMX
ISN 78 BX=BX+DELX
ISN 79 BY=AR-DSQRT(2.*AR*BX-BX*BX)
ISN 80 BR(I)=BY*RDY
ISN 81 JBR(I)=BR(I)
ISN 82 DR(I)=BR(I)-DFLOAT(JBR(I))
ISN 83 NB=JBL(I)-JBR(I)
ISN 84 IF(NB .EQ. 0) GO TO 60
ISN 85 IF(NB .EQ. 1) GO TO 90
ISN 86 IF(NB .EQ. 2) GO TO 130

```

.....1.....2.....3.....4.....5.....6.....7......8

```

87 ISN 60 TO 150
88 ISN CB=DL+DR(I)
89 ISN IF(CB.LT. 1.5) GO TO 150
90 ISN JB(I)=JBL(I)+3
91 ISN GO TO 160
92 ISN 90 BK=DL/(1.-DR(I))
93 ISN RBX=1./(1.+BK)
94 ISN CB=(1.-DR(I))/(1.+BK)
95 ISN IF(CB.GE. .5) GO TO 150
96 ISN JB(I)=JBR(I)+3
97 ISN IF(RBX.GE. 0.5) GO TO 160
98 ISN HB(I)=.5*(CY+BY)
99 ISN GO TO 170
100 ISN 130 CONTINUE
101 ISN JB(I)=JBR(I)+3
102 ISN HB(I)=.5*(BY+CY)
103 ISN GO TO 170
104 ISN 150 HB(I)=.5*(BY+CY)
105 ISN JB(I)= HB(I)*RDY+1.E-8+2.
106 ISN GO TO 170
107 ISN 160 HB(I)=(FLOAT(JB(I))-2)*DELY
108 ISN 170 CONTINUE
109 ISN SLOB(I)= -(AR-(BX-.5*DELY))/(AR-HB(I))
110 ISN SLOB(IMAX-I+1)=SLOB(I)
111 ISN CTA(I)=DATAN(SLOB(I))
112 ISN CTA(IMAX-I+1)=-CTA(I)
113 ISN HB(IMAX-I+1)=HB(I)
114 ISN JB(IMAX-I+1)=JB(I)
115 ISN HBN(I)=HB(I)
116 ISN HBN(IMAX-I+1)=HBN(I)
117 ISN 180 CONTINUE
118 ISN CY=BY
119 ISN BL(I+1)=BR(I)
120 ISN JBL(I+1)=JBR(I)
121 ISN DL=DR(I)
122 ISN DR(IMAX-I+1)=DR(I)
123 ISN 190 CONTINUE
124 ISN HB(1)=HB(2)
125 ISN HB(IMAX)=HB(IM1)
126 ISN JB(1)=JB(2)
127 ISN JB(IMAX)=JB(IM1)
128 ISN CTA(1)=CTA(2)
129 ISN CTA(IMAX)=CTA(IM1)
130 ISN DO 200 I=2,IM1
131 ISN VCOS(I)=DCOS(CTA(I))
132 ISN VSIN(I)=DSIN(CTA(I))
133 ISN 200 CONTINUE

C
C * * COMPONENTS OF ROTATION VELOCITY
C
134 ISN DO 210 I=2,IMX
135 ISN BU(I)=AS*VCOS(I)
136 ISN BV(I)=AS*VSIN(I)
137 ISN BU(IMAX-I+1)=DU(I)
138 ISN BV(IMAX-I+1)=-BV(I)
139 ISN 210 CONTINUE

```

*.....1.....2.....3.....4.....5.....6.....7.....8

C * * COMPUTE INITIAL TOP SURFACE CONFIGURATION

```

140 ISN      DO 230 I=1,IMAX
141 ISN      YTH(I)=0.
142 ISN      H(I)=HB(I)
143 ISN      HN(I)=H(I)
144 ISN      JT(I)=JB(I)
145 ISN      RLH(I)=0.
146 ISN      CODTB(I)=0
147 ISN      CODBB(I)=0
148 ISN      RLH(I+IMAX)=0.
149 ISN      CODTB(I+IMAX)=0
150 ISN      CODBB(I+IMAX)=0
151 ISN      230 CONTINUE
152 ISN      TX=AR-DSQRT(2.*AR*FLHT-FLHT*FLHT)
153 ISN      IL= TX/RDX+1.E-8+2.
154 ISN      JLHT = FLHT*RDY+1.E-8+1.
155 ISN      240 IF(JLHT .GE. JB(IL)) GO TO 250
156 ISN      IL=IL+1
157 ISN      GO TO 240
158 ISN      250 CONTINUE
159 ISN      IR=IMAX-IL+1
160 ISN      ILI=IL-1
161 ISN      IRI=IR+1
162 ISN      DO 260 I=ILI,IRI
163 ISN      H(I)=FLHT
164 ISN      HN(I)=H(I)
165 ISN      JT(I)=JLHT+1
166 ISN      RLH(I)=H(I)-HB(I)
167 ISN      260 CONTINUE
168 ISN      FFVOL=0.
169 ISN      DO 270 I=IL,IR
170 ISN      FFVOL=FFVOL+(H(I)-HB(I))*DELX
171 ISN      270 CONTINUE

```

C * * CALCULATE HYDROSTATIC PRESSURE

```

172 ISN      DO 280 I=2,IM2
173 ISN      DO 280 J=1,JM2
174 ISN      P(I,J)=0.
175 ISN      280 CONTINUE
176 ISN      DO 290 I=IL1,IMX
177 ISN      JT1=JT(I)
178 ISN      JB1=JB(I)
179 ISN      DO 290 J=JB1,JT1
180 ISN      P(I,J)=-6Y*(H(I)-(DFLOAT(J)-1.5)*DELY)
181 ISN      P(IMAX-I+1,J)=P(I,J)
182 ISN      290 CONTINUE

```

C * * SET INITIAL VELOCITY FIELD INTO U AND V ARRAYS

```

183 ISN      DO 300 I=1,IM2
184 ISN      DO 300 J=1,JM2
185 ISN      U(I,J)=UJ
186 ISN      V(I,J)=VJ

```

.....1.....2.....3.....4.....5.....6.....7......8

```

ISN 187 300 CONTINUE
ISN 188 DO 350 I=IL1,IR1
ISN 189 JB1=JB(I)
ISN 190 U(I,JB1)=BU(I)
ISN 191 V(I,JB1)=BV(I)
ISN 192 350 CONTINUE
ISN 193 DO 400 I=IL,IR
ISN 194 CODBB(I)=1
ISN 195 JB1=JB(I)
ISN 196 IF(U(I-1,JB1) .EQ. 0) U(I-1,JB1)=U(I,JB1)
ISN 198 V(I,JB1-1)=V(I,JB1)+DELY*RDY*(U(I,JB1)-U(I-1,JB1))
ISN 199 U(I,JB1-1)=(AS-V(I,JB1-1)*VSIN(I))/VCOS(I)
ISN 200 400 CONTINUE

C * * SET RIGHT AND LEFT BOUNDARY BY SLOP
C
ISN 201 ALMT=DELY*RDY
ISN 202 I=2
ISN 203 420 IF(DABS(SLOB(I)) .LE. ALMT) GO TO 500
ISN 204 DO 450 J=2,JM2
ISN 205 U(I,J)=0.
ISN 206 V(I,J)=-AS
ISN 207 U(IMAX-I+1,J)=0.
ISN 208 V(IMAX-I+1,J)=AS
ISN 209 BU(I)=0.
ISN 210 BV(I)=-AS
ISN 211 BU(IMAX-I+1)=0.
ISN 212 BV(IMAX-I+1)=AS
ISN 213 450 CONTINUE
ISN 214 I=I+1
ISN 215 GO TO 420
ISN 216 500 CONTINUE
ISN 217 ML=I
ISN 218 MR=IMAX-I+1
ISN 219 BU(MR)=0.
ISN 220 BV(MR)=AS
ISN 221 MRU=2*IMAX - MR
ISN 222 MLU=2*IMAX - ML
ISN 223 DO 520 J=2,JM2
ISN 224 U(MR,J)=0.
ISN 225 V(MR,J)=AS
ISN 226 U(MLU,J)=0.
ISN 227 V(MLU,J)=-AS
ISN 228 520 CONTINUE

C * * TOP BOUNDARY FOR LOWER HALF PORTION
C
ISN 229 ML1=ML-1
ISN 230 MR1=MR+1
ISN 231 DO 550 I=ML1,MR1
ISN 232 JIU(I)=JM1+JMAX-JB(I)
ISN 233 HTU(I)=2.*AR -HB(I)
ISN 234 TU(I)=-BU(I)
ISN 235 TV(I)=BV(I)
ISN 236 TVSIN(I)=VSIN(I)
ISN 237 TVCOS(I)=-VCOS(I)

```

*.....1.....2.....3.....4.....5.....6.....7.....8

```

ISN 238 550 CONTINUE
ISN 239   JB1=JB(MR)
ISN 240   JT1=JTU(MR)-1
ISN 241   DO 580 J=JB1,JT1
ISN 242     U(MR,J)=0.
ISN 243     UN(MR,J)=0.
ISN 244     U(MR+1,J)=0.
ISN 245     UN(MR+1,J)=0.
ISN 246 580 CONTINUE
C
C * * TOP BOUNDARY FOR UPPER HALF PORTION
C
ISN 247   DO 600 I=ML1,MR1
ISN 248     IU=2*IMAX-I
ISN 249     TU(IU)=BU(I)
ISN 250     TV(IU)=BV(I)
ISN 251     TVCOS(IU)=VCOS(I)
ISN 252     TVSIN(IU)=VSIN(I)
ISN 253     JTU(IU)=JTU(I)
ISN 254     HTU(IU)=HTU(I)
ISN 255     HBU(IU)=HB(I)
ISN 256     JBU(IU)=JB(I)
ISN 257     JT(IU)=JTU(IU)
ISN 258     H(IU)=HTU(IU)
ISN 259     JB(IU)=JT(IU)
ISN 260     HB(IU)=H(IU)
ISN 261     HBN(IU)=HB(IU)
ISN 262 600 CONTINUE
C
C * * INITIAL VELOCITIES AND PRESSURE IN UPPER HALF
C
ISN 263   DO 610 I=IL1,IM1
ISN 264     IU=2*IMAX-I
ISN 265     DO 610 J=2,JM2
ISN 266       U(IU,J)=0.
ISN 267       V(IU,J)=0.
ISN 268       P(IU,J)=0.
ISN 269 610 CONTINUE
C
C * * BOTTOM BOUNDARY CONDITIONS
C
ISN 270   ILS=IL
ISN 271   IRS=IR
ISN 272   MRS=MR
ISN 273   ITB=0
ISN 274   BFLG=0.
ISN 275   ITB=ITB+1
ISN 276   IF(ITB .GE. 51) GO TO 670
ISN 277   DO 660 I=ILS,IRS
ISN 278     J=JB(I)
ISN 279     VTM=RDY*(HB(I)-(J-2)*DELY)
ISN 280     VBM=RDY*((J-1)*DELY-HB(I))
ISN 281     F=-0.25*RDY*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
ISN 282     I+VBM*(V(I,J)+DELY*RDY*(U(I,J)-U(I-1,J)))
ISN 283     DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*RDY**2)

```

*.....1.....2.....3.....4.....5.....6.....7.....8

```

ISN 284 DELP=-F/DFDP
ISN 285 CORP=DELP/P(I,J)
ISN 286 IF(DABS(CORP) .GE. .05) BFLG=1.
ISN 288 P(I,J)=P(I,J)+DELP
ISN 289 U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
ISN 290 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 291 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 292 U(I,J)=U(I,J)+DELT*RDY*DELP
ISN 293 660 CONTINUE
ISN 294 IF(BFLG .EQ. 0.) GO TO 670
ISN 295 GO TO 650
ISN 296 670 CONTINUE

```

C * * SET BOUNDARY CONDITIONS

```

ISN 297 HN(IL-1)=HN(IL)
ISN 298 JT(IL-1)=JT(IL)
ISN 299 HN(IR+1)=HN(IR)
ISN 300 JT(IR+1)=JT(IR)
ISN 301 JB2=JB(IL)-1
ISN 302 JT1=JT(IL)+1
ISN 303 DO 700 J=JB2,JT1
ISN 304 U(IL-1,J)=U(IL,J)
ISN 305 V(IL-1,J)=V(IL,J)
ISN 306 700 CONTINUE
ISN 307 JT1=JT(IR)+1
ISN 308 JB2=JB(IR)-1
ISN 309 DO 750 J=JB2,JT1
ISN 310 U(IR+1,J)=U(IR,J)
ISN 311 V(IR+1,J)=V(IR,J)
ISN 312 750 CONTINUE
ISN 313 DO 800 I=IL,IR1
ISN 314 JT2=JT(I)+1
ISN 315 JB2=JB(I)-1
ISN 316 DO 800 J=JB2,JT2
ISN 317 UN(I,J)=U(I,J)
ISN 318 VN(I,J)=V(I,J)
ISN 319 HN(I)=H(I)
ISN 320 800 CONTINUE

```

C * * START CYCLE

```

ISN 321 DO 20300 CYCLE=1,561
ISN 322 FLG=1.
ISN 323 ALPHA=DLPHA

```

C * * COMPUTE TEMPORARY U AND V

```

ISN 324 DO 1100 I=IL,IR
ISN 325 JT1= JT(I)
ISN 326 JT11=JT1-1
ISN 327 IF(CODTB(I) .EQ. 1) JT1=JT11
ISN 329 JB1=JB(I)+1
ISN 330 DO 1100 J=JB1,JT1
ISN 331 IF(I .EQ. MRS) GO TO 1050
ISN 332 FUX=((UN(I,J)+UN(I+1,J))*(UN(I,J)+UN(I+1,J))+ALPHA*DABS(UN(I,J)

```


*.....1.....2.....3.....4.....5.....6.....7.....8

```

333      1+UN(I+1,J)*UN(I,J)-UN(I+1,J)-(UN(I-1,J)+UN(I,J))*UN(I-1,J)
        2+UN(I,J)-ALPHA*DABS(UN(I-1,J)+UN(I,J))*UN(I-1,J)-UN(I,J))
        3/(4.0*DELX)
        FUY=((VN(I,J)+VN(I+1,J))*UN(I,J)+UN(I,J+1))
        1+ALPHA*DABS(VN(I,J)+VN(I+1,J))*UN(I,J)-UN(I,J+1)
        2-(VN(I,J-1)+VN(I+1,J-1))*UN(I,J-1)+UN(I,J)
        3-ALPHA*DABS(VN(I,J-1)+VN(I+1,J-1))*UN(I,J-1)-UN(I,J)))/(4.*DELY)
        VTSX=NU*((UN(I+1,J)-2.*UN(I,J)+UN(I-1,J))/DELX**2+
        1*(UN(I,J+1)-2.*UN(I,J)+UN(I,J-1))/DELY**2)
        U(I,J)=UN(I,J)+DELT*(P(I,J)-P(I+1,J))*RDX+GX-FUX-FUY  +VISX
1050 CONTINUE
        FVX=((VN(I,J)+UN(I,J+1))*VN(I,J)+VN(I+1,J)+ALPHA*DABS(UN(I,J)
        1+UN(I,J+1))*VN(I,J)-VN(I+1,J)-(UN(I-1,J)+UN(I,J))*VN(I-1,J)+VN(I,J)
        2+VN(I,J)-ALPHA*DABS(UN(I-1,J)+UN(I,J+1))*VN(I-1,J)-VN(I,J)))/
        3/(4.0*DELX)
        FVY=((VN(I,J)+VN(I+1,J))*VN(I,J)+VN(I,J+1)+ALPHA*DABS(VN(I,J)+VN
        1(I,J+1))*VN(I,J)-VN(I+1,J-1)-(VN(I,J-1)+VN(I,J))*VN(I,J-1)+VN(I,J)
        2))-ALPHA*DABS(VN(I,J-1)+VN(I,J+1))*VN(I,J-1)-VN(I,J)))/(4.*DELY)
        VISY=NU*((VN(I+1,J)-2.*VN(I,J)+VN(I-1,J))/DELX**2+
        1*(VN(I,J+1)-2.*VN(I,J)+VN(I,J-1))/DELY**2)
        V(I,J)=VN(I,J)+DELT*(P(I,J)-P(I,J+1))*RDY+GY-FVX-FVY  +VISY
1100 CONTINUE
C
C * * BOTTOM BOUNDARY CONDITIONS
C
342      ITB=0
343      CONTINUE
344      BFLG=0.
345      ITB=ITB+1
346      IF(ITB .GE. 51) GO TO 1400
347      DO 1300 I=ILS,IRS
348         J=JB(I)
349         VTH=RDY*(HB(I)-(J-2)*DELY)
350         VBM=RDY*((J-1)*DELY-HB(I))
351         F=-0.25*RDX*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTH
        1+VBM*(V(I,J)+DELY*RDX*(U(I,J)-U(I-1,J)))
        DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*RDX**2)
        DELP=-F/DFDP
        CORP=DELP/P(I,J)
        IF(DABS(CORP) .GE. .05) BFLG=1.
        P(I,J)=P(I,J)+DELP
        IF(I .EQ. LL) GO TO 1270
        U(I-1,J)=U(I-1,J)-DELT*RDX*DELP
1270 CONTINUE
        V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
        V(I,J)=V(I,J)+DELT*RDY*DELP
        IF(I .EQ. MRS) GO TO 1300
        U(I,J)=U(I,J)+DELT*RDX*DELP
1300 CONTINUE
        IF(BFLG .EQ. 0.) GO TO 1400
        GO TO 1250
1400 CONTINUE
C
C * * TOP RIGID BOUNDARY CONDITIONS
C
369      ITU=0

```

```

*.....1.....2.....3.....4.....5.....6.....7.....8
ISN 370 1500 ITU=ITU+1
ISN 371 UFLG=0.
ISN 372 IF(ITU .GE. 50) GO TO 1700
ISN 373 DO 1600 I=ILS,IRS
ISN 374 IF(CODTB(I) .EQ. 0) GO TO 1600
ISN 375 J=JT(I)
ISN 376 VTM=RDY*(HTU(I)-(J-2)*DELY)
ISN 377 F=-.25*RDY*(HTU(I+1)-HTU(I-1))*(U(I,J)+U(I-1,J)) +V(I,J-1)
1-VTM*DELY*RDY*(U(I,J)-U(I-1,J))
DFDP=-DELT*RDY*(1.+2.*VTM*DELY**2*RDY**2)
DELP=-F/DFDP
CORP=DELP/P(I,J)
IF(DABS(CORP) .GE. .05) UFLG=1.
P(I,J)=P(I,J)+DELP
V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
V(I,J)=V(I,J)+DELT*RDY*DELP
U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
IF(I .EQ. MRS) GO TO 1600
U(I,J)=U(I,J)+DELT*RDY*DELP
1600 CONTINUE
ISN 389
ISN 390 IF(UFLG .EQ. 0.) GO TO 1700
ISN 391 GO TO 1500
ISN 392 1700 CONTINUE
C
C * * HAS CONVERGENCE BEEN REACHED
C
C DO 3600 ITER=1,200
C * * SET LEFT AND RIGHT BOUNDARY CONDITIONS
C
ISN 394 JB2=JB(IR)-1
ISN 395 JT2=JT(IR)+1
ISN 396 IF(IR .LT. MR) GO TO 1810.
ISN 397 DO 1800 J=JB2,JT2
ISN 398 V(IR+1,J)=V(IR,J)*2. -V(IR-1,J)
ISN 399 1800 CONTINUE
ISN 400 GO TO 1855
ISN 401 1810 CONTINUE
ISN 402 DO 1850 J=JB2,JT2
ISN 403 IF(J .EQ. JB(IR+1)) GO TO 1850
ISN 404 IF(IR+1 .EQ. MR) GO TO 1850
ISN 405 V(IR+1,J)=V(IR,J)
ISN 406 U(IR+1,J)=U(IR,J)
ISN 407 1850 CONTINUE
ISN 408 1855 CONTINUE
ISN 409 U(IR+1,JB2+1)=BU(IR+1)
ISN 410 V(IR+1,JB2+1)=BV(IR+1)
C
C * * FREE SURFACE CONDITIONS
C
ISN 411 DO 2620 I=IL,IR
ISN 412 IF(H(I) .GE. HTU(I)) GO TO 2620
ISN 413 JT1=JT(I)
ISN 414 IF(I .LE. 6) GO TO 2600
ISN 415 IF(JT(I+1).LT.JT(I)) U(I,JT1)=U(I,JT1-1)
ISN 417 2600 CONTINUE

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*.....1.....2.....3.....4.....5.....6.....7.....8

```

ISN 418 V(I,JTI)=V(I,JTI-1)-DELY*RDY*(U(I,JTI)-U(I-1,JTI))
ISN 419 U(I,JTI+1)=U(I,JTI)
ISN 420 2620 CONTINUE

```

C * * COMPUTE UPDATED CELL PRESSURE AND VELOCITY

```

ISN 421 3050 FLG=0.0
ISN 422 DO 3500 I=IL,IR
ISN 423 D=0.
ISN 424 JTI=JT(I)
ISN 425 JTI=JT(I)-1
ISN 426 IF(CODTB(I) .EQ. 1) JTI=JTI+1
ISN 428 JBI=JB(I)+1
ISN 429 DO 3500 J=JBI,JTI
ISN 430 IF(J .NE. JTI) GO TO 3200
ISN 431 IF(CODTB(I) .EQ. 1) GO TO 3500
ISN 432 PETA=DELY/(HN(I)-(DFLOAT(JTI)-2.5)*DELY)
ISN 433 DELP=(1.0-PETA)*P(I,JTI-1)-P(I,JTI)
ISN 434 GO TO 3300
ISN 435 3200 CONTINUE
ISN 436 D=RDY*(U(I,J)-U(I-1,J))+RDY*(V(I,J)-V(I,J-1))
ISN 437 IF(DABS(D) .GE.EPSI)FLG=1.0
ISN 439 DELP=-BETA*D
ISN 440 P(I,J)=P(I,J)+DELP
ISN 441 IF(I .EQ. MR) GO TO 3400
ISN 442 U(I,J)=U(I,J)+DELT*RDY*DELP
ISN 443 3400 CONTINUE
ISN 444 IF(I .EQ. LL) GO TO 3450
ISN 445 U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
ISN 446 3450 CONTINUE
ISN 447 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 448 IF(J .EQ. JBI) GO TO 3500
ISN 449 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 450 3500 CONTINUE
ISN 451 IF(FLG .EQ. 0.) GO TO 4000
ISN 452 3600 CONTINUE

```

C * * COMPUTE NEW SURFACE POSITION

```

ISN 453 4000 CONTINUE
ISN 454 DO 4100 I=IL,IR
ISN 455 IF(I .EQ. MR) GO TO 4050
ISN 456 JTI=JTI+1
ISN 457 HV=RDY*(HN(I)-DFLOAT(JTI-2))*DELY
ISN 458 UAV=0.5*(U(I-1,JTI)+U(I,JTI))
ISN 459 H(I)=HN(I)+DELT*(HV*V(I,JTI)+(1.0-HV)*V(I,JTI-1))
ISN 460 1 -0.5*RDY*(UAV*HN(I+1)+GAMMA*DABS(UAV)*(HN(I)-HN(I+1)))
ISN 461 2 -UAV*HN(I-1)-GAMMA*DABS(UAV)*(HN(I-1)-HN(I)))
ISN 462 IF(H(I) .LT. HB(I)) H(I)=HB(I)
ISN 463 GO TO 4080
ISN 464 4050 CONTINUE
ISN 465 H(I)=HN(I)+AS*DELT
ISN 466 4080 CONTINUE
ISN 467 IF(H(I) .GT. HTU(I)) H(I)=HTU(I)
ISN 468 RLH(I)=H(I)-HB(I)
ISN 469 4100 CONTINUE

```

.....1.....2.....3.....4.....5.....6.....7......8

C * * CALCULATE CELL IN WHICH SURFACE IS LOCATED AND UPDATE ARRAY

```

470 ISN DO 4250 I=IL,IR
471 ISN JTO(I)=JT(I)
472 ISN JT(I)= H(I)*RDY+1.0E-8+2.
473 ISN IF(JT(I) .GT. JTO(I)) JT(I)=JTO(I)
475 ISN IF(JT(I) .EQ. JTO(I)) GO TO 4250
476 ISN JTI=JT(I)
477 ISN IF(JTI .GT. JTO(I)) GO TO 4240
478 ISN JI=JTI+1
479 ISN JTOI=JTO(I)
480 ISN DO 4230 J=JI,JTOI
481 ISN U(I,J)=0.
482 ISN V(I,J)=0.
483 ISN P(I,J)=0.
484 ISN 4230 CONTINUE
485 ISN
486 ISN 4240 JTI=JTO(I)+1
487 ISN DO 4250 J=JTI,JTI
488 ISN P(I,J)=-GY*(H(I)-(DFLOAT(J)-1.5)*DELY)
489 ISN 4250 CONTINUE

```

C * * CHECK REACH THE TOP SURFACE

```

490 ISN ILUB=0
491 ISN DO 4270 I=IL,IR
492 ISN CODBB(I)=0
493 ISN CODTB(I)=0
494 ISN IF(H(I) .NE. HTU(I)) GO TO 4270
495 ISN IF(ILUB .NE. 0) GO TO 4260
496 ISN ILUB=2*IMAX - I
497 ISN HBN(ILUB)=HBN(I)
498 ISN JB2=JB(I)-1
499 ISN JT2=JT(I)+1
500 ISN DO 4260 J=JB2,JT2
501 ISN U(ILUB-1,J)=-U(I,J)
502 ISN U(ILUB,J)=-U(I-1,J)
503 ISN V(ILUB,J)= V(I,J)
504 ISN V(ILUB-1,J)=V(I+1,J)
505 ISN P(ILUB,J)=P(I,J)
506 ISN 4260 CONTINUE
507 ISN IU=2*IMAX-I
508 ISN CODBB(IU)=1
509 ISN CODTB(IU)=1
510 ISN CODTB(I)=1
511 ISN JTI=JT(I)
512 ISN U(I,JTI)=TU(I)
513 ISN V(I,JTI)=TV(I)
514 ISN U(I,JTI+1)=U(I,JTI)
515 ISN V(I,JTI+1)=V(I,JTI)
516 ISN 4270 CONTINUE

```

C * * NEW IL LOCATION

```

517 ISN IF(MEET .EQ. 0) GO TO 4550

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*.....1.....2.....3.....4.....5.....6.....7.....8

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ISN 518      IL8=IL
ISN 519      GO TO 4650
ISN 520      4550 CONTINUE
ISN 521      IL8=IL+4
ISN 522      K=IL
ISN 523      DO 4600 I=K,IL8
ISN 524      IF(JT(I) .GT. JB(I)) GO TO 4600
ISN 525      CODBB(I)=0
ISN 526      IL=I+1
ISN 527      4600 CONTINUE
ISN 528      4650 CONTINUE
ISN 529      IR2=IR-1
ISN 530      DO 4700 I=IL8,IR2
ISN 531      IF(JT(I) .GT. JB(I)) GO TO 4700
ISN 532      JT(I)=JB(I)+1
ISN 533      J=JT(I)
ISN 534      H(I)=(DFLOAT(J)-2.)*DELY
ISN 535      RLH(I)=H(I)-HB(I)
ISN 536      V(I,J)=V(I+1,J)
ISN 537      U(I,J)=U(I-1,J)+(V(I,J-1)-V(I,J))*DELX*RDY
ISN 538      P(I,J)=-GY*(H(I)-(DFLOAT(J)-1.5)*DELY)
ISN 539      4700 CONTINUE
ISN 540      ILL=IL-1
C
C * * CALCULATE NEW VOLUME
C
ISN 541      FVOL=0.
ISN 542      DO 4280 I=IL,IR
ISN 543      FVOL=FFVOL+(H(I)-HB(I))*DELX
ISN 544      4280 CONTINUE
ISN 545      IF(FFVOL .LE. FVOL) GO TO 4400
C
C * * NEW IR LOCATION
C
ISN 546      4300 CONTINUE
ISN 547      IF (IR .EQ. MR) GO TO 4400
ISN 548      NIR=JT(IR)-JB(IR)
ISN 549      IF(NIR .LE. 0) GO TO 4350
ISN 550      NIR=JF(IR)-JB(IR+1)
ISN 551      IF(NIR .LT. 1) GO TO 4400
ISN 552      H(IR+1)=HB(IR+1)+(FFVOL-FVOL)*RDX
ISN 553      JT(IR+1)= H(IR+1)*RDY+2.
ISN 554      IF(JT(IR+1)-JB(IR+1) .LT. 1) GO TO 4400
ISN 555      PRINT 30,IR,JT(IR+1),H(IR+1)
ISN 556      IR=IR+1
ISN 557      CODBB(IR)=1
ISN 558      JTL=JT(IR)
ISN 559      JB1=JB(IR)
ISN 560      DO 4320 J=JB1,JTL
ISN 561      P(IR,J)=P(IR-1,J)
ISN 562      IF (IR .EQ. MR) GO TO 4320
ISN 563      U(IR,J)=U(IR-1,J)
ISN 564      V(IR,J)=V(IR-1,J)
ISN 565      4320 CONTINUE
ISN 566      U(IR,JB1)=BU(IR)
ISN 567      V(IR,JB1)=BV(IR)

```

.....1.....2.....3.....4.....5.....6.....7..........8

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568 ISN RLH(IR)=H(IR)-HB(IR)
569 ISN U(IR,JBI-1)=V(IR,JBI)+(U(IR,JBI)-U(IR-1,JBI))*DELY*RDY
570 ISN U(IR,JBI-1)=(AS-V(IR,JBI-1)*VSIN(IR))/VCOS(IR)
571 ISN GO TO 4400
572 ISN IR=IR-1
573 ISN COBB(IR+1)=0
574 ISN NIR=JT(IR)-JB(IR)
575 ISN IF(NIR .LT. 1) GO TO 4350
576 ISN 4400 CONTINUE
577 ISN IF(IR .EQ. MR) GO TO 4500
578 ISN IR1=IR+1
579 ISN JB1=JB(IR1)
580 ISN U(IR1,JBI)=BU(IR1)
581 ISN V(IR1,JBI)=BV(IR1)
582 ISN P(IR1,JBI)=P(IR,JBI)

```

C * * BOTTOM BOUNDARY CONDITIONS

```

583 ISN 4500 CONTINUE
584 ISN MRS=MR
585 ISN IRS=IR
586 ISN ILS=IL
587 ISN ITB=0
588 ISN 4710 CONTINUE
589 ISN BFLG=0.
590 ISN ITB=ITB+1
591 ISN IF(ITB .GE. 51) GO TO 4730
592 ISN DO 4720 I=ILS,IRS
593 ISN J=JB(I)
594 ISN VTM=RDY*(HB(I)-(J-2)*DELY)
595 ISN VBM=RDY*((J-1)*DELY-HB(I))
596 ISN F=-0.25*RDY*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
597 ISN 1+VBM*(V(I,J)+DELY*RDY*(U(I,J)-U(I-1,J)))
598 ISN DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*RDY**2)
599 ISN DELP=-F/DFDP
600 ISN CORP=DELP/P(I,J)
601 ISN IF(DABS(CORP) .GE. .05) BFLG=1.
602 ISN P(I,J)=P(I,J)+DELP
603 ISN IF(I .EQ. LL) GO TO 4715
604 ISN U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
605 ISN 4715 CONTINUE
606 ISN V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
607 ISN V(I,J)=V(I,J)+DELT*RDY*DELP
608 ISN IF(I .EQ. MRS) GO TO 4720
609 ISN U(I,J)=U(I,J)+DELT*RDY*DELP
610 ISN 4720 CONTINUE
611 ISN IF(BFLG .EQ. 0.) GO TO 4730
612 ISN GO TO 4710
613 ISN 4730 CONTINUE

```

C * * TOP BOUNDARY CONDITIONS

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614 ISN ILS=IL
615 ISN IRS=IR
616 ISN MRS=MR
617 ISN ITU=0

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*.....1.....2.....3.....4.....5.....6.....7.....8

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ISN 618 4750 ITU=YTU+1
ISN 619 UFLG=0.
ISN 620 IF(ITU .GE. 50) GO TO 4770
ISN 621 DO 4760 I=ILS,IRS
ISN 622 IF(COBTB(I) .EQ. 0) GO TO 4760
ISN 623 J=JT(I)
ISN 624 VTM=RDY*(HTU(I)-(J-2)*DELY)
ISN 625 F=-.25*RDY*(HTU(I+1)-HTU(I-1))*(U(I,J)+U(I-1,J)) +V(I,J-1)
      1-VTM*DELY*RDY*(U(I,J)-U(I-1,J))
ISN 626 DFDP=-DELT*RDY*(1.+2.*VTM*DELY**2*RDY**2)
ISN 627 DELP=-F/DFDP
ISN 628 CORP=DELP/P(I,J)
ISN 629 IF(DABS(CORP) .GE. .05) UFLG=1.
ISN 630 P(I,J)=P(I,J)+DELP
ISN 631 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 632 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 633 U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
ISN 634 IF(I .EQ. MRS) GO TO 4760
ISN 635 U(I,J)=U(I,J)+DELT*RDY*DELP
ISN 636 4760 CONTINUE
ISN 637 IF(UFLG .EQ. 0.) GO TO 4770
ISN 638 GO TO 4750
ISN 639

C * * IR VELOCITIES
C
C
4770 CONTINUE
ISN 640 JB2=JB(IR)-1
ISN 641 JT2=JT(IR)+1
ISN 642 IF(IR .LT. MR) GO TO 4790
ISN 643 DO 4780 J=JB2,JT2
ISN 644 V(IR+1,J)=V(IR,J)*2. -V(IR-1,J)
ISN 645 4780 CONTINUE
ISN 646 GO TO 4798
ISN 647
ISN 648 4790 CONTINUE
ISN 649 DO 4795 J=JB2,JT2
ISN 650 IF(J .EQ. JB(IR+1)) GO TO 4795
ISN 651 V(IR+1,J)=V(IR,J)
ISN 652 IF(IR+1 .EQ. MR) GO TO 4795
ISN 653 U(IR+1,J)=U(IR,J)
ISN 654 4795 CONTINUE
ISN 655 V(IR+1,JB2+1)=BV(IR+1)
ISN 656 U(IR+1,JB2+1)=BU(IR+1)
ISN 657 4798 CONTINUE

C * * FREE SURFACE CONDITION
C
C
4800 I=IL,IR
ISN 658 IF(H(I) .GE. HTU(I)) GO TO 4800
ISN 659 JT1=JT(I)
ISN 660 IF(I .LE. 6) GO TO 4799
ISN 661 IF(JT(I+1).LT.JT(I)) U(I,JT1)=U(I,JT1-1)
ISN 662 4799 CONTINUE
ISN 664 V(I,JT1)=V(I,JT1-1)-DELY*RDY*(U(I,JT1)-U(I-1,JT1))
ISN 665 U(I,JT1+1)=U(I,JT1)
ISN 666 4800 CONTINUE
ISN 667
C

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*.....1.....2.....3.....4.....5.....6.....7.....8
C * * CALCULATE TOTAL VOLUME
C
5000 CONTINUE
FVOL=0.
DO 5100 I=IL,IR
  CODBB(I)=1
  FVOL=FVOL+(H(I)-HB(I))*DELX
5100 CONTINUE
PRINT 49,ITER,T,CYCLE,FVOL
C
C * * CALCULATE VELOCITY VECTOR
C
5200 CONTINUE
IF(T+1.E-6.LT.TWPRT) GO TO 6000
TWPRT=TWPRT+CWPRT*DELT
DO 5300 I=IL,IR
  JB2=JB(I)-1
  JT2=JT(I)+1
  DO 5300 J=JB2,JT2
    UV(I,J)=DSQRT(U(I,J)*U(I,J)+V(I,J)*V(I,J))
    IF(DABS(U(I,J)).LE.1.E-6) U(I,J)=1.E-6
    ANG(I,J)=57.3*DATAN(V(I,J)/U(I,J))
    IF(U(I,J).EQ.1.E-6) U(I,J)=0.
5300 CONTINUE
C
C * * LIST VELOCITY, PRESSURE, AND SURFACE POSITION
C
PRINT 35
PRINT 47
DO 5900 I=IL,IR
  JT1=JT(I)
  JT2=JT1+1
  JB1=JB(I)
  JB2=JB1-1
  DO 5900 J=JB2,JT2
    PRINT 48,I,J,U(I,J),V(I,J),P(I,J), H(I),JTL,JB1,UV(I,J),ANG(I,J)
5900 CONTINUE
C
C * * SET THE ADVANCE TIME VELOCITIES U AND V INTO THE UN AND VN ARRAYS
C * * AND THE ADVANCE TIME SURFACE HEIGHT H INTO THE HN ARRAY
C
6000 CONTINUE
DO 6010 I=IL,IR1
  JT2=JT(I)+1
  JB2=JB(I)-1
  DO 6010 J=JB2,JT2
    UN(I,J)=U(I,J)
    VN(I,J)=V(I,J)
    HN(I)=H(I)
6010 CONTINUE
C
C * * INPUT TO UPPER SECTION
C
IF(ILUB.EQ.0) GO TO 16200
TDVOL=FFVOL-FVOL-FV0U-FLOL
IRUT=ILUB+1

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*.....1.....2.....3.....4.....5.....6.....7.....8
711 IF(IRU.EQ.0) GO TO 6020
712 IF(IRUT.LT.IRU) GO TO 6030
713 IF(IRUT.EQ.ILU) GO TO 6040
714 IF(IRUT.GT.ILU) GO TO 6050
715 KASE=5.
716 IRU=IRUT
717 PRINT 31,IRU,ILU,IRUT,TDVOL,KASE
718 GO TO 6060
719 KASE=1.
720 GO TO 6060
721 KASE=2.
722 GO TO 6060
723 KASE=3.
724 FVOU=(H(ILU)-HB(ILU))*DELX
725 TDVOL=FFVOL-FVOL-FVOU
726 PRINT 31,IRU,ILU,IRUT,TDVOL,KASE
727 GO TO 6060
728 KASE=4.
729 TDVOL=FFVOL-FVOL
730 CONTINUE
731 GO TO (6105,7000,11000,6105,11000),KASE
732 CONTINUE
733 PRINT 31,IRU,ILU,IRUT,TDVOL,KASE
734 IF(TDVOL.LE.0.) GO TO 16200
735 HB(IRUT)=H(IRUT)-TDVOL*RDY
736 JB(IRUT)= HB(IRUT)*RDY + 1.E-6*2.
737 PRINT 32,IRUT,JB(IRUT),HB(IRUT)
738 IF(HB(IRUT).LE.H(2*IMAX-IRUT)) GO TO 6110
739 IF(JT(IRUT).EQ.JB(IRUT)) GO TO 16200
740 IRU=IRUT
741 IRU1=IRU-1
742 GO TO 6120
743 CONTINUE
744 IRU=IRUT
745 HB(IRU)=H(2*IMAX-IRU)
746 IRU1=IRU-1
747 JB(IRU)=JT(2*IMAX-IRU)
748 CONTINUE
749 HBN(IRU)=HB(IRU)
750 ILU=IRU
751 ILU1=ILU+1
752 JTI=JT(IRU)
753 JB1=JB(IRU)
754 IF(ILUB.NE.2*IMAX-MR) GO TO 6150
755 DO 6130 J=JB1,JTI
756 U(IRU,J)=U(ILUB,J)
757 V(IRU,J)=V(ILUB,J)
758 P(IRU,J)=GY*((DFLOAT(J)-1.5)*DELY-HB(IRU))
759 CONTINUE
760 U(IRU,JTI)=TU(IRU)
761 V(IRU,JTI)=TV(IRU)
762 CODTB(IRU)=1
763 GO TO 15000
764 CONTINUE
765 JTI=JT(IRU)
766 JB1=JB(IRU)

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*.....1.....2.....3.....4.....5.....6.....7.....8

ISN 767 DO 6170 J=JBI,JTI
 ISN 768 JI=J+JT(IRU-1)-JT(IRU)
 ISN 769 J2=J+JT(IRU-2)-JT(IRU)
 ISN 770 U(IRU,J)=2.*U(IRU-1,J1)-U(IRU-2,J2)
 ISN 771 V(IRU,J)=2.*V(IRU-1,J1)-V(IRU-2,J2)
 ISN 772 P(IRU,J)=GY*((DFLOAT(J)-1.5)*DELY-HB(IRU))
 ISN 773 6170 CONTINUE
 ISN 774 . CODTB(IRU)=1
 ISN 775 U(IRU,JTI)=TU(IRU)
 ISN 776 V(IRU,JTI)=TV(IRU)

C * * TOP BOUNDARY CONDITIONS

ISN 777 6250 CONTINUE
 ISN 778 ITU=0
 ISN 779 6300 ITU=ITU+1
 ISN 780 UFLG=0.
 ISN 781 IF(ITU .GE. 500) GO TO 6400
 ISN 782 DO 6350 I=IRU,ILU
 ISN 783 J=JT(I)
 ISN 784 VTM=RDY*(HTU(I)-(J-2)*DELY)
 ISN 785 IF(DABS(U(I-1,J)) .LE. 1.E-6) U(I-1,J)=U(I,J)
 ISN 787 F=-0.25*RDY*(H(I+1)-H(I-1))*(U(I,J)+U(I-1,J))+V(I,J-1)
 ISN 788 1 -VTM*DELY*RDY*(U(I,J)-U(I-1,J))
 DFDP= -DELT*RDY*(1.0+2.0*VTM*DELY**2*RDY**2
 1 +0.25*RDY**2*DELY*(H(I+1)-H(I-1))*(AMAX0(0,JT(I)-JT(I-1))
 2 -AMAX0(0,JT(I)-JT(I+1))))
 DELP=-F/DFDP
 IF(P(I,J) .EQ. 0.) P(I,J)=P(I,J-1)
 IF(P(I,J) .EQ. 0.) GO TO 6350
 CORP=DELP/P(I,J)
 IF(DABS(CORP) .GE. 0.05 .AND. DABS(DELP) .GE. 200.) UFLG=1.
 P(I,J)=P(I,J)+DELP
 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
 V(I,J)=V(I,J)+DELT*RDY*DELP
 U(I,J)=U(I,J)+DELT*RDY*DELP
 IF(I .EQ. IRU) GO TO 6350
 U(I-1,J)=U(I-1,J)-DELT*RDY*DELP

6350 CONTINUE

IF(UFLG .EQ. 0.) GO TO 6400

GO TO 6300

6400 CONTINUE

IF(ITU .GE. 500) GO TO 6410

GO TO 6420

6410 PRINT 25

6420 CONTINUE

C

C

ISN 810 DO 6450 I=IRU,ILU
 ISN 811 JTI=JT(I)
 ISN 812 JT2=JTI+1
 ISN 813 U(I,JT2)=U(I,JTI)
 ISN 814 V(I,JT2)=V(I,JTI)
 ISN 815 6450 CONTINUE

C * * BOTTOM SURFACE CONDITIONS -- FREE SURFACE

.....1.....2.....3.....4.....5.....6.....7..........8

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ISN 816      6470 CONTINUE
ISN 817      DO 6500 I=IRU,ILU
ISN 818      JB1=JB(I)
ISN 819      V(I,JB1-1)=(V(I,JB1)+V(I,JB1-2)+V(I-1,JB1-1)+V(I+1,JB1-1))*0.25
ISN 820      IF(I .EQ. MLU) GO TO 6500
ISN 821      U(I,JB1-1)=(U(I,JB1)+U(I,JB1-2)+U(I-1,JB1-1)+U(I+1,JB1-1))*0.25
ISN 822      6500 CONTINUE
C
ISN 823      JTI=JT(ILU+1)
ISN 824      JB1=JB(ILU)-1
ISN 825      DO 6550 J=JB1,JTI
ISN 826      U(ILU,J)=U(ILU,J)
ISN 827      V(ILU,J)=V(ILU,J)
ISN 828      6550 CONTINUE
ISN 829      JTI=JT(ILU+1)
ISN 830      U(ILU,JTI)=TU(ILU)
ISN 831      V(ILU,JTI)=TV(ILU)
ISN 832      IF(KASE .EQ. 3) GO TO 6590
ISN 833      GO TO 15000
ISN 834      6590 CONTINUE
ISN 835      DO 6600 I=IRU,ILU
ISN 836      JB2=JB(I)-1
ISN 837      JT2=JT(I)+1
ISN 838      DO 6600 J=JB2,JT2
ISN 839      UN(I,J)=U(I,J)
ISN 840      VN(I,J)=V(I,J)
ISN 841      6600 CONTINUE
ISN 842      GO TO 11000
C
ISN 843      7000 CONTINUE
ISN 844      PRINT 31,IRU,ILU,IRUT,TDVOL,KASE
ISN 845      DO 7100 I=IRUT,IRU
ISN 846      JT2=JT(I)+1
ISN 847      JB2=JB(I)-1
ISN 848      DO 7100 J=JB2,JT2
ISN 849      UN(I,J)=-UN(2*IMAX-I-1,J)
ISN 850      VN(I,J)=VN(2*IMAX-I,J)
ISN 851      7100 CONTINUE
ISN 852      IRU=IRUT
C * * START CYCLE
C
ISN 853      11000 CONTINUE
ISN 854      FLG=1.
ISN 855      ALPHA=ALPHA
C * * COMPUTE TEMPORARY U AND V
C
ISN 856      DO 11100 I=IRU,ILU
ISN 857      JTI=JT(I)-1
ISN 858      JB1=JB(I)
ISN 859      DO 11100 K=JB1,JTI
ISN 860      J=JTI+JB1-K

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*.....1.....2.....3.....4.....5.....6.....7.....8

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861 ISN IF(I .EQ. MLU)GO TO 11050
862 ISN FUX=((UN(I,J)+UN(I+1,J))*UN(I,J)+UN(I+1,J))+ALPHA*DABS(UN(I,J)
      1+UN(I+1,J))*UN(I,J)-UN(I+1,J))-UN(I-1,J)+UN(I,J))*UN(I-1,J)
      2+UN(I,J))-ALPHA*DABS(UN(I-1,J)+UN(I,J))*UN(I-1,J)-UN(I,J)))
      3/(4.0*DELX)
863 ISN FUY=((UN(I,J)+VN(I+1,J))*UN(I,J)+UN(I,J+1))
      1+ALPHA*DABS(VN(I,J)+VN(I+1,J))*UN(I,J)-UN(I,J+1))
      2-(VN(I,J-1)+VN(I+1,J-1))*UN(I,J-1)+UN(I,J))
      3-ALPHA*DABS(VN(I,J-1)+VN(I+1,J-1))*UN(I,J-1)-UN(I,J)))/(4.*DELY)
      VISX=NU*((UN(I+1,J)-2.*UN(I,J)+UN(I-1,J))/DELY**2+
      1*(UN(I,J+1)-2.*UN(I,J)+UN(I,J-1))/DELY**2)
864 ISN UI,I,J)=UN(I,J)+DELT*((P(I,J)-P(I+1,J))*RDX+GX-FUX-FUY +VISX)
      11050 CONTINUE
865 ISN FVY=((UN(I,J)+UN(I,J+1))*VN(I,J)+VN(I+1,J))+ALPHA*DABS(UN(I,J)
      1+UN(I,J+1))*VN(I,J)-VN(I+1,J))-UN(I-1,J)+UN(I,J+1))*VN(I-1,J)
      2+VN(I,J))-ALPHA*DABS(UN(I-1,J)+UN(I,J+1))*VN(I-1,J)-VN(I,J))
      3/(4.0*DELX)
866 ISN FVY=((UN(I,J)+VN(I,J+1))*VN(I,J)+VN(I+1,J))+ALPHA*DABS(VN(I,J)+VN
      1(I,J+1))*VN(I,J)-VN(I+1,J))-UN(I-1,J)+VN(I,J+1))*VN(I-1,J)
      2))-ALPHA*DABS(VN(I,J-1)+VN(I,J))*VN(I,J-1)-VN(I,J)))/(4.*DELY)
      VISY=NU*((VN(I+1,J)-2.*VN(I,J)+VN(I-1,J))/DELY**2+
      1*(VN(I,J+1)-2.*VN(I,J)+VN(I,J-1))/DELY**2)
867 ISN VI,I,J)=VN(I,J)+DELT*((P(I,J)-P(I,J+1))*RDY+GY-FVX-FVY +VISY)
      11100 CONTINUE
      C
      C * * TOP RIGID BOUNDARY CONDITIONS
      C
872 ISN 11200 CONTINUE
873 ISN ITU=0
874 ISN ITU=ITU+1
875 ISN UFLG=0.
876 ISN IF(ITU .GE. 500) GO TO 11500
877 ISN DO 11400 I=IRU,ILU
878 ISN J=JT(I)
879 ISN VTM=RDY*(HTU(I)-(J-2)*DELY)
880 ISN IF(DABS(U(I-1,J)) .LE. 1.E-6) U(I-1,J)=UI,I,J)
882 ISN F=-0.25*RDY*(H(I+1)-H(I-1))*UN(I,J)+U(I-1,J))+V(I,J-1)
      1 -VTM*DELY*RDY*(U(I,J)-U(I-1,J))
883 ISN DFDP=-DELT*RDY*(1.0+2.0*VTM*DELY**2*RDY**2
      1 +0.25*RDY**2*DELY*(H(I+1)-H(I-1))*AMAX0(0,JT(I)-JT(I-1))
      2 -AMAX0(0,JT(I)-JT(I+1)))
884 ISN DELP=-F/DFDP
885 ISN IF(P(I,J) .EQ. 0.) P(I,J)=P(I,J-1)
887 ISN CORP=DELP/P(I,J)
888 ISN IF(DABS(CORP) .GE. 0.05 .AND. DABS(DELP) .GE. 200.) UFLG=1.
889 ISN P(I,J)=P(I,J)+DELP
891 ISN VI,I,J)=V(I,J-1)-DELT*RDY*DELP
892 ISN VI,I,J)=V(I,J)+DELT*RDY*DELP
893 ISN IF(I .EQ. MLU) GO TO 11350
894 ISN UI,I,J)=UI,I,J)+DELT*RDY*DELP
895 ISN 11350 CONTINUE
896 ISN IF(I .EQ. IRU) GO TO 11400
897 ISN UI,I,J)=UI,I-1,J)-DELT*RDY*DELP
898 ISN 11400 CONTINUE
899 ISN IF(UFLG .EQ. 0.) GO TO 11500
900 ISN GO TO 11300

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*.....1.....2.....3.....4.....5.....6.....7.....8

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901 ISN 11500 CONTINUE
902 ISN IF(ITU .GE. 500) GO TO 11510
903 ISN GO TO 11520
904 ISN 11510 PRINT 25
905 ISN 11520 CONTINUE
C
906 ISN DO 11600 I=IRU,ILU
907 ISN JT1=JT(I)
908 ISN JT2=JT1+1
909 ISN U(I,JT2)=U(I,JT1)
910 ISN V(I,JT2)=V(I,JT1)
911 ISN 11600 CONTINUE
C
C * * BOTTOM RIGID BOUNDARY
C
912 ISN ITB=0
913 ISN 11650 CONTINUE
914 ISN BFLG=0.
915 ISN ITB=ITB+1
916 ISN IF(ITB .GE. 200) GO TO 11750
917 ISN DO 11700 I=IRU,ILU
918 ISN IF(I .EQ. MLU .OR. CDBB(I) .EQ. 0) GO TO 11700
919 ISN J=JB(I)
920 ISN VTM=RDY*(HB(I)-(J-2)*DELY)
921 ISN VBM=RDY*((J-1)*DELY-HB(I))
922 ISN F=-0.25*RDY*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
          1+VBM*(V(I,J)+DELY*RDY*(U(I,J)-U(I-1,J)))
          DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*RDY**2)
          DELP=-F/DFDP
          CORP=DELP/P(I,J)
          IF(DABS(CORP) .GE. .05) BFLG=1.
          P(I,J)=P(I,J)+DELP
          U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
          V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
          V(I,J)=V(I,J)+DELT*RDY*DELP
          U(I,J)=U(I,J)+DELT*RDY*DELP
923 ISN 11700 CONTINUE
924 ISN IF(BFLG .EQ. 0.) GO TO 11750
925 ISN GO TO 11650
926 ISN 11750 CONTINUE
C
C * * HAS CONVERGENCE BEEN REACHED
C
927 ISN DO 13800 ITER=1,500
C
C * * SET BOUNDARY CONDITIONS
C
928 ISN JB2=JB(ILU) -1
929 ISN JT2=JT(ILU) +1
930 ISN IF(ILU .LT. MLU) GO TO 11850
931 ISN DO 11800 J=JB2,JT2
932 ISN V(ILU+1,J)=2.*V(ILU,J)-V(ILU-1,J)
933 ISN 11800 CONTINUE
934 ISN GO TO 11900
935 ISN 11850 CONTINUE
936 ISN
937 ISN
938 ISN
939 ISN
940 ISN
941 ISN
942 ISN
943 ISN
944 ISN
945 ISN

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*.....1.....2.....3.....4.....5.....6.....7.....8

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946      DO 11900 J=JB2,JT2
947      V(ILU+1,J)=V(ILU,J)
948      IF(IILU+1.EQ.MLU) GO TO 11900
949      U(ILU+1,J)=U(ILU,J)
950      CONTINUE
951      JTI=JT(ILU)
952      U(ILU+1,JTI)=TU(ILU+1)
953      V(ILU+1,JTI)=TV(ILU+1)

C * * FREE SURFACE AND SLOPED BOUNDARY CONDITIONS
C
12000 CONTINUE
954      DO 12630 I=IRU,ILU
955      IF(HB(I).LE.HBU(I)) GO TO 12630
956      JB1=JB(I)
957      V(I,JB1-1)=V(I,JB1)+DELY*RDY*(U(I,JB1)-U(I-1,JB1))
958      U(I,JB1-1)=U(I,JB1)
959      CONTINUE
960      12630 CONTINUE

C * * COMPUTE UPDATED CELL PRESSURE AND VELOCITY
C
961      FLG=0.
962      DO 13600 I=IRU,ILU
963      JB1=JB(I)
964      JB11=JB(I)+1
965      JTI1=JT(I)-1
966      DO 13600 K=JB1,JTI1
967      J=JB1+JTI1-K
968      IF(J.NE.JB1) GO TO 13200
969      PETA=DELY/((DFLOAT(JB1)-0.5)*DELY-HB(I))
970      DELP=(1.-PETA)*P(I,JB1+1)-P(I,JB1)
971      GO TO 13300
972      CONTINUE
973      D=RDY*(U(I,J)-U(I-1,J))+RDY*(V(I,J)-V(I,J-1))
974      IF(DABS(D).GE.UPSI)FLG=1.0
975      DELP=-UBETA*D
976      P(I,J)=P(I,J)+DELP
977      V(I,J)=V(I,J)+DELT*RDY*DELP
978      IF(I.EQ.IRU) GO TO 13400
979      U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
980      CONTINUE
981      IF(I.EQ.MLU) GO TO 13500
982      U(I,J)=U(I,J)+DELT*RDY*DELP
983      CONTINUE
984      V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
985      CONTINUE
986      IF (FLG.EQ.0.) GO TO 14000
987      13800 CONTINUE
988

C * * NEW POSITION OF BOTTOM SURFACE
C
14000 CONTINUE
989      DO 14200 I=IRU,ILU
990      IF(I.EQ.MLU) GO TO 14210
991      JB1=JB(I)
992      HBV=RDY*(HBN(I)-FLOAT(JB1-2)*DELY)
993

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*.....1.....2.....3.....4.....5.....6.....7.....8

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994 UAV=.5*(U(I-1,JBI)+U(I,JBI))
995 HB(I)=HBN(I)+DELT*(HBVAV(I,JBI)+(1.-HBV)*V(I,JBI-1)-.5*RDX*(UAV
1 *HBN(I+1)+GAMMA*DABS(UAV)*(HBN(I)-HBN(I+1))-UAV*HBN(I-1)
2 -GAMMA*DABS(UAV)*(HBN(I-1)-HBN(I))))
996 IF(HB(I).LT.HBU(I)) HB(I)=HBU(I)
998 IF(HB(I).GT.HTU(I)) HB(I)=HTU(I)
1000 IU=IMAX*2-I
1001 IF(HB(I).LE.H(IU)) HB(I)=H(IU)
1003 RLH(I)=H(I)-HB(I)
1004 CONTINUE
1005 GO TO 14220
1006 CONTINUE
1007 FVOU=0.
1008 DO 14110 I=IRU,ILU
1009 FVOU=FVOU+(H(I)-HB(I))*DELX
1010 CONTINUE
1011 HUD=(FFVOL-FVOL-FVOU)*RDX
1012 ASDT=AS*DELT
1013 IF(HUD.LT.ASDT) HUD=ASDT
1015 IF(HUD.GT.2.*DELY) HUD=2.*DELY
1017 HB(MLU)=HBN(MLU)-HUD
1018 IF(HB(MLU).LT.HBU(MLU)) HB(MLU)=HBU(MLU)
1020 CONTINUE
C
C * * CALCULATE CELL IN WHICH SURFACE IS LOCATED AND UPDATE ARRAY
C
1021 DO 14250 I=IRU,ILU
1022 JBN=JB(I)
1023 JBN1=JBN-1
1024 JB0(I)=JB(I)
1025 JB1(I)= HB(I)*RDY+1.E-6+2.
1026 JBI=JB(I)
1027 IF(JBN-JB1)14224,14250,14240
1028 CONTINUE
1029 J1=JB1-2
1030 DO 14230 J=JBN1,J1
1031 V(I,J)=0.
1032 U(I,J)=0.
1033 P(I,J)=0.
1034 CONTINUE
1035 P(I,J1)=0.
1036 GO TO 14250
1037 CONTINUE
1038 DO 14248 JE=JB1,JBN
1039 J=JB1+JBN-JE
1040 IF(.GE.JT(I)) GO TO 14250
1041 TTM(I)=1.
1042 IF(.EQ. MLU) GO TO 14243
1043 U(I,J)=(U(I+1,J)+U(I-1,J)+U(I,J+1)+U(I,J-1))*0.25
1044 CONTINUE
1045 V(I,J)=(V(I,J-1)+V(I,J+1)+V(I-1,J)+V(I+1,J))*0.25
1046 P(I,J)=GY*(DFLOAT(J)-1.5)*DELY-HB(I)
1047 IF(SIGNL.LE.450) GO TO 14248
1048 PRINT 14245,I,J,U(I,J),V(I,J)
1049 FORMAT(2X,'ADDED',2(I3),3X,2(2X,1PE12.5))
1050 CONTINUE

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*.....1.....2.....3.....4.....5.....6.....7.....8

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1051 14250 CONTINUE
1052 DO 14270 I=IRU,ILU
1053   CODB(I)=0
1054   IF(HB(I).NE. HBU(I)) GO TO 14270
1055   CODB(I)=1
1056   IF(JB(I).EQ. JBO(I)) GO TO 14270
1057   JB1=JB(I)
1058   U(I,JB1)=BU(2 *IMAX-I)
1059   V(I,JB1)=BV(2 *IMAX-I)
1060 14270 CONTINUE
1061 14271 CONTINUE
1062   NIL=JT(ILU)-JB(ILU)
1063   IF(NIL .GE. 1) GO TO 14273
1064   ILU=ILU-1
1065   GO TO 14271
1066 14273 CONTINUE
1067 DO 14275 I=IRU,ILU
1068   IF(JT(I) .GT. JB(I)) GO TO 14275
1069   YTH(I)=1.
1070   JB(I)=JT(I)-1
1071   J=JB(I)
1072   HB(I)=(DFLOAT(J)-1.) *DELY
1073   V(I,J)=V(I,J+1)+(U(I,J+1)-U(I-1,J+1)) *DELY *RDX
1074   U(I,J)=0.5*(U(I-1,J)+U(I,J+1))
1075   P(I,J)=GY*((DFLOAT(J)-1.5) *DELY-HB(I))
1076   IF(SIGNL .LE. 450) GO TO 14275
1077   PRINT 14274,I,J
1078 14274 FORMAT(10X,'MAKEUP 2ND LAYER AT I,J=',2(2X,I4))
1079 14275 CONTINUE
1080   FVOU=0.
1081 DO 14290 I=IRU,ILU
1082   FVOU=FVOU+(H(I)-HB(I)) *DELX
1083 14290 CONTINUE
1084   IF(FVOU .LT. 0.) FVOU=0.
C
C * * NEW IL LOCATION
C
1086 14300 CONTINUE
1087   IF(ILU .EQ. MLU) GO TO 14500
1088   IF(ILU .LT. 117) GO TO 14305
1089   JB(ILU+1)=JT(ILU+1)-1
1090   HB(ILU+1)=H(ILU+1)-1.11 *DELY
1091   GO TO 14307
1092 14305 CONTINUE
1093   HB(ILU+1)=H(ILU+1)-(FFVOL-FVOL-FVOU) *RDX
1094   JB(ILU+1)= HB(ILU+1) *RDY+2.
1095   IF(JB(ILU+1)+1 .GT. JT(ILU+1)) GO TO 14500
1096 14307 CONTINUE
1097   ILU=ILU+1
1098   JB1=JB(ILU)
1099   JT1=JT(ILU)
1100   IF(ILU .LT. 90) GO TO 14315
1101 14310 CONTINUE
1102   DO 14312 JE=JB1,JT1
1103     J=JB1+JT1-JE
1104     U(ILU,J)=U(ILU-1,J)

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*.....1.....2.....3.....4.....5.....6.....7.....8

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1105 V(ILU,J)=V(ILU-1,J)
1106 P(ILU,J)=GY*(DFLOAT(J)-1.5)*DELY-HB(ILU))
1107 IF(U(ILU,J) .EQ. 0. .OR. V(ILU,J) .EQ. 0.) GO TO 14322
1108 CONTINUE
1109 GO TO 14324
1110 CONTINUE
1111 DO 14320 JE=JBI, JTI
1112 J=JBI+JTI-JE
1113 JI=J+JT(ILU-1)-JT(ILU)
1114 J2=J+JT(ILU-2)-JT(ILU)
1115 U(ILU,J)=2.*U(ILU-1,J1)-U(ILU-2,J2)
1116 V(ILU,J)=2.*V(ILU-1,J1)-V(ILU-2,J2)
1117 P(ILU,J)=GY*(DFLOAT(J)-1.5)*DELY-HB(ILU))
1118 IF(U(ILU,J) .EQ. 0. .OR. V(ILU,J) .EQ. 0.) GO TO 14322
1119 CONTINUE
1120 GO TO 14324
1121 JB(ILU)=J+1
1122 HB(ILU)=(DFLOAT(J)-1.)*DELY
1123 PRINT 27
1124 CONTINUE
1125 U(ILU,JTI)=TU(ILU)
1126 V(ILU,JTI)=TV(ILU)
1127 IF(ILU .EQ. MLU) GO TO 14325
1128 GO TO 14340
1129 CONTINUE
1130 JTI=JT(ILU)
1131 DO 14330 J=JBI, JTI
1132 U(ILU,J)=0.
1133 V(ILU,J)=-AS
1134 CONTINUE
1135 CONTINUE
1136 RLH(ILU)=H(ILU)-HB(ILU)
1137 CONTINUE
1138 ILU=ILU+1
1139 JTI=JTU(ILU)
1140 U(ILU, JTI)=TU(ILU)
1141 V(ILU, JTI)=TV(ILU)
1142 P(ILU, JTI)=P(ILU, JTI)
1143 CONTINUE
C
C * * TOP RGID BOUNDARY CONDITIONS
C
1144 ITU=0
1145 ITU=ITU+1
1146 UFG=0.
1147 IF(ITU .GE. 200) GO TO 14750
1148 DO 14700 I=IRU, ILU
1149 J=JT(I)
1150 VTM=RDY*(HTU(I)-(J-2)*DELY)
1151 IF(DABS(U(I-1,J)) .LE. 1.E-6) U(I-1,J)=U(I,J)
1152 F=-0.25*RDY*(H(I+1)-H(I-1))*(U(I,J)+U(I-1,J))+V(I,J-1)
1153 1 -VTM*DELY*RDY*(U(I,J)-U(I-1,J))
1154 DFDP= -DELT*RDY*(1.0+2.0*VTM*DELY**2*RDY**2
1 1 +0.25*RDY**2*DELY*(H(I+1)-H(I-1))*(AMAX0(0, JT(I)-JT(I-1))
2 -AMAX0(0, JT(I)-JT(I+1))))
1155 DELP=-F/DFDP

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*.....1.....2.....3.....4.....5.....6.....7.....8
ISN 1156 IF(P(I,J) .EQ. 0.) P(I,J)=P(I,J-1)
ISN 1158 CORP=DELP/P(I,J)
ISN 1159 IF(DABS(CORP) .GE. 0.05 .AND. DABS(DELP) .GE. 200.) UFLG=1.
ISN 1161 P(I,J)=P(I,J)+DELP
ISN 1162 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 1163 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 1164 IF(I .EQ. MLU) GO TO 14650
ISN 1165 U(I,J)=U(I,J)+DELT*ROX*DELP
ISN 1166 CONTINUE
ISN 1167 IF(I .EQ. IRU) GO TO 14700
ISN 1168 U(I-1,J)=(U(I-1,J)-DELT*ROX*DELP
ISN 1169 CONTINUE
ISN 1170 IF(UFLG .EQ. 0.) GO TO 14750
ISN 1171 GO TO 14660
ISN 1172 CONTINUE
C
C * * BOTTOM RIGID BOUNDARY
C
ISN 1173 ITB=0
ISN 1174 CONTINUE
ISN 1175 BFLG=0.
ISN 1176 ITB=ITB+1
ISN 1177 IF(ITB .GE. 200) GO TO 14820
ISN 1178 DO 14800 I=IRU,ILU
ISN 1179 IF(I .EQ. MLU .OR. CODBB(I) .EQ. 0) GO TO 14800
ISN 1180 J=JB(I)
ISN 1181 VTM=RDY*(HB(I)-(J-2)*DELY)
ISN 1182 VBM=RDY*((J-1)*DELY-HB(I))
ISN 1183 F=-0.25*ROX*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
      1+VBM*(V(I,J)+DELY*ROX*(U(I,J)-U(I-1,J)))
      DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*ROX**2)
      DELP=F/DFDP
ISN 1184 CORP=DELP/P(I,J)
ISN 1185 IF(DABS(CORP) .GE. .05) BFLG=1.
ISN 1186 P(I,J)=P(I,J)+DELP
ISN 1187 U(I-1,J)=U(I-1,J)-DELT*ROX*DELP
ISN 1188 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 1189 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 1190 U(I,J)=U(I,J)+DELT*ROX*DELP
ISN 1191 CONTINUE
ISN 1192 IF(BFLG .EQ. 0.) GO TO 14820
ISN 1193 GO TO 14780
ISN 1194 CONTINUE
ISN 1195
ISN 1196
ISN 1197
C
C
ISN 1198 DO 14850 I=IRU,ILU
ISN 1199 JT1=JT(I)
ISN 1200 JT2=JT1+1
ISN 1201 U(I,JT2)=U(I,JT1)
ISN 1202 V(I,JT2)=V(I,JT1)
ISN 1203 CONTINUE
C
C
ISN 1204 JB2=JB(ILU+1) -1
ISN 1205 JT2=JT(ILU+1) +1
ISN 1206 IF(ILU+1-ILU) 14805,14865,14855

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*.....1.....2.....3.....4.....5.....6.....7.....8

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1207 14855 CONTINUE
1208   DO 14860 J=JB2,JT2
1209   V(ILU+1,J)=2.*V(ILU,J)-V(ILU-1,J)
1210 14860 CONTINUE
1211   GO TO 14900
1212 14865 CONTINUE
1213   DO 14870 J=JB2,JT2
1214   V(ILU+1,J)=V(ILU,J)
1215 14870 CONTINUE
1216   GO TO 14900
1217 14885 CONTINUE
1218   DO 14900 J=JB2,JT2
1219   U(ILU+1,J)=U(ILU,J)
1220   V(ILU+1,J)=V(ILU,J)
1221 14900 CONTINUE
1222   JTI=JT(ILU+1)
1223   U(ILU+1,JTI)=TU(ILU+1)
1224   V(ILU+1,JTI)=TV(ILU+1)

C * * FREE SURFACE AND VOLUME
C
1225   DO 15000 I=IRU,ILU
1226   IF(HB(I) .LE. HBU(I)) GO TO 15000
1227   JBI=JB(I)
1228   IF(TTM(I) .NE. 1.) GO TO 14950
1229   V(I,JBI-1)=(V(I,JBI)+V(I,JBI-2)+V(I-1,JBI-1)+V(I+1,JBI-1))*0.25
1230   IF(I .EQ. MLU) GO TO 15000
1231   U(I,JBI-1)=(U(I,JBI)+U(I,JBI-2)+U(I-1,JBI-1)+U(I+1,JBI-1))*0.25
1232   GO TO 15000
1233 14950 CONTINUE
1234   V(I,JBI-1)=V(I,JBI)+DELY*RDY*(U(I,JBI)-U(I-1,JBI))
1235   U(I,JBI-1)=(U(I,JBI)+U(I,JBI-2)+U(I-1,JBI-1)+U(I+1,JBI-1))*0.25
1236 15000 CONTINUE
1237   FVOU=0.
1238   DO 15100 I=IRU,ILU
1239   FVOU=FVOU+(H(I)-HB(I))*DELX
1240 15100 CONTINUE
1241   PRINT 39, ITER,T,SIGNL,FVOU

C * * TRANSFER VELOCITY TO LOWER HALF
C
1242   DO 15200 IU=IRU,ILU
1243   I=2*IMAX-IU
1244   IF(HB(IU) .GT. H(I)) GO TO 15150
1245   H(I)=H(IU)
1246   HN(I)=H(I)
1247   JTI=JT(IU)
1248 15150 CONTINUE
1249   JBI=JB(IU)
1250   JTI=JT(IU)
1251   DO 15200 J=JBI,JTI
1252   U(I-1,J)=U(IU,J)
1253   UN(I,J)=U(I,J)
1254   V(I,J)=V(IU,J)
1255   VN(I,J)=V(I,J)
1256   P(I,J)=P(IU,J)

```

*.....1.....2.....3.....4.....5.....6.....7.....8

ISN 1257

15200 CONTINUE

C

C * * CHECK REACH BOTTOM SURFACE

C

ISN 1258 IF(ILU .LT. MLU) GO TO 15260

ISN 1259 ILLJ=0

ISN 1260 DO 15250 I=IRU,ILU

ISN 1261 IF(COBB(I) .NE. 1) GO TO 15250

ISN 1262 IF(ILL1 .NE. 0) GO TO 15240

ISN 1263 ILLJ=2*IMAX-I

ISN 1264 HN(ILL1)=H(I)

ISN 1265 H(ILL1)=H(I)

ISN 1266 JT(ILL1)=JT(I)

ISN 1267 JB(ILL1)=JB(I)

ISN 1268 JB2=JB(I)-1

ISN 1269 JT2=JT(I)+1

ISN 1270 DO 15230 J=JB2,JT2

ISN 1271 U(ILL1,J)=-U(I-1,J)

ISN 1272 U(ILL1-1,J)=-U(I,J)

ISN 1273 V(ILL1,J)=V(I,J)

ISN 1274 V(ILL1-1,J)=V(I+1,J)

ISN 1275 P(ILL1,J)=P(I,J)

ISN 1276 15230 CONTINUE

ISN 1277 JB1=JB(ILL1)

ISN 1278 U(ILL1,JB1)=BU(ILL1)

ISN 1279 V(ILL1,JB1)=BV(ILL1)

ISN 1280 V(ILL1,JB1-1)=V(ILL1,JB1)+DELY*RDY*(U(ILL1,JB1)-U(ILL1-1,JB1))

ISN 1281 U(ILL1,JB1-1)=(AS-V(ILL1,JB1-1)*VSIN(ILL1))/VCOS(ILL1)

ISN 1282 15240 CONTINUE

ISN 1283 IU=2*IMAX-I

ISN 1284 CODTB(IU)=1

ISN 1285 15250 CONTINUE

ISN 1286 15260 CONTINUE

C

C * * CALCULATE VELOCITY VECTOR

C

ISN 1287 IF(TTU+1.E-6 .LT. TUPRT) GO TO 16000

ISN 1288 TUPRT=TUPRT+CUPRT*DELTA

ISN 1289 DO 15300 I=IRU,ILU

ISN 1290 JB2=JB(I)-1

ISN 1291 JT2=JT(I)+1

ISN 1292 DO 15300 J=JB2,JT2

ISN 1293 UV(I,J)=DSQRT(U(I,J)*U(I,J)+V(I,J)*V(I,J))

ISN 1294 IF(DABS(U(I,J)) .LE. 1.E-6) U(I,J)=1.E-6

ISN 1295 ANG(I,J)=57.3*DATAN(V(I,J)/U(I,J))

ISN 1296 IF(U(I,J) .EQ. 1.E-6) U(I,J)=0.

ISN 1299 15300 CONTINUE

C

C * * LIST VELOCITY, PRESSURE, AND SURFACE POSITION

C

ISN 1300 PRINT 35

ISN 1301 PRINT 47

ISN 1302 DO 15900 I=IRU,ILU

ISN 1303 JB1=JB(I)

ISN 1304 JTI=JT(I)

ISN 1305 IB=2*IMAX-I

```

*.....1.....2.....3.....4.....5.....6.....7.....8
1306      JB2=JB(I)-1
1307      JT2=JT(I)+1
1308      DO 15900 J=JB2,JT2
1309      PRINT 38,I,IB,J,U(I,J),V(I,J),P(I,J), HB(I),JTI,JBI,UV(I,J),
        .LANG(I,J)
1310      15900 CONTINUE
C
C * * SET THE ADVANCE TIME VELOCITIES U AND V INTO THE UN AND VN ARRAYS
C * * AND THE ADVANCE TIME SURFACE HEIGHT H INTO THE HN ARRAY
C
1311      16000 CONTINUE
1312      IRU=IRU-1
1313      ILU=ILU+1
1314      DO 16100 I=IRU,ILU
1315      TTM(I)=0.
1316      JB2=JB(I)-1
1317      JT2=JT(I)+1
1318      DO 16100 J=JB2,JT2
1319      UN(I,J)=U(I,J)
1320      VN(I,J)=V(I,J)
1321      HBN(I)=HB(I)
1322      16100 CONTINUE
C
C * * ADVANCE TIME T=T+DELTA
C
1323      TTU=TTU+DELTA
1324      SIGNAL=SIGNAL+1
1325      16200 CONTINUE
C
C * * INPUT TO LOWER SECTION
C
1326      IF(ILL1 .EQ. 0) GO TO 20000
1327      IF(MEET .EQ. 0) GO TO 16300
1328      IL=ILL1+1
1329      LI=IL
1330      FL0L=0.
1331      GO TO 20000
1332      16300 CONTINUE
1333      TOVOL=FFVOL-FVOL-FVOU-FL0L
1334      ILL=ILL1+1
1335      IF(ILL .LE. LR) GO TO 17000
1336      16400 CONTINUE
1337      IF(TDVOL .LE. 0.) GO TO 20000
1338      H(ILL)=HB(ILL)+TDVOL*RDY
1339      JT(ILL)=H(ILL)*RDY+1.E-6 +2.
1340      PRINT 29, ILL,JT(ILL),H(ILL)
1341      IF(JT(ILL) .EQ. JB(ILL)) GO TO 20000
1342      LL=ILL
1343      HN(LL)=H(LL)
1344      LR=LL
1345      JTI=JT(LL)
1346      JBI=JB(LL)
1347      IF(ILL1 .NE. ML) GO TO 16470
1348      DO 16450 J=JBI,JTI
1349      U(EL,J)=0.5*U(ML,J)
1350      V(ILL,J)=0.5*V(ML,J)

```

*.....1.....2.....3.....4.....5.....6.....7.....8

```

1351 ISN P(LL,J)=-GY*(H(LL)-(DFLOAT(J)-1.5)*DELY)
1352 ISN 16450 CONTINUE
1353 ISN U(LL,JBI)=TU(LL)
1354 ISN V(LL,JBI)=TV(LL)
1355 ISN CODBB(LL)=1
1356 ISN GO TO 16500
1357 ISN 16470 CONTINUE
1358 ISN DO 16490 J=JBI,JTI
1359 ISN J1=J-JB(LL)+JB(LL-1)
1360 ISN J2=J-JB(LL)+JB(LL-2)
1361 ISN U(LL,J)=2.*U(LL-1,J1)-U(LL-2,J2)
1362 ISN V(LL,J)=2.*V(LL-1,J1)-V(LL-2,J2)
1363 ISN P(LL,J)=-GY*(H(LL)-(DFLOAT(J)-1.5)*DELY)
1364 ISN 16490 CONTINUE
1365 ISN 16500 CONTINUE
1366 ISN CODBB(LL)=1
1367 ISN U(LL,JBI)=TU(LL)
1368 ISN V(LL,JBI)=TV(LL)
1369 ISN V(LL,JBI-1)=V(LL,JBI)+(U(LL,JBI)-U(LL-1,JBI))*DELY*RDY
1370 ISN U(LL,JBI-1)=(AS-V(LL,JBI-1)*VSIN(LL))/VCOS(LL)

```

C * * BOTTOM BOUNDARY CONDITION

C

```

1371 ISN DO 16600 ITB=1,100
1372 ISN BFLG=0.
1373 ISN DO 16550 I=LL,LR
1374 ISN J=JBI(I)
1375 ISN VTM=RDY*(HB(I)-(J-2)*DELY)
1376 ISN VBM=RDY*((J-1)*DELY-HB(I))
1377 ISN F=-0.25*RDY*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
1378 ISN 1+VBM*(V(I,J)+DELY*RDY*(U(I,J)-U(I-1,J)))
1379 ISN DFDP=DELT*RDY*(1.0+2.0*VBM*DELY**2*RDY**2)
1380 ISN DELP=-F/DFDP
1381 ISN CORP=DELP/P(I,J)
1382 ISN IF(DABS(CORP) .GE. .05 .AND. DABS(DELP) .GE. 200) BFLG=1.
1383 ISN P(I,J)=P(I,J)+DELP
1384 ISN V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
1385 ISN V(I,J)=V(I,J)+DELT*RDY*DELP
1386 ISN U(I,J)=U(I,J)+DELT*RDY*DELP
1387 ISN IF(I .EQ. LL) GO TO 16550
1388 ISN U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
1389 ISN 16550 CONTINUE
1390 ISN IF(BFLG .EQ. 0.) GO TO 16650
1391 ISN 16600 CONTINUE
1392 ISN 16650 CONTINUE
1393 ISN U(LL,JBI-1)=V(LL,JBI)+(U(LL,JBI)-U(LL-1,JBI))*DELY*RDY
1394 ISN V(LL,JBI-1)=(AS-V(LL,JBI-1)*VSIN(LL))/VCOS(LL)

```

C * * FREE SURFACE CONDITIONS

C

```

1395 ISN DO 16700 I=LL,LR
1396 ISN JTI=JT(I)
1397 ISN V(I,JTI)=V(I,JTI-1)-DELY*RDY*(U(I,JTI)-U(I-1,JTI))
1398 ISN U(I,JTI+1)=U(I,JTI)
1399 ISN 16700 CONTINUE
1400 ISN JB2=JB(LR)-1

```

*.....1.....2.....3.....4.....5.....6.....7.....8

```

1401 ISN      JT2=JT(LR)+1
1402 ISN      LR1=LR+1
1403 ISN      DO 16800 J=JB2,JT2
1404 ISN      U(LR1,J)=U(LR,J)
1405 ISN      V(LR1,J)=V(LR,J)
1406 ISN      16800 CONTINUE
1407 ISN      JB2=JB(LR1)-1
1408 ISN      U(LR1,JB2+1)=BU(LR1)
1409 ISN      V(LR1,JB2+1)=BV(LR1)
1410 ISN      GO TO 19700
1411 ISN      17000 CONTINUE
1412 ISN      LL=ILL
1413 ISN      FLG=1.
1414 ISN      ALPHA=DLPHA
    
```

C * * COMPUTE TEMPORARY U AND V
C

```

1415 ISN      DO 17100 I=LL,LR
1416 ISN      JTI= JT(I)
1417 ISN      JB1=JB(I)+1
1418 ISN      DO 17100 J=JB1,JTI
1419 ISN      FUX=((UN(I,J)+UN(I+1,J))*UN(I,J)+UN(I+1,J))+ALPHA*DABS(UN(I,J)
1420 ISN      1*UN(I+1,J))*UN(I,J)-UN(I+1,J))-UN(I-1,J)+UN(I,J))*UN(I-1,J)
1421 ISN      2*UN(I,J))-ALPHA*DABS(UN(I-1,J)+UN(I,J))*UN(I-1,J)-UN(I,J))
1422 ISN      3/(4.0*DELX)
1423 ISN      FUY=((VN(I,J)+VN(I+1,J))*UN(I,J)+UN(I+1,J)
1424 ISN      1*ALPHA*DABS(VN(I,J)+VN(I+1,J))*UN(I,J)-UN(I,J)+1)
1425 ISN      2-(VN(I,J)-1)+VN(I+1,J-1))*UN(I,J-1)+UN(I,J)
1426 ISN      3-ALPHA*DABS(VN(I,J)-1)+VN(I+1,J-1))*UN(I,J-1)-UN(I,J))/(4.*DELY)
1427 ISN      VISX=NU*((UN(I+1,J)-2.*UN(I,J)+UN(I-1,J))/DELX**2+
1428 ISN      1*(UN(I,J)+1)-2.*UN(I,J)+UN(I,J-1))/DELY**2)
1429 ISN      U(I,J)=UN(I,J)+DELTA*(PI,I,J)-P(I+1,J))*RDX+GX-FUX-FUY +VISX)
1430 ISN      FVX=((UN(I,J)+UN(I+1,J))*VN(I,J)+VN(I+1,J))+ALPHA*DABS(UN(I,J)
1431 ISN      1*VN(I+1,J))*VN(I,J)-VN(I+1,J))-UN(I-1,J)+UN(I,J))*VN(I-1,J)
1432 ISN      2*VN(I,J))-ALPHA*DABS(UN(I-1,J)+UN(I,J))*VN(I-1,J)-VN(I,J))
1433 ISN      3/(4.0*DELX)
1434 ISN      FVY=((VN(I,J)+VN(I+1,J))*VN(I,J)+VN(I+1,J))+ALPHA*DABS(VN(I,J)+VN
1435 ISN      1(I,J)+1))*VN(I,J)-VN(I,J+1))-UN(I,J-1)+VN(I,J))*VN(I,J-1)+VN(I,J)
1436 ISN      2))-ALPHA*DABS(VN(I,J)-1)+VN(I+1,J))*VN(I,J-1)-VN(I,J))/(4.*DELY)
1437 ISN      VISY=RU*((VN(I+1,J)-2.*VN(I,J)+VN(I-1,J))/DELX**2+
1438 ISN      1*(VN(I,J)+1)-2.*VN(I,J)+VN(I,J-1))/DELY**2)
1439 ISN      V(I,J)=VN(I,J)+DELTA*(PI,I,J)-P(I,J+1))*RDY+GY-FVX-FVY +VISY)
1440 ISN      17100 CONTINUE
    
```

C * * * BOTTOM BOUNDARY CONDITIONS
C

```

1428 ISN      DO 17200 ITB=1,100
1429 ISN      BFLG=0.
1430 ISN      DO 17150 I=LL,LR
1431 ISN      J=JB1I
1432 ISN      VTM=RDY*(HB(I)-(J-2)*DELY)
1433 ISN      VBH=RDY*((J-1)*DELY-HB(I))
1434 ISN      F=-0.25*RDX*(HB(I+1)-HB(I-1))*U(I,J)+U(I-1,J))+V(I,J)*VTM
1435 ISN      1*VBH*(V(I,J)+DELY*RDX*(U(I,J)-U(I-1,J)))
1436 ISN      DFDP=DELT*RDY*(1.0+2.0*VBH*DELY**2*RDX**2)
1437 ISN      DELP=-F/DFDP
    
```

*.....1.....2.....3.....4.....5.....6.....7.....8

```

ISN 1437 CORP=DELP/P(I,J)
ISN 1438 IF(DABS(CORP) .GE. .05 .AND. DABS(DELP) .GE. 200) BFLG=1.
ISN 1440 P(I,J)=P(I,J)+DELP
ISN 1441 V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
ISN 1442 V(I,J)=V(I,J)+DELT*RDY*DELP
ISN 1443 U(I,J)=U(I,J)+DELT*RDY*DELP
ISN 1444 IF(I .EQ. LL) GO TO 17150
ISN 1445 U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
ISN 1446 U(I,J)=U(I,J)+DELT*RDY*DELP
ISN 1447 IF(BFLG .EQ. 0.) GO TO 17300
ISN 1448 17200 CONTINUE

```

C * * HAS CONVERGENCE BEEN REACHED
C
C 17300 CONTINUE
C DO 18000 ITER=1,200

C * * SET LEFT AND RIGHT BOUNDARY CONDITIONS
C

```

ISN 1451 JB2=JB(LR)-1
ISN 1452 JT2=JT(LR)+1
ISN 1453 DO 17400 J=JB2,JT2
ISN 1454 U(LR+1,J)=U(LR,J)
ISN 1455 V(LR+1,J)=V(LR,J)
ISN 1456 17400 CONTINUE
ISN 1457 JB1=JB(LR+1)
ISN 1458 U(LR+1,JB1)=BU(LR+1)
ISN 1459 V(LR+1,JB1)=BV(LR+1)

```

C * * FREE SURFACE CONDITIONS
C

```

ISN 1460 DO 17500 I=LL,LR
ISN 1461 JT1=JT(I)
ISN 1462 IF(I .LE. 6) GO TO 17450
ISN 1463 IF(JT(I+1) .LT. JT(I)) U(I,JT1)=U(I,JT1-1)
ISN 1464 17450 CONTINUE
ISN 1465 V(I,JT1)=V(I,JT1-1)-DELT*RDY*(U(I,JT1)-U(I-1,JT1))
ISN 1466 U(I,JT1+1)=U(I,JT1)
ISN 1467 17500 CONTINUE
ISN 1468

```

C * * COMPUTE UPDATED CELL PRESSURE AND VELOCITY
C

```

ISN 1469 FLG=0.
ISN 1470 DO 17800 I=LL,LR
ISN 1471 JT1=JT(I)
ISN 1472 JB1=JB(I)+1
ISN 1473 DO 17800 J=JB1,JT1
ISN 1474 IF(J .NE. JT1) GO TO 17600
ISN 1475 PETA=DELY/(HN(I)-(DFLOAT(JT1)-2.5)*DELY)
ISN 1476 DELP=(1.0-PETA)*P(I,JT1-1)-P(I,JT1)
ISN 1477 GO TO 17700
ISN 1478 17600 CONTINUE
ISN 1479 D=RDY*(U(I,J)-U(I-1,J))+RDY*(V(I,J)-V(I,J-1))
ISN 1480 IF(DABS(D) .GE. EPSI) FLG=1.0
ISN 1481 DELP=BETA*D
ISN 1482 P(I,J)=P(I,J)+DELP
ISN 1483 17700

```


*.....1.....2.....3.....4.....5.....6.....7.....8

```

ISN 1484      U(I,J)=U(I,J)+DELT*RDY*DDEL
ISN 1485      IF(I.EQ.LL) GO TO 17750
ISN 1486      U(I-1,J)=U(I-1,J)-DELT*RDY*DDEL
ISN 1487      17750 CONTINUE
ISN 1488      V(I,J)=V(I,J)+DELT*RDY*DDEL
ISN 1489      IF(J.EQ.JB1) GO TO 17800
ISN 1490      V(I,J-1)=V(I,J-1)-DELT*RDY*DDEL
ISN 1491      17800 CONTINUE
ISN 1492      IF(IFLG.EQ.0.) GO TO 18100
ISN 1493      18000 CONTINUE
C
C * * COMPUTE NEW SURFACE POSITION
C
ISN 1494      18100 CONTINUE
ISN 1495      DO 18200 I=LL,LR
ISN 1496      JTI=JT(I)
ISN 1497      HV=RDY*(HN(I)-DFLOAT(JTI-2))*DDELY
ISN 1498      UAV=0.5*(U(I-1,JTI)+U(I,JTI))
ISN 1499      H(I)=HN(I)+DELT*(HV*V(I,JTI)+(1.0-HV)*V(I,JTI-1))
          1  -0.5*RDY*(UAV*HN(I+1)+GAMMA*DABS(UAV))*(HN(I)-HN(I+1))
          2  -UAV*HN(I-1)-GAMMA*DABS(UAV)*(HN(I-1)-HN(I)))
ISN 1500      IF(H(I).LT.HB(I)) H(I)=HB(I)
ISN 1502      18200 CONTINUE
C
C * * CALCULATE CELL IN WHICH SURFACE IS LOCATED AND UPDATE ARRAY
C
ISN 1503      DO 18500 I=LL,LR
ISN 1504      JTO(I)=JT(I)
ISN 1505      JTI(I)= H(I)*RDY+1.0E-8+2.
ISN 1506      IF(JTI(I).EQ.JTO(I)) GO TO 18500
ISN 1507      JTI=JTI(I)
ISN 1508      IF(JTI.GT.JTO(I)) GO TO 18400
ISN 1509      JI=JTI+1
ISN 1510      JTOI=JTO(I)
ISN 1511      DO 18300 J=JI,JTOI
ISN 1512      U(I,J)=0.
ISN 1513      V(I,J)=0.
ISN 1514      P(I,J)=0.
ISN 1515      18300 CONTINUE
ISN 1516      GO TO 18500
ISN 1517      18400 CONTINUE
ISN 1518      JTI=JTO(I)+1
ISN 1519      DO 18500 J=JTI,JTI
ISN 1520      P(I,J)=-GY*(H(I)-(DFLOAT(J)-1.5)*DDELY)
ISN 1521      18500 CONTINUE
C
C * * CALCULATE NEW VOLUME
C
ISN 1522      LR2=LR-1
ISN 1523      DO 18600 I=LL,LR2
ISN 1524      IF(JTI(I).GT.JB(I)) GO TO 18600
ISN 1525      JTI(I)=JB(I)+1
ISN 1526      J=JTI(I)
ISN 1527      H(I)=(DFLOAT(J)-2.)*DDELY
ISN 1528      V(I,J)=0.5*(V(I-1,J)+V(I+1,J))
ISN 1529      U(I,J)=U(I-1,J)+(V(I,J-1)-V(I,J))*DELX*RDY

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.....1.....2.....3.....4.....5.....6.....7......8

```

1530 ISN P(I,J)=-GY*(H(I)-(DFLOAT(J)-1.5)*DELY)
1531 ISN 18600 CONTINUE
1532 ISN FLOL=0.
1533 ISN DO 18700 I=LL,LR
1534 ISN FLOL=FLOL+(H(I)-HB(I))*DELX
1535 ISN 18700 CONTINUE

```

C * * NEW IR LOCATION

```

1536 ISN NIR=JT(LR)-JB(LR)
1537 ISN IF(NIR .LE. 0) GO TO 18900
1538 ISN IF(LR+2 .LT. IL) GO TO 18730
1539 ISN MEET=1
1540 ISN H(LR+1)=0.5*(H(LR)+H(IL))
1541 ISN JT(LR+1)=H(LR+1)*RDY+2.
1542 ISN LR=LR+1
1543 ISN PRINT 42, LR, JT(LR), H(LR)
1544 ISN CODBB(LR)=1
1545 ISN JTI=JT(LR)
1546 ISN JBI=JB(LR)
1547 ISN DO 18710 J=JBI, JTI
1548 ISN P(LR, J)=-GY*(H(LR)-(DFLOAT(J)-1.5)*DELY)
1549 ISN U(LR, J)=0.5*(U(LR-1, J)+U(LR+1, J))
1550 ISN V(LR, J)=0.5*(V(LR-1, J)+V(LR+1, J))
1551 ISN 18710 CONTINUE
1552 ISN U(LR, JBI)=BU(LR)
1553 ISN V(LR, JBI)=BV(LR)
1554 ISN V(LR, JBI-1)=V(LR, JBI)+(U(LR, JBI)-U(LR-1, JBI))*DELY*RDY
1555 ISN U(LR, JBI-1)=(AS-V(LR, JBI-1))*VSIN(LR)/VCOS(LR)
1556 ISN GO TO 19100

```

18730 CONTINUE

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1557 ISN H(LR+1)=HB(LR+1)*(FFVOL-FVOL-FVOU-FLOL)*RDX
1558 ISN JT(LR+1)=H(LR+1)*RDY+2.
1559 ISN IF(JT(LR+1) .LE. JB(LR+1)) GO TO 19000
1560 ISN LR=LR+1
1561 ISN PRINT 42, LR, JT(LR), H(LR)
1562 ISN CODBB(LR)=1
1563 ISN JTI=JT(LR)
1564 ISN JBI=JB(LR)
1565 ISN DO 18800 J=JBI, JTI
1566 ISN P(LR, J)=-GY*(H(LR)-(DFLOAT(J)-1.5)*DELY)
1567 ISN U(LR, J)=U(LR-1, J)
1568 ISN V(LR, J)=V(LR-1, J)
1569 ISN 18800 CONTINUE
1570 ISN U(LR, JBI)=BU(LR)
1571 ISN V(LR, JBI)=BV(LR)
1572 ISN RLH(LR)=H(LR)-HB(LR)
1573 ISN V(LR, JBI-1)=V(LR, JBI)+(U(LR, JBI)-U(LR-1, JBI))*DELY*RDY
1574 ISN U(LR, JBI-1)=(AS-V(LR, JBI-1))*VSIN(LR)/VCOS(LR)
1575 ISN GO TO 19000

```

18900 CODBB(LR)=0

```

1576 ISN LR=LR-1
1577 ISN IF(JT(LR)-JB(LR) .LT. 1) GO TO 18900
1578 ISN 19000 CONTINUE
1579 ISN LRI=LR+1
1580 ISN JBI=JB(LR+1)
1581 ISN
1582 ISN

```

.....1.....2.....3.....4.....5.....6.....7..........8

```

1583 ISN      U(LR1,JB1)=BU(LR1)
1584 ISN      V(LR1,JB1)=BV(LR1)
1585 ISN      P(LR1,JB1)=P(LR,JB1)

C * * BOTTOM BOUNDARY CONDITIONS
C
19100 CONTINUE
1586 ISN      DO 19300 ITB=1,100
1587 ISN      BFLG=0.
1588 ISN      DO 19200 I=LL,LR
1589 ISN      J=JB(I)
1590 ISN      VTM=RDY*(HB(I)-(J-2)*DELY)
1591 ISN      VBN=RDY*(J-1)*DELY-HB(I))
1592 ISN      F=-0.25*RDY*(HB(I+1)-HB(I-1))*(U(I,J)+U(I-1,J))+V(I,J)*VTM
1593 ISN      1+VBN*(V(I,J)+DELY*RDY*(U(I,J)-U(I-1,J)))
1594 ISN      DFDP=DELT*RDY*(1.0+2.0*VBN*DELY**2*RDY**2)
1595 ISN      DELP=-F/DFDP
1596 ISN      CORP=DELP/P(I,J)
1597 ISN      IF(DABS(CORP) .GE. .05 .AND. DABS(DELP) .GE. 200) BFLG=1.
1598 ISN      P(I,J)=P(I,J)+DELP
1599 ISN      V(I,J-1)=V(I,J-1)-DELT*RDY*DELP
1600 ISN      V(I,J)=V(I,J)+DELT*RDY*DELP
1601 ISN      U(I,J)=U(I,J)+DELT*RDY*DELP
1602 ISN      U(I, J)=U(I, J)+DELT*RDY*DELP
1603 ISN      IF(I .EQ. LL) GO TO 19200
1604 ISN      U(I-1,J)=U(I-1,J)-DELT*RDY*DELP
1605 ISN      19200 CONTINUE
1606 ISN      IF(BFLG .EQ. 0.) GO TO 19400
1607 ISN      19300 CONTINUE
C
C * * IR+1 VELOCITY
C
19400 CONTINUE
1608 ISN      IF(LR+1 .EQ. IL) GO TO 19550
1609 ISN      JB2=JB(LR)-1
1610 ISN      JT2=JT(LR)+1
1611 ISN      LRI=LR+1
1612 ISN      DO 19500 J=JB2,JT2
1613 ISN      U(LR1,J)=U(LR,J)
1614 ISN      V(LR1,J)=V(LR,J)
1615 ISN      19500 CONTINUE
1616 ISN      U(LR1,JB2+1)=BU(LR1)
1617 ISN      V(LR1,JB2+1)=BV(LR1)
1618 ISN
C
C * * FREE SURFACE CONDITION
C
19550 CONTINUE
1619 ISN      DO 19600 I=LL,LR
1620 ISN      JTI=JT(I)
1621 ISN      IF(I .LE. 6) GO TO 19570
1622 ISN      IF(JT(I+1) .LT. JT(I)) U(I,JTI)=U(I,JTI-1)
1623 ISN      19570 CONTINUE
1624 ISN      V(I,JTI)=V(I,JTI-1)-DELY*RDY*(U(I,JTI)-U(I-1,JTI))
1625 ISN      U(I,JTI+1)=U(I,JTI)
1626 ISN      19600 CONTINUE
1627 ISN
C
C * * CALCULATE TOTAL VOLUME

```

*.....1.....2.....3.....4.....5.....6.....7.....8

ISN 1629
ISN 1630
ISN 1631
ISN 1632
ISN 1633
ISN 1634
ISN 1635

```
C
19700 CONTINUE
FLOL=0.
DO 19800 I=LL,LR
  COBB(I)=I
  FLOL=FLOL+(H(I)-HB(I))*DELX
19800 CONTINUE
PRINT*9,ITER,TTD,LYCLE,FLOL
```

ISN 1636
ISN 1637
ISN 1638
ISN 1639
ISN 1640
ISN 1641
ISN 1642
ISN 1643

```
C * * TRANSFER TO UPPER SECTION
C
I=LL
IU=2*IMAX-I
JB2=JB(I)-1
JT2=JT(I)+1
DO 19850 J=JB2,JT2
  U(IU,J)=-U(I-1,J)
  V(IU,J)=V(I,J)
19850 CONTINUE
```

ISN 1644
ISN 1645
ISN 1646
ISN 1647
ISN 1648
ISN 1649
ISN 1650
ISN 1651
ISN 1653
ISN 1654
ISN 1656

```
C * * CALCULATE VELOCITY VECTOR
C
IF(TTD+1.E-6 .LT. TLPRT) GO TO 19930
TLPRT=TLPRT+CLPRT*DELX
DO 19900 I=ILL,LR
  JB2=JB(I)-1
  JT2=JT(I)+1
  DO 19900 J=JB2,JT2
    UV(I,J)=DSQRT(U(I,J)*U(I,J)+V(I,J)*V(I,J))
    IF(DABS(U(I,J)) .LE. 1.E-6) U(I,J)=1.E-6
    ANG(I,J)=57.3*DATAN(V(I,J)/U(I,J))
    IF(U(I,J) .EQ. 1.E-6) U(I,J)=0.
19900 CONTINUE
```

ISN 1657
ISN 1658
ISN 1659
ISN 1660
ISN 1661
ISN 1662
ISN 1663
ISN 1664
ISN 1665
ISN 1666

```
C * * LIST VELOCITY, PRESSURE, AND SURFACE POSITION
C
PRINT 35
PRINT 47
DO 19920 I=ILL,LR
  JT1=JT(I)
  JT2=JT1+1
  JB1=JB(I)
  JB2=JB1-1
  DO 19920 J=JB2,JT2
    PRINT 48,I,J,U(I,J),V(I,J),P(I,J),H(I),JT1,JB1,UV(I,J),ANG(I,J)
19920 CONTINUE
```

ISN 1667
ISN 1668
ISN 1669
ISN 1670
ISN 1671
ISN 1672

```
C * * SET THE ADVANCE TIME VELOCITIES U AND V INTO THE UN AND VN ARRAYS
C * * AND THE ADVANCE TIME SURFACE HEIGHT H INTO THE HN ARRAY
C
19930 CONTINUE
LR1=LR+1
DO 19940 I=ILL,LR1
  JT2=JT(I)+1
  JB2=JB(I)-1
  DO 19940 J=JB2,JT2
```

..........1.....2.....3.....4.....5.....6.....7.....*.....8

ISN 1673 UN(I,J)=U(I,J)
ISN 1674 VN(I,J)=V(I,J)
ISN 1675 HN(I)=H(I)
ISN 1676 19940 CONTINUE

C * * ADVANCE TIME T=T+DELT

ISN 1677 TTD=TTD+DELT
ISN 1678 LYCLE=LYCLE+1
ISN 1679 20000 CONTINUE
ISN 1680 IF(ITER .GT. 500) GO TO 20500
ISN 1681 T=T+DELT
ISN 1682 20300 CONTINUE
ISN 1683 20500 STOP
ISN 1684 END

STATISTICS SOURCE STATEMENTS = 1638, PROGRAM SIZE = 381744 BYTES, PROGRAM NAME = MAIN PAGE: 1.

STATISTICS NO DIAGNOSTICS GENERATED.

***** END OF COMPILATION 1 *****

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